



Sri Indu Institute of Engineering & Technology

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3.2.2 Number of books and chapters in edited volumes/books published and papers published in national/international conference proceedings per teacher during last five years (10)

SUMMARY SHEET

Academic Years	2019-20	2018-19	2017-18	2016-17	2015-16	TOTAL
TOTAL	19	5	2	3	8	37

PRINCIPAL
Sri Indu Institute of Engineering & Tech
Sheriguda(V), Ibrahimpatnam(M)
R.R. Dist. Telangana. -501 510



Academic Year 2019-2020

Research Publications – Book & Book Chapters


Sl. No	Name of the Faculty Author	Title of the Book Chapter	Name of the Publisher	ISBN Number	URL
1	Dr.B.Gangadhara Bula Reddy	A Brevity & Descriptive View On Big Data And Data Science	Shashwat	978-93-90290-55-0	https://shashwatpublication.com/books/A%20Brevity%20&%20Descriptive%20View%20On%20Big%20Data%20And%20Data%20Science
2	Gaddam Rahul Paul	Fundamentals of Management	Walnut Publication	978-938974439-2	https://www.walnutpublication.com/book/9789389744392/
3	U.Fanidara Baradwaja Sharma	Fundamentals of Management	Walnut Publication	978-938974439-2	https://www.walnutpublication.com/book/9789389744392/
4	Dr. Dayaydi Lakshmaiah	Analog And Pulse Circuits	BS Publications / BSP Books	9789389354096	https://www.bspublications.net/book_detail.php?bid=1546
5	Dr. Dayaydi Lakshmaiah	Minimization Of Dynamic Power Through Structural Changes And Threshold Voltage Techniques	LAP Lambert Academic Publishing	978-620-2-67229-0	https://www.lap-publishing.com/catalog/details/store/gb/book/978-620-2-67229-0/minimization-of-dynamic-power-through-structural-changes-amp;vt-techniques



Academic Year 2015-2016

Research Publications – Book & Book Chapters

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1	Dr. D. Premalatha	Advances in Bioprocess Technology /Starter Culture Technology: Fermented Foods	Springer	978-3-319-1794-8	https://books.google.co.in/books?id=Y4RgCgAAQBAJ&printsec=frontcover&source=gbs_ge_summary_r&cad=0#v=onepage&q&f=true


Dr. D. Premalatha
Sri Indu Institute of Engineering & Technology
JNTU, Hyderabad



A Brevity & Descriptive view on Big Data and Data Science

DATA SCIENCE

Nazim Shaik
Dr.K.B.S. Venkatesan
Dr.B. Gangadhara
Obula Reddy

About the Book

Big Data approach cannot be easily achieved using traditional data analysis methods. Instead, understanding how various and novel data-related data modeling technologies work can help us make a rational choice. Statistics and mathematics is applied to explain them. Data Science has been a versatile approach that applies mathematical and statistical methods and computer tools for making processed big data. Here, science and applied and focused field that combines various fields such as science, mathematics, intelligent data capture technologies, data discovery, mining and programming to prepare and shape big data. Intelligent analysis to extract insights and information.

Data Science and Big Data are a rapidly growing fields. The field applications in many areas like health care, business, advertising, marketing and sales. This book looks at how the big data and Big Data are being generated and integrated to various communication areas by other technologies for making a good impact on the users. Actually, we are not intended to make the chapter but on the very people who are having such application in their mind by the book.

Key Features

- 1. It provides a history and a critical coverage on Big Data and Data Science in a descriptive manner.
- 2. Some of the contemporary technologies associated along with them in a hybrid way and it gives you a wide information in more about such contemporary as well, which are being associated with these technologies such as Big Data and Data Science.
- 3. It describes the differences among Data Science with other data technologies and Big Data differences with other technologies as well.

- 4. It allows to understand every topic in a new way and will also be important and specific and detailed manner.
- 5. This book will not be made a platform over these technologies for the students and business and industry people who really know it or really do know.
- 6. This book explains that how quickly technologies are being created a head by its right as well as being made a use in the job market.


 P. R. S. RAO
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A Brevity

&

Descriptive View on Data
Science
and
Big Data

Authors

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I dedicate to the my mighty God Almighty
 — Dr. Reginald C. (Daddy) —

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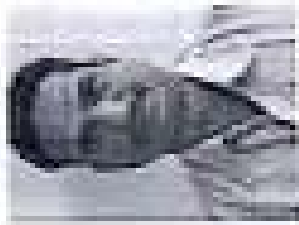
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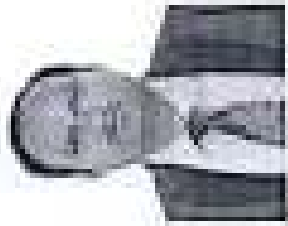
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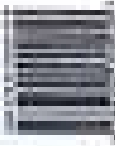


About the Book:

The book presents a comprehensive overview of the various aspects of the field of data science, covering topics such as data collection, data analysis, data visualization, data mining, and data security. The book is designed to be a valuable resource for students, researchers, and professionals alike, providing a solid foundation in the field of data science. The book is divided into several chapters, each focusing on a different aspect of the field. The first chapter discusses the importance of data in the modern world, while the second chapter explores the various methods used to collect and store data. The third chapter covers the different types of data, including structured and unstructured data, and the fourth chapter discusses the various techniques used to analyze data. The fifth chapter focuses on data visualization, while the sixth chapter discusses the various applications of data science. The seventh chapter covers data security, and the eighth chapter discusses the future of data science. The book is written in a clear and concise style, making it easy to read and understand. It is a valuable resource for anyone interested in the field of data science.

The book is a comprehensive guide to the field of data science, covering all the major topics and providing a solid foundation in the field. It is a valuable resource for anyone interested in the field of data science, and it is a must-read for students, researchers, and professionals alike. The book is written in a clear and concise style, making it easy to read and understand. It is a valuable resource for anyone interested in the field of data science, and it is a must-read for students, researchers, and professionals alike.

SCIENTIFIC



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Handwritten signature and notes in blue ink:
Dr. B. Gangathara
11/11/2023
11/11/2023
11/11/2023

Fundamentals of Management



Gaddam Rahul Paul
U.Fanidara Baradwaja Sharma

Fundamentals of

Management is a process of planning, decision making, organizing, leading, motivation and controlling the human resources, financial, physical and information resources of an organization to reach its goals in an efficient and effective manner. Every course has to examine from the management perspective to absorb more insights into its relevance and public utility. When Engineering students learn management concepts and practices and apply these to their core engineering curriculum, they explore the fathoms of their subject in terms of value proposition, value creation, and value capture. This is the very purpose of teaching Fundamentals of Management as a core subject to all engineering branches. The book is written considering the academic requirements of B.Tech (all branches) course of JNTUH. Considering the previous examination pattern and in the context of the future examinations, this book can be referred to as the guide, consisting of a brief of the topics for the preparation of the students.

Goddam Rahul Paul



Mr. Goddam Rahul Paul is a young and dynamic faculty at Sri Indu Institute of Engineering and Technology, Affiliated to JNTUH. He is also designated as the Editorial Board Member for the DORF Journals. He is qualified in 15-Ser – 2019 and done his dual masters in MBA (Human Resources and Marketing) and MSc (Psychology).

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Gaddam Rahul Paul
U.Faizudara Baradwaj Sharma


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P. R. Indu Nagar, 501 518

Syllabus

Fundamentals of management

B.Tech. III Year I Sem.
Course Code: SM504MS

LT/PC
1003

Course Objective: To understand the Management Concepts, applications of Concepts in Practical aspects of business and development of Managerial Skills.

Course Outcome: The students understand the significance of Management in their Profession. The various Management Functions like Planning, Organizing, Staffing, Leading, Motivation and Control aspects are learnt in this course. The students can explore the Management Practices in their domains area.

Unit - I

Introduction to Management: Definition, Nature and Scope, Functions, Managerial Roles, Levels of Management, Managerial Skills, Challenges of Management, Evolution of Management- Classical Approach- Scientific and Administrative Management, The Behavioral approach, The Quantitative approach, The Systems Approach, Contingency Approach, IT Approach.

Unit - II

Planning and Decision Making: General Framework for Planning - Planning Process, Types of Plans, Management by Objectives, Development of Business Strategy, Decision making And Problem Solving - Programmed and Non-Programmed Decisions, Steps in Problem Solving and Decision Making, Bounded Rationality and Influences on Decision Making, Group Problem Solving and Decision Making, Creativity and Innovation in Managerial Work.

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Colombo 11

Unit - III
Organization and HRM: Principles of Organization: Organizational Design & Organizational Structures: Departmentalization, Delegation, Empowerment, Centralization, Decentralization, Recentralization; Organizational Culture, Organizational Climate and Organizational Change. Human Resource Management & Business Strategy: Talent Management, Talent Management Models and Strategic Human Resource Planning; Recruitment and Selection; Training and Development; Performance Appraisal.

Unit - IV
Leading and Motivation: Leadership, Power and Authority, Leadership Styles, Behavioral Leadership, Situational Leadership, Leadership Skills, Leader as Mentor and Coach, Leadership during adversity and Crisis Handling Employee and Customer Commitment, Team Leadership, Motivation - Types of Motivation, Relationship between Motivation, Performance and Engagement, Content Motivational Theories - Needs Hierarchy Theory, Two Factor Theory, Theory X and Theory Y.

Unit - V
Controlling: Control, Types and Strategies for Control, Steps in Control Process, Budgetary and Non-Budgetary Controls, Characteristics of Effective Controls, Establishing control Systems, Control Frequency and Methods.


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Mumbai
Date: _____

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Lakshmaiah Deyadi
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Minimization of Dynamic Power Through Structural Changes & Vt Techniques

low power cmos circuits



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Starter Culture Technology: Fermented Foods

D. Prema Latha, S.M. Reddy, K.S. Yana, and Pogaku Ravindra

Introduction

Since the ancient time's fermentation technology have been in common practice of man for the production of different products such as cereal, wine, alcohol, idly, dosa etc. In the initial stages these food products were produced without the knowledge of scientific principles involved in the process of product formation. It was due to spontaneous microorganisms present in the environment or container. Traditional fermentations were captured to store the food products (Holezafel 2002). This concept was originated after later half of nineteenth century when the fermented products developed after pasteurization (Kalancharonee and Fitzgerald 1994). The role of microorganisms in food fermentations were realized (Caplan and Ficoaldi 1999; Balduo et al. 2003) with passing of time the clear picture of the fermentation and role played by starter cultures or microorganisms.

In the past fermentation process was utilized to enhance the quality of product (Holezafel 1997, 2002). Final Product was the result of interaction of flora of the raw material and the environmental conditions prevailing such as temperature, oxygen, water activity, pH etc. the microorganisms beside selecting necessary enzymes they also produce beneficial substances which are needed for production of fermented products (Kalancharonee and Fitzgerald 1994) such as lactic acid, alcohol, organic acid etc. (Ranson 2002). These products act as preservatives, enhance flavor and increase shelf life of product (Nout 1991). Preservation of food is man's struggle for survival since prehistoric times was to ensure required food for all the time. Microbial fermentation not only preserves the food but also contributes the taste, aroma, flavor, stability etc. (Hamrick 1990).

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2	DR. S.SURESH SUBRAMANAIAN	DESIGN AND DEVELOPMENT OF RF ENERGY HARVESTING LOOP ANTENNA	IEEE	978-1-7281-5198-4	https://ieeexplore.ieee.org/abstract/document/9074402
3	DR. S.SURESH SUBRAMANAIAN	A DETAILED INVESTIGATION ON REDUCTION OF FALSE POSITIVE RATE IN BREAST CANCER DETECTION	IEEE	978-1-7281-5198-4	https://ieeexplore.ieee.org/document/9074382
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Academic Year 2018-2019

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Academic Year 2015-2016

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Sl. No	Name of the Faculty Author	Title of the paper	Name of the Publisher	ISBN Number	URL
1	DR. I.SATYANARAYAN A	COOLING AND HEATING OF A SUBSTANCE USING VAPOUR COMPRESSION REFRIGERATION CYCLE	MREC		
2	DR .I.SATYANARAYAN A	THERMAL ANALYSIS COMPARISONS ON NATURAL VENTILATED OPEN POLY GREENHOUSE TO CLOSED POLY GREENHOUSE	MREC		
3	DR. I.SATYANARAYAN A	DESIGN AND ANALYSIS OF TORSEN DIFFERENTIAL	MREC		
4	DR. I.SATYANARAYAN A	CFD ANALYSIS ON TURBULENT CONVECTION HEAT TRANSFER WITH FE3O4 NANOPARTICLES IN WATER	MREC		

5	DR.I.SATYANARAYANA	ADVANCED FUEL OPERATED INTERNAL COMBUSTION ENGINES	MREC		
6	DR.I.SATYANARAYANA	PERFORMANCE TEST ON SINGLE CYLINDER FOUR STROKE DIESEL ENGINES BY USING CONVERGENT DIVERGENT INLET MANIFOLD	MREC		
7	DR.I.SATYANARAYANA	DESIGN AND THERMAL ANALYSIS OF WARM (COLD) FORGING PROCESS AND DIES	MREC		



Principal
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Compact Micro Strip Fed Koch Fractal Monopole Loop Antenna For Multiband Performance

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Abstract— A compact Koch fractal monopole loop antenna with SRR using strip feed for multiband application is presented. The proposed antenna is designed on a dielectric FR4 substrate with the dimensions of length 37 mm x Width: 47 mm x h: 1.6 mm with the proposed slot of dimensions L1: 3.5 mm x W1: 4.92 mm and L2: 6W1 - 2.57 mm (FR4/FR4), 4.72 GHz - 6.6W1 (FR4/FR4) and 8.7 GHz - 10.47 GHz (FR4/FR4) resonate for RCS/PC/EMTS, EMAT, MIMO, Radar, navigation respectively. The proposed antenna exhibits good return loss in the simulation and the symmetrical geometric structure of the antenna provides dual directional radiation pattern.

Keywords—Micro strip fed, Koch fractal, monopole antenna, multiband characteristics.

I. INTRODUCTION

Fractal describe galaxies and some other natural phenomena. Fractal geometry and its concepts have become useful tools in most of the natural sciences such as physics, chemistry, biology, geology, and even in material science. The rapid development in the wireless communication field requires design of small antennas that can be used in three dots one frequency band. The self-similarity and space filling property of fractals makes them especially suitable for multi-frequency antenna design [2-3]. Monopole antennas [4]. In this paper, a micro-strip fed Koch fractal monopole loop antenna for multiband application is designed and analyzed. This antenna consists a fractal monopole of the second iteration of Koch geometry. The radiation pattern, which is simulated, adds additional value for the proposed antenna design.

II. KOCH FRACTAL ANTENNA DESIGN

Initially the curve of Koch curve is obtained by dividing a line of any given length into 3 equal parts. Figure 1 shows the achievement of Koch's curve which clearly shows the transformation of a line into equilateral triangles. Then the triangles which are identical to each other are placed one above the other in such a way that all their corners coincide with each other [5]. The same procedure is repeated for the entire triangle in all of its sides. This results in the formation of new triangles and the same procedure is applied to the newly formed triangles too which end up with the generation of Koch loop of higher order. The formation of various Koch fractal loop is as shown in Figure 2 and Figure 3 (a). The antenna is in the xy plane and the normal direction is parallel to the z axis.

Duroid 3680 is used as the dielectric material with $\epsilon_r = 2.2$ and the dimension of the substrate is 57 mm x 48mm x 1.6 (mm).



Figure 1 Koch curve formation



Figure 2 Koch loop generation with higher order.

The radiation element is a hexagonal [6] loop with the size of each side is 16.6mm. A 50 Ohm micro-strip line with feed length (L₁) = 17 mm and width (W₁) = 4.92 mm is used as excite the antenna. A gap of 3.5 mm is maintained between antenna and ground plane for achieving required impedance bandwidth. The ground plane dimension is 16.2mm*16 mm. The design and simulation results are analyzed using HFSS [8] (High Frequency Structure Simulator) for various configuration of Koch fractal geometry (Zeroth order to Second order). The basic structure could not achieve the impedance bandwidth properly. So, iterations are carried out. During the simulation, it has been observed that, in the second iteration the multiband frequency is obtained with required band width and return loss. In order to improve the return loss of each band [7], a slit with the dimension of [L] (5.5 mm x [W]) 4.92 mm is removed from the ground plane. The structure of the slit in the ground plane is shown in the Figure 3(b).

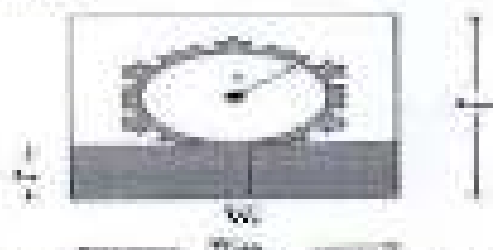


Figure 3 (b) Schematic of Micro-strip fed Koch-Fractal monopole antenna

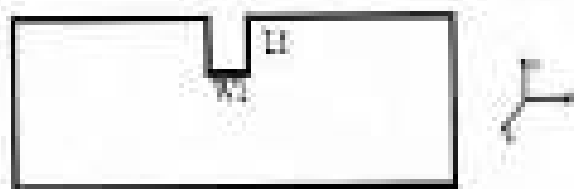


Figure 2 (b) Ground structure of proposed antenna

III. RESULT AND DISCUSSION

The fractal based Fig.4 and Fig.5 has a impedance band width as large as 51.8MHz and return loss (15.93-19.66 MHz) band with center frequency of 1.90GHz.

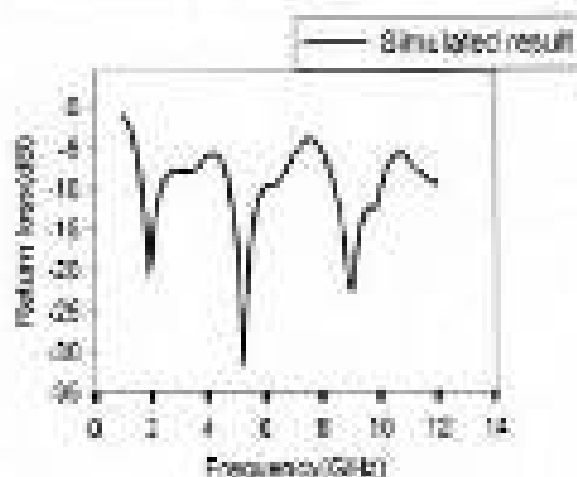


Figure 4 Simulated antenna

The upper band has the impedance bandwidths of 100% and covers the RADIO NAVIGATION band (8550MHz-15000MHz) with center frequency of 9GHz. In all the frequency band range, we can see that the VSWR is maintained less than 2.

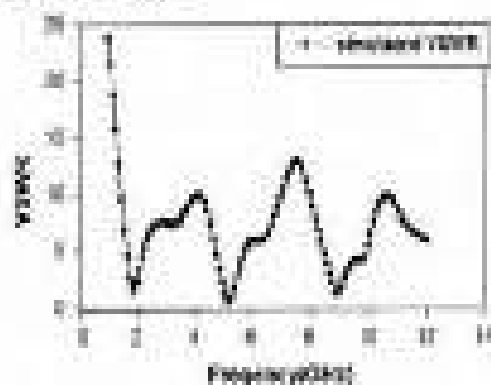


Figure 5 Simulated proposed antenna

The figure 6 (a) shows the nature of H and E plane radiation pattern are nearly Omni-directional and dumbbell shaped respectively. Here, the simulated radiation pattern in H plane and E plane is more likely monopole radiation pattern for all bands.



Figure 6 (a) Simulated radiation pattern of y-z plane

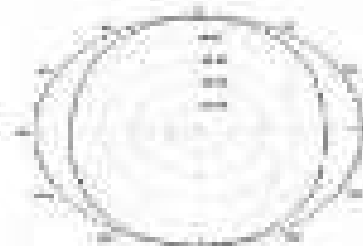


Figure 6 (b) Simulated radiation pattern of x-y plane

IV. CONCLUSION

The Koch fractal monopole loop antenna with ground slot for multi band characteristics for the second iteration has been designed and results are analyzed. The triple band covers 1.9GHz, 9GHz, and 9GHz, which is useful for the application of DC/DC/MTS, IEEE 802.11, and RADIO NAVIGATION respectively with impedance bandwidths of 100%, 100% and 100% respectively. The radiation pattern for azimuth and elevation angle for the designed structure is more likely monopole radiation pattern. The proposed multiband micro strip fed Koch fractal monopole antenna is simple to design and easy to fabricate.

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Design and Development of RF Energy Harvesting Loop Antenna

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Abstract— Radio waves from air are captured by Radio Frequency (RF) signals and retransmitted by different remote systems. To improve the range of wireless energy effectively, RF Energy Harvesting System is designed for the applications of smart devices, battery chargers, and remote devices. The fundamental components of the low power harvesting scheme are the antennas. High-Impedance Surfaces (HIS) have been widely employed in order to upgrade the field of the antenna. Elimination of such meta-surfaces is the in-phase back of an incident wave, which is based on a flat metal frequency dimension to act as the Artificial Magnetic Conductor (AMC). The research proposes a loop antenna realized on an AMC for RF energy harvesting to operate at ISM frequency of 915MHz.

Keywords— Radio Frequency, High-Impedance Surface, Artificial Magnetic Conductor

I. INTRODUCTION

The power harvesting scheme is utilized to capture and collect the ambient power that should be converted into electrical energy. It is also called energy scavenging. Typically, there are abundant energy resources available around us like wind, solar, thermal and mechanical but their energies are not used as electrical load as a power supply. The different form of energy is available in our surroundings the micro-energy will be to differentiate them and analyze what form that would be most beneficial for Radio Frequency [1]. The radio waves and static RF sources are available like broadcasting digital TV signals in the band of 300MHz and download mobile phone services in the band of 915MHz.

Loop antenna is used to design the band of 90-MHz and 175-MHz because of its high input impedances, it may be used in transmitter section and added advantage to convert RF signal to Direct current [2]. To enhance the efficiency of the receiving section of the antenna, a reflector also required because of the sensitivity in both the broadcast directions of the loop antenna. Such type of reflectors made up of an electrically periodic metal gap (EMG) for the purpose of achieving high gain antennas and low profile.

Nowadays researchers are showing too much interest on doing research in electromagnetic artificial materials [3], for example Double Negative metamaterials, LHM [4], zero/low-loss positive photonic periodic structures [5]. These types of structures are called as metamaterials and characterized by engineered that periodic dielectric substrates and many constitutive parameters. These Metamaterials shows

the novel features in electromagnetic that cannot take place in nature, and have correlations with large area in application of propagating antenna and fields.

To enhance the performance of patch antenna because of the band-gap of surface waves and suppression, LHM resonator is integrated with patch antennas. In order to achieve low profile structures, they additionally etched on the ground plane of real and equal antennas.

Metamaterial having the interesting properties is that special interesting feature is the reflection phase. The electric field is reflected at the reflecting surface with phase is called the reflection phase. At reflecting surface, the reflection phase is considered with respect to the phases of the incident electric segment. The loss magnetic and electric conductance pose a reflection angle of 180° for a normal wave plane wave that often the magnetic conductor, which do not exist.

Metallic-dielectric HIS have been proposed [4], the artificial magnetic conductor (AMC) is equivalent to the illumination from the incident plane wave. This type of structure is reflected with a reflection phase of 0° for the incident plane wave and it is complementary to the electric conductor which introduces 180° phase shift for the incident wave.

II. PROPERTIES OF ARTIFICIAL MAGNETIC CONDUCTOR

The 2-D artificial ground plane EMG surface consists of the top surface which is aligned with metal patches periodically and bottom surface is solid ground plane [5]. Here all the metal patches are connected with the ground plane via the vertical post known as the meandered structure. Because of the periodic condition in the EMG, surface waves have been stopped to transmit frequency in the band gap.

A significant feature of metamaterial property is that incident radio waves are reflected without changing the phase in the band gap at the frequencies, as if the surface was considered a magnetic conductor. In that case, it is known as the AMC surface and shown in Fig.1. In general, the distance between the antenna and metal reflectors must have the quarter-wavelength to eliminate any forms of destructive interference between the direct and reflected waves because of phase reversal of 180° immediately after the reflection. In the traditional, AMC surfaces are very close to the antenna because of the reflected wave phase does not change for the incident plane wave. Hence the high radiation efficiency is

A handwritten signature in green ink is written over a purple circular stamp. The stamp contains the text: 'Sri Aurore Institute of Engineering & Technology, Coimbatore-641 019'.

A Detailed Investigation on Reduction of False Positive Rate in Breast Cancer Detection

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Abstract— Breast cancer is known as the most common form of cancer in Indian women and is about 14% of women cancer. Thus a detailed investigation is done to improve the level of screening, medication and accuracy of breast cancer detection by Computer Aided Diagnostic systems. The initial level of screening of breast can be performed using thermography and then the progression can be provided for accuracy of further screening using mammography for cancer detection. If the presence of breast cancer is confirmed using mammograms, the same patient can be diagnosed using biopsy images for further confirmation of the presence of breast cancer. Thus a detailed investigation on employability of various modalities for the breast cancer screening is conducted to minimize the false positive rate in breast cancer detection.

Keywords—Breast Cancer, Biopsy, Thermography, Mammography, Biopsy.

1. INTRODUCTION

According to World Health Organization, breast cancer is found predominantly which affects the lives of 2.1 million women each year[1]. The survey shows that one third of women have been subjected to the mortality rate and the breast cancer rate because the cancer is also increased to 17%[2]. It is obvious that early detection of progression and all type of breast cancer can have lives of many women across different countries.

Global breast cancer also increases everywhere in the world. The incidence of breast cancer are diagnosed each 27 minutes which makes it one for women in 31 from the breast after one minute. In the year 20 years, other 47 million novel instances are supposed to be throughout and other 13 million individuals will perish due to the disease [3].

Figure 1 shows that the disease is a most predominant type of cancer in India [4]. Correlation of Indian Industry (CII) report says that young Indian women are more vulnerable to breast cancer [1]. Indian Council of Medical Research says that out of 100,000 women, 25.8 women are

having cancer in the year 2018 and there is an expectation that the number will rise to 33 in the year 2024[5].

Breast cancer is developed when the cells grow in an uncontrolled manner[7]. This uncontrolled growth of cells can be treated in many ways as drugs. If the growth of cells expanded to nearby regions, then the tumor is malignant and if the other hand, if restricted to a specific portion, the tumor is benign. No doubt, an identification of malignant or benign helps the clinicians to proceed with further treatment. Thus the challenge lies in building an intelligent system to detect whether the patient suffers from breast cancer.

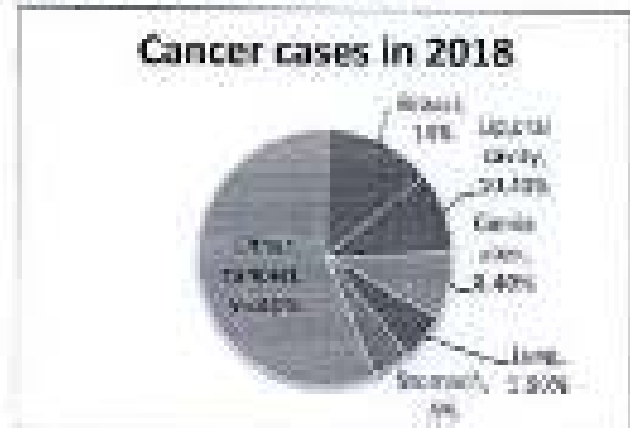


Fig. 1. Cancer cases in 2018

The doctor recommended that women with age more than 40 years should undergo periodic breast cancer screening to confirm the availability of breast cancer at its earlier stages [8]. Early detection is possible before the woman notices physical symptoms using better screening techniques. This early detection of cancer enables patients to provide less time for the treatments and save the lives of patients.

This research is arranged as follows. Part II explains the literature review concerned to some modality methods that are applicable in the process of detecting breast cancer. Part III



presents the methodology that can be adopted to reduce the false positive rate in breast cancer detection. Section IV concludes the paper.

II. BREAST CANCER DETECTION TECHNIQUES

The survey shows that the best widely used screening mechanism for breast cancer is mammography. The abnormality is found in visualization easily with mammography images. The false low cancer detection with mammography screening such as exposure to repeated radiation, painful and uncomfortable for breast is compressed with a vacuum [9].

There is a solution to the drawbacks of mammography, the diagnosis mechanism can be integrated with thermal screening using thermography, which is non-invasive and non-contact [10-11]. Thus the pain and discomforts due to compression of the breast is removed. Also, it is not exposed to any radiation. However, a solution would mammography screening for women just before their menstrual cycle including that period, breasts become more soft and tender than other normal days [12]. But thermography can be applicable at any time. The breast cancer using infrared technology, when the skin of the breast with different temperature levels and parts there is a heat map [13]. The higher level of temperature after the death of presence of cancer. Thus the thermography can be used as an initial screening mechanism which starts the accurate strategy mammography for further test.

Though the drawbacks of mammography are addressed with thermography, the other major drawback is identified. The survey shows that 10% of mammography screening gives false positive results [14]. Many researchers are going on researches to reduce the false positive rate in breast cancer detection. Thus it can be preferred to integrate breast biopsy after mammography screening to monitor the presence of cancer further. Breast biopsy is considered as the best method to determine whether the suspected area is subjected to cancer.

III. RESEARCHERS TO REDUCE FALSE POSITIVE RATE IN BREAST CANCER DETECTION

The advantages of Deep Learning in Computer Aided Diagnostic systems, namely, the best accuracy in predicting even complex breast performance and less computation time spend in pipeline throughout the model and every researcher focus their attention in applying the different deep learning methods and in a real world tasks.

Yasir Khatib et al used deep learning with convolutional neural network used in diagnosing the illness using Mammograph RLAS datasets. They applied preprocessing techniques such as Watershed segmentation, Adaptive Mean Filter and Canny's based segmentation and achieved 98% accuracy [15]. Abdulrahim M. Alabdulhadi and his colleagues for classification of breast cancer with thermography images [16].

Mohammed Najmeddine Al-Dabbas got accuracy of 80.8% in classification and used deep learning with convolutional neural network [17]. Mei Huan Yap applied Le-Net, FC7L, Alex-Net and U-Net and got 88% accuracy [18]. Tarata Aranjana used ImageJ image and achieved 89.8% accuracy with convolutional neural network [19]. The classifier proposed by R. G. Ghargade are 89% sensitivity and 89-ELM that is considered to be 88% accuracy [20]. Mulualem Tessema got 89% accuracy in pictures and achieved 98% accuracy based on SVM classification [21].

Kavita A. Sahasrabudhe et al validated the breast cancer prediction results with 4 publicly available datasets, namely, Kvasir16, Kvasir18, Mammography and Dermatology datasets and obtained 98.12%, 97.87%, 94.96% and 87.10% respectively [22]. They adopted an ensemble of DenseNet, MobileNet and VGG19 for feature extraction and used multi-layer perceptron model for.

Das M. V et al used transferable domain and applied CNN with boosting tree classifier for detecting breast cancer [23]. An ensemble of biological feature was performed on Breast In contrast by Suresh Prakash et al [24]. Ramesh Jay et al worked with histopathological images in CMAR dataset and used probabilistic classifier [25]. Zhenyi Han et al used GoogleNet and Xception [26] and Babak Ghahramani for work of a dual channel CNN for classification [27]. Fathy Alexander Sparked classification of breast malignancy cases with the help of AlexNet architecture [28].

Since many researches focused in suggesting solutions for breast cancer, the survey is done extensively for breast cancer screening mechanism and summarized as given in the following table.

Table 1. Comparison of Breast Cancer Screening Mechanisms

Technique	Accuracy	Specificity	Method	Remarks
Mammography	98%	98%	Computer Aided Breast	Produce the output
ECG	99%	98%	Feature extraction method and dynamic time warping	Screening in multiple ways can be done better
FTT	96%	96%	Small dataset of unknown size	Attention of breast can spread over the part of body
U-Net	97%	97%	Automatic threshold based	Accuracy is also good
Transfer learning	94%	97%	Computer for breast	Screening and diagnosis
Ensemble machine learning (7%)	94%	97%	Deep-Learning ensemble with a better	Put in use after the initial testing
Thermography	98%	98%	Preprocessed images extracted by the data mining	Screening
General Purpose Thermography	87%	87%	Thresholds extracted by the data mining	Screening

The table clearly depicts that Mammography is the best screening mechanism for breast cancer. Thus the survey focus the research activities towards different technologies available for breast cancer using mammography with incorporating thermograph in the initial stage and image of the later stage. It is illustrated in figure 2.



Fig. 2. Methodology to reduce false positive rate in breast cancer detection

IV. Discussion

Early detection of the disease shall drastically the rate of mortality in women. Though the incidence rate is expected to increase every year, we have seen initiatives to reduce the mortality rate. Some practical screening steps to breast cancer at an early stage, proceeding with mammography is one investigation used to analyze and effective and practical screening. This facilitates the asymptomatic patients to undergo further diagnosis, only when required thereby preventing unnecessary costs and painful mammography screening after initial stage itself. Though mammography is accepted as a standard procedure for breast cancer screening, the survey shows that the false positive rate is also high, hence to minimize the potential false positive rate, additional screening using deep learning is done for further confirmation of presence of cancer. Thus our findings suggest the inclusion of the preprocessing and post processing of standard mammography to improve the breast cancer of women. Since the increase in worldwide rate of breast cancer is beyond our control, in turn we can take some initiatives to reduce the mortality rate in the breast cancer.

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Evaluation of Pullout Resistance of Reinforcing Strips Embedded in Cement Modified Marginal Backfill of Mechanically Stabilized Earth (MSE) Walls

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Abstract. Pullout resistance of reinforcing materials is one of the lateral stability criteria in Mechanically Stabilized Earth (MSE) structures. The pullout resistance is normally calculated using the friction coefficient equal to unity. Finding the angle of internal friction in case of rigid reinforcing materials. For frictionless rigid reinforcing strips, the friction coefficient cannot be directly observed in the direction, several attempts have been made by various researchers to evaluate suitable pull-out parameters. However, the complexity increases with the use of marginal backfill soils because, suitable soils are not available. The present work is an effort to determine the pull-out parameters of different types of reinforcing materials embedded in marginal backfill and without and with cement modification. The testing was carried out in laboratory using a test rig and in the field by measuring model settlements. The pull-out factor F^* and μ as suggested by FHWA were evaluated using the test data and compared them with those obtained using conventional sand backfill.

Keywords: Mechanically Stabilized Earth - Marginal soil - Cement modification - Lateral settlements - Pull-out resistance

1. Introduction

Mechanically Stabilized Earth walls gained prominence in place of conventional retaining walls in view of the inherent advantages of such systems. These walls consist of free draining frictional backfills reinforced with suitable materials having connection to facing units. Though the criterion for standard backfill is specified, at several work sites, such materials are not available in abundance and hence, the use of locally available marginal backfills is reported by various investigators (Sami 2006; Long *et al.* 2007). With this wall type standard backfills, low failures of reinforced soil structures were reported (Farrag 2004; Abu-Farrah *et al.* 2007). It is reported that majority of failures were attributed to the use of poor quality backfills with high plasticity and low permeability (Bergado *et al.* 2001; Bergado *et al.* 1999; Ghodrati



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Effect of Eccentric Load on Footing Resting on Planar Geosynthetic Reinforcement

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Abstract. A structural foundation transfers all the loads from the super structure to the soil underneath safely. Due to earthquake or wind loading, the line of action of total load from the superstructure does not pass through the center of gravity of the footing resulting in eccentric loading. The compensation for reduction of the bearing capacity by increasing the size of footing is uneconomical. Geosynthetics can be used to improve the bearing capacity of the footing. In the present study a series of tests were conducted and an exploratory study is made when Geotextile (Wholec G-10) is used as planar reinforcement to improve the bearing capacity of soil. Also the improvement of bearing capacity in accordance to eccentricity was studied. The test results indicate that there is significant improvement in the bearing capacity and reduction in the settlement by providing planar reinforcement and the eccentricity is 0.1 times the width of footing.

Keywords: Fiber reinforcement · Eccentric loading · Bearing Capacity · Soil

1 Introduction

Earth provides the ultimate bearing support for all the civil engineering structures like buildings, bridges, reinforced fills, various embankments and concrete slabs. Hence the behaviour of the bearing ground defines the stability of whole structure. Hence the structural foundation should be able to transfer the load from super structure such that the oversteering of soil underneath is avoided besides preventing the excessive settlement of structure. If line of action of the load does not pass through the center of loaded area, the contact area decreases. The reduction in contact area results in the reduction of effective width of footing resulting in decrease in bearing capacity of supporting soil [2]. Eccentric loading of footing can occur in extreme column loading or due to earthquake or wind loading.

Several studies conducted on eccentrically loaded footing and bearing capacity improvement using geosynthetics and found that significant bearing capacity improvement can be achieved using geosynthetics so that the reduction of load-carrying capacity due to the eccentricity of the footing can be compensated [1, 3, 4]. Geosynthetics can be used to improve the load carrying capacity [5, 6]. Though there are many types of geosynthetics, a non-woven geotextile is used in this study to find the effect of

Study on Interaction of Reinforcing Strips Embedded in Cement Modified Marginal Backfill

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Abstract. Full-scale interaction of reinforcing materials is one of the broadest and the critical in Mechanically Stabilized Earth (MSE) construction which is normally achieved using the facing reinforced equal to length. Among the type of facing blocks is case of rigid reinforcing materials. For flexible/reinforce reinforcing strips, the facing reinforcement is usually flexible and in this direction, several attempts have been made by various researchers to evaluate suitable pull-out parameters. Further, the complexity increases with the use of marginal backfill soils where suitable soils are not available. The present work aims to determine the pull-out parameters of different types of reinforcing materials embedded in marginal backfill soil reinforced with cement modification. The testing was carried out in laboratory using a test box. The pull-out factors P and n as suggested by BS594 were determined using the test data.

Keywords: Mechanically Stabilized Earth - Marginal soil - Cement modification - Geotextile reinforcement - Pull-out parameter

1 Introduction

MSE walls gained prominence in place of conventional retaining walls in view of the inherent advantages of such systems. These walls consist of face retaining structural backfills reinforced with suitable materials connected to facing units. Though the criterion for standard backfill is specified, in several work sites, such materials are not available in abundance and hence, the use of locally available marginal backfills is reported by various investigators [1, 2]. With this shift from standard backfills, the failures of reinforced soil structures were reported [3, 4]. It is reported that majority of failures were attributed to the use of poor quality backfills with high plasticity and low permeability [3, 4]. Few attempts were also made by investigators to overcome the ill-effects of marginal backfills by the use of electro-chemical dechlorination [5], preloading and unloading and the use of cementitious materials [15]. The researchers have also studied the efficacy of cement modification of marginal soils for their use in reinforced soil structures [17] by conducting load tests on metal embankments. In order to evaluate the design parameters for internal stability in case of cement modified backfills, an

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Resilient Modulus of Unsaturated Soil – A Comprehensive Review

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Abstract. Unsaturated soils are predominant in semi-arid regions which has the property to show moisture changes. Compacted pavement soil is in the intermediate state thus exhibiting the tendency to weaken or strengthen with varying water content. With ever changing vehicle loading conditions, the necessity to assess the dynamic behavior of pavement materials in the design of pavements have been realized in numerous research works. Thus the elastic models of pavement material have been incorporated as a key parameter in MEVDG, to assess condition of the compacted soil present the ability to exhibit changes in strength with associated moisture variation. This paper is intended to summarize the findings and the necessity to calibrate the pre-compaction moisture variation and their influence on the mechanical property of pavement materials. Furthermore, the importance of moisture suction and soil test methods have been highlighted in many developing countries due to inadequate information. Hence, the present work aims at bringing qualitative reference to be updated on available available models.

Keywords: Unsaturated soils · Soil suction · Degree of saturation · Compacted subgrade · System modulus · Revised test method

1 Introduction

With the rapid progress of cities, the urban further inland is developed to accommodate the increasing population. The behavior of soil in areas of low or moderate rainfall fall in the category of unsaturated or partially saturated soils. The soil is represented as a two-phase system with solid and liquid, whereas the unsaturated soil mechanics considers the soil as a two-phase system in most of the cases with air and air-water interface in case [2]. The geotechnical engineering practice dealing with these soils is still ingrained in conventional soil mechanics [1]. Research infrastructure is predominantly for soil as saturated soils and the compacted subgrade soil also resembles that of saturated soil. Thus the ability of soil to contract or desaturate with varying moisture content leads to show the property of the porous material which in turn influences their mechanical behavior.

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Journal of Materials Science: Materials in Civil Engineering

Evaluation of Material Properties and Abrasive Resistance of Tantalum Carbide-Based Hardox Steel for Construction Purpose

Authors: [S. S. Ghosh](#), [S. Ghosh](#), [S. Ghosh](#)

Journal of Materials Science: Materials in Civil Engineering, Volume 22, No. 1, 2011

Keywords

Hardox steel; Abrasive resistance



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Abstract

In recent days, the demand of high strength and wear resistant steel has been increased. The

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Effect of Diethyl Ether on LHR Engine Characteristics using Papaya Methyl Ester-Eucalyptus Oil Blend

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The present experiment deals the study of addition of diethyl ether on the performance and emission characteristics of LHR engine using papaya methyl ester-eucalyptus oil blends. The test blends are CPME80Eu20 (Carion papaya methyl ester 80% and Eucalyptus oil 20%), CPME80Eu20+10%DEE and diesel. The optimum results we get with presence of DEE in CPME80Eu20 in LHR engine. The presence of DEE creates a lean mixture and its low viscosity, high octane number and volatility improves performance for a large degree. The graph depicts that addition of 10% diethyl ether gives the best performance in BSEC, BSFC, BTE and emission wise when coupled with LHR engine. Most notably NO_x emission rate is decreased by the presence of the DEE and BSFC is brought under acceptable limit. BSEC decreases in CPME80Eu20+10%DEE and better the performance of diesel in conventional engine. It also add to improve the cold flow properties of the CPME-eucalyptus oil blend.



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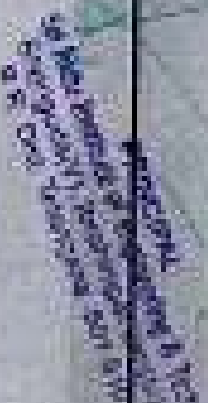
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Dr. Ramesh Babu
Organizing Secretary


Dr. Ramesh Babu
Convener & Principal



Testing and Implementation of Smart Brake Pedal System with Signal Diagnostic and Failure Detection

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Abstract— Brake pedal signal diagnostic is an important feature in driver assistance system. The primary aim of implementing this feature is to avoid any false detection of brake pedal signal. It also provides information to the driver regarding brake switch failure. Along with proper implementation it is necessary that the feature is tested thoroughly in order to make it fail proof before it's stage is rolled. The code for this feature in vehicle is implemented in language: National Instruments Real Time (NI-RT) and Polyspace tools are used for testing of this feature. The project focuses on implementation and testing of Brake Pedal Signal Diagnostic in vehicle.

Keywords— Brake pedal signal diagnostic, Brake switch signal, Fault Mode, Brake Light switch diagnostic, Low fuel condition, High fuel condition, vehicle.

I. INTRODUCTION

Brake pedal signal has to be correctly diagnosed in vehicles. It alerts the driver in case of situations when the brake switches have failed or wrong signals are being generated. It is an important safety aspect in each vehicle. Accurate detection of brake signals as well as any failure in brake vehicles can avoid any kind of casualty (Chavan et al., 2013). This feature aims at detecting brake pedal signals under high fuel and low fuel conditions. Requirements are given by the customer, depending on which the spec. author will develop the specification document. Once the specification document has been developed the code to implement the same has to be developed. In the next phase the developed code has to be tested. In the testing phase any errors in the code will be caught. The test reports will be generated at the end. These reports will be analyzed in order to confirm the errors are valid ones or not. In case of valid errors, the code needs to be corrected and again testing of the same is carried out. Once the testing reports are without any Defects the code will be sent to the validation phase. After the validation of the code it will be sent to the production phase. The software development follows V model. This model defines the different stages of development. The name is given because of the typical process that is being followed during the software development phase. This model shows how each stage in the development has been related to each other thus giving a clear picture of software development.

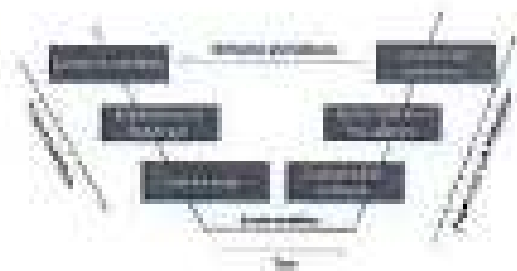


Fig. 1 V-model of software engineering process

II. V-MODEL

Fig. 1, shows the V-model that is being followed in an automotive industry. As said earlier it shows the different stages of software development (i.e., Babar et al., 2016). The different stages are shown in the figure and the explanation is given below.

A. Requirement phase

1. Requirement analysis

In this phase the customer requirements for brake pedal feature were collected. This was done by examining the needs of client. This document contains the system selected feature, functionality, data, performance etc. (Liu and Peng, 2002).

2. System design

In this phase the requirement document is studied and ideas are proposed to implement the same. If any of the requirements can't be implemented, then it has to be intimated to the customer at this phase.

3. Module design

This phase is said to be low-level design. Here the design of the system is split into smaller units. This will help in making the programmer aware about the requirement of each smaller unit. The coder then after understanding the requirement can start programming.

B. Validation phase

1. Unit testing

The main code that has been developed is divided into many small modules or programs that can work independently. These are tested to find any bugs. These bugs can be resolved at the basic level and later the modules



tested, results can be integrated (Alexander Sebastianam et al., 2017).

2. Integration testing

This test will give confirmation that the individual codes can be integrated and are able to communicate without any issues (Johannes et al., 2017).

3. System testing

This testing plan will be developed by the client. The test is made to make sure that the developed functionality meets all the requirements of client. This test confirms that both functional and non-functional requirements have been met (Vishal et al., 2017).

4. User acceptance testing

This test plan will confirm that the functionality that has been developed and validated meets client requirements and is ready to be used in real-time (Almad et al., 2008).

III. IMPLEMENTATION

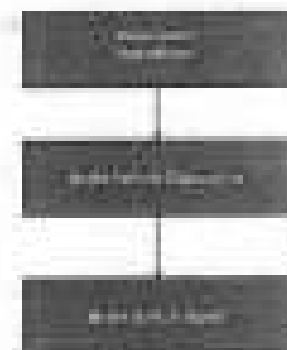


Fig. 3. Code Implementation

The code for this feature has been developed in three stages and its flow is depicted in "Fig. 3"

A. Brake switch operational

In the first stage a check is done to make sure that the brake switches have been powered. This check forms the base of this functionality (Miran Jo et al., 2017). Only when the brake switches are powered, signals can be generated and can be used for further analysis. Basically two vehicles are used; the second vehicle is used as a backup. In case the brake switch is not working properly or has failed due to some reasons and the driver is unaware about the same then the backup vehicle can detect this. The logic is that both the vehicles should give the same signal at one instance. If this doesn't happen then the vehicle needs to be checked. Both the switches are given recovery time. Recovery time is the time given to the sensors to detect and send the signals. If the signals are not received even after the recovery time, then a check on the brake switches needs to be performed.

B. Brake switch diagnostic

The second stage that is brake switch diagnostic can be further sub divided in three main stages as shown in

"Fig. 3". Mainly brake diagnostic condition, brake light switch diagnostic and brake safety switch diagnostic.

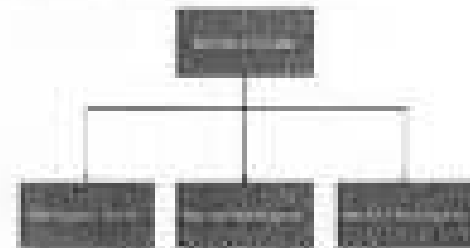


Fig. 3. Brake Switch Diagnostic

1. Brake diagnostic condition

Several signals are built depending on vehicle speed and pedal position thresholds in order to perform pedal diagnosis. Sensors will sense the speed of the vehicle and the angle of accelerator pedal. Based on both these parameters signals are generated which are further used for diagnosis of the pedal signal. If the automatic brake or the ESP (brake pressure control) are activated by the Vehicle Stability Assistance, then brake diagnosis could be frozen to avoid mis-detection.

2. Brake light switch diagnostic

The brake light during low fuel condition is set if the brake light switch operational is stuck to false during the following conditions:

- Vehicle speed is decreasing from 45km/hr to 0km/hr.
- No automatic brake activation.
- Previous condition increment the counter. If a diagnostic is enabled, when this counter becomes higher or equal to specified value.

3. Brake safety switch diagnostic

The fault brake light during high fuel condition is set if brake light switch operational is stuck to true during specific conditions:

- Vehicle speed is higher than 20 km/hr
- Accelerator pedal is pressed (1 to 5 pedalage)
- Previous condition increment the counter.

The diagnostic is enabled when this counter becomes higher or equal to specified value.

If brake light switch operational is detected in false during more than the maximum brake light off time in the driving cycle the result of the diagnostic is considered to be okay, and the flag is enabled to set the readiness flag. As soon as the readiness flag is set, the fault is frozen until the next key-off. Brake safety switch acts as back up for the brake light switch. Its implementation is similar to brake light switch (Richard et al., 2017).

IV. RESULTS

Code implementation of these blocks and verification results are as shown below:

A. Brake switch operational

Code implementation of brake switch operational has been implemented as shown in "Fig. 4". In the below part

of the code a check is done to make sure that the brake switch of the vehicle is working perfectly.

```

// Brake Switch operation
// If brake switch is pressed then the car will stop the motor
// If the brake switch is not pressed then the car will start the motor
// If the brake switch is pressed then the car will stop the motor
// If the brake switch is not pressed then the car will start the motor
// If the brake switch is pressed then the car will stop the motor
// If the brake switch is not pressed then the car will start the motor
// If the brake switch is pressed then the car will stop the motor
// If the brake switch is not pressed then the car will start the motor

```

Fig. 4 Logic of Brake Switch Operation

B. Characteristic Time (ack condition)

Characteristic is used to measure time. The characteristic will have three conditions (enable and level based on which the output will be produced). The output produced will be in the units of time. Initially the test condition will be checked. If test is true the characteristic output will be zero. The characteristic will start giving the output in units of time when the enable condition is true. Characteristic means will be having three parameters one for the driver, second is enable condition, and the third parameter is previous value of enable condition. When the previous enable is true then only characteristic will produce the output irrespective of the current enable condition. Fig. 5 shows the code snippet of characteristic under level based condition.

```

// Characteristic time based condition
// If enable is true then the output will be zero
// If enable is false then the output will be the previous value
// If enable is true then the output will be the previous value
// If enable is false then the output will be the previous value
// If enable is true then the output will be the previous value
// If enable is false then the output will be the previous value
// If enable is true then the output will be the previous value
// If enable is false then the output will be the previous value

```

Fig. 5 Code snippet of characteristic (level based condition)

C. Verification Results

Verification results are analyzed below in "Fig. 6".

Test Case	Pass	Fail	Warning
Test Case 1	Pass	Fail	Warning
Test Case 2	Pass	Fail	Warning
Test Case 3	Pass	Fail	Warning

Fig. 6 Verification results of RTST

V. Conclusion

The competition in automotive industry has increased recently with the increase of electronics in vehicles. This has made it important that proper testing and validation is being done. The electronic system in vehicle is critical because of its passenger's safety. This emphasizes the importance of testing along with the correct feature implementation. In this project the implementation of code based on the feature requirement has been done. After implementation of the code it was moved to testing phase, here RTST and polymorphic tools were used for testing purpose. The polymorphic tool was used for static analysis of the code. It gives information regarding potential problems in the code which would have occurred during run time. Devolve by area, range overflow, underflow is some of the run time problems examples. The RTST tool is a dynamic testing tool where the code was tested by writing different test strategies and varying the inputs accordingly. The testing revealed that the code is error free and will be used for validation phase.

VI. FUTURE WORK

There is the main constraint across different organizations. A developer and tester lacks the time to implement and test any feature from the client side. On time delivery is the top priority because of the intense competition from other organizations and the dynamic market requirements. Implementation of the feature i.e. writing codes and testing can be automated. Software can be written to automate this process in such a way that minimal human involvement is needed. This will help in delivering the feature on time to the customer. It will also save the cost.

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Face Detection Based Automation of Electrical Appliances

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Abstract. This paper addresses the problem of automation of electrical appliances with the use of sensors. The proposed method uses Viola-Jones method a real time face detection algorithm for controlling the electrical appliances which are part of our day to day life. A detector is initialized by Viola-Jones based face detection algorithm automatically without the need for any manual intervention. After initializing the detector, the position where the face detected is compared. The region captured by the camera is split into four different regions. Based on the position of the detected face the electrical appliances subjected to that particular region can be controlled.

Keywords: Face detection • Viola-Jones • Split to four regions circuitry

1 Introduction

With the growth in the field of electronics, constant efforts have been made to automate various day-to-day tasks. One of these tasks is the automation of electrical appliances. This can be accomplished using GSM which requires the users to check the status of the appliances through mobile phones when they step out, or it can also be done through voice commands [1]. However this is still subjected to errors as it requires human effort and is not completely automated. Another way of controlling electrical devices is through the use of sensors [2] which provides complete automation, however the poor responsiveness of the sensors and complexity of network troubleshooting brings down the efficiency of the entire system.

Due to the drawbacks posed by the already existing methods, this paper proposes automation of electrical appliances by using face detection. The simplicity of the circuit involved, the wide coverage range and a fully automated setup ensures power efficient control of appliances. Face detection is used to compare the size and identify the position of the face in a given input image. The background information are ignored and only the facial feature from the image are detected. From the detected face we can able to predict the age of the person, gender, facial expression etc. In general term face



detection is to localize the face in the given input image. Localization helps to know the amount of faces and where the faces are detected in the image.

2 Related Work

This section addresses few techniques which are used for computer vision face detection.

Detection based on skin colour segmentation aims at analysing the visuals in different colour spaces and identifying the face with certain threshold values [14]. The different colour spaces that have been widely used are YCbCr, HIS and binary and some works have proposed the combination of different spaces to threshold the facial features. However this method can be efficient for stationary images but not for real time video streams.

The second section involves feature extraction for object detection using features from SMQT. In order to increase the computation of SVM classifier, the classifier is divided into multiple stages on which a single classifier is trained to form a weak classifier and these weak classifiers are combined to form a strong classifier in cascade which help in face detection [1].

Neural network consists of perceptron which are artificial neurons, are available in multiple layers and are connected to each other. Perceptron network, an under supervised learning and no kind of programming is required [2]. Perceptron network comprises of a threshold function or summation function. Face detection using neural network is determined by checking whether the detected region is face or not at a small window level. It is not necessary to train with non-face images and thus computational speed is also reduced [3]. Face detection using neural network comprises of two steps. A part of image of $20 * 20$ size is fed to the neural network as input in the first step. The output of the filter determines whether the detected region is face or not a face. In the second step the efficiency of the overall neural network is increased by fusing all the single neural network to find the overlapping detections and this reduces the false detections.

Maximum hyperplane or set of hyper planes in an infinite dimensional space is constructed using a supervised learning model which groups the information in two classes and separation is from two categories is through the hyperplane [4]. The area between hyper plane and closest data points is called the margin. Data points that exist on the margin boundary of the hyper plane are called as support vectors. This method can be applied for object detection then the object can be assumed from positive samples and non-object classes are from the negative samples.

The kernels can be of type polynomial kernels or radial kernels. The main difficulty lies in finding the appropriate kernel for the given problem. The selected kernel is used for the classification of problems and the results produced by them may not be good when compared to the result produced from sample set. They are used in application like detection of pedestrians. Haar wavelets are applied on samples which are positive as well as negative to extract features and they are helpful in discrimination of classes [5].

3 Proposed Method

3.1 Face Detection

The face detection was carried out through Viola Jones Algorithm. Paul viola and Michael Jones proposed the algorithm in 2001 [6, 7]. Since this many works have been done using this algorithm [8–11]. It is mainly used for detecting faces in real time rather than object detection. The algorithm is very robust and also the computation speed is less. The main goal of face detection is to distinguish face from non faces. It is the first step in face recognition. The algorithm comprises of four steps which are:

- (1) **Haar Feature Selection:** Haar-like features are used in object recognition. Computation of image intensity (i.e. calculation of RGB pixel values from every pixel made the computation to be very slower and expensive. In a human face the nose bridge region is lighter than the eye region etc. Using Haar basis function the properties of face are compared using Haar features. They are rectangular pixels of dark and white regions and the resultant of each haar feature is calculated by subtracting the black pixel count from the white pixel count (Fig. 1).

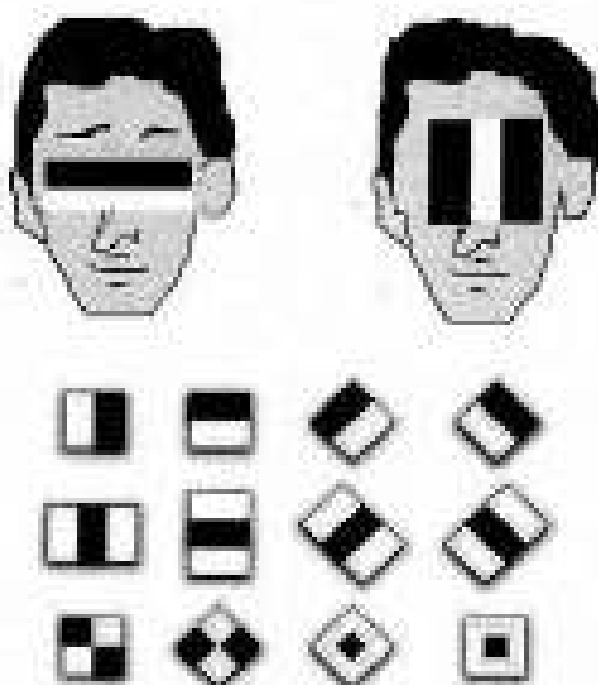


Fig. 1. Haar features

- (2) **Creating an Integral Image:** For a faster and effective way for calculating the pixel values we go for integral image. If we have to use integral image we go for the conversion of original image into gray scale image. Computation of adjacent pixel values with respect to the pixel value at (x, y) forms the integral image which helps in speeding up the computation. Integral image is the summation of values

pixels above, left and its own pixel value at (x, y) . This is done to make the Haar feature compute efficiently (Fig. 2).

0	1	1	1
1	2	2	3
1	2	1	1
1	3	1	0

 \Rightarrow

0	1	2	3
1	4	7	11
2	7	11	16
3	11	16	21

Fig. 2. Integral Image

- (3) **Adaboost Training:** In order to separate relevant and irrelevant Haar like features, we go for Adaboost training. Irrelevant features doesn't give any information about the detected image, whether it is face or not. Relevant features are the features which provide information about face. Relevant features are provided with weights to form a weak classifier. Adaboost training gives a strong classifier which is able to say whether the detected image is a face or not a face. These strong classifiers are nothing but a linear combination of all the weighted weak classifiers.

$$f(x) = \alpha_1 f_1(x) + \alpha_2 f_2(x) + \dots + \alpha_n f_n(x) \quad (1)$$

- (4) **Cascading Classifiers:** Adaboost gives information about the classifiers which will be able to say whether the detected image is face or not. Cascade classifiers are used to integrate many classifiers which are useful in neglecting background windows so that more operation are performed on regions where the face is present. Each stage in the classifier consists of distinct sets of face features which is checked step by step for the input video to detect the presence of a face. The stronger classifiers are kept in set 1, so a when the detected image is not a face no need to check for all the classifiers. From the first set of features itself we can be able to predict whether the detected image is face or not a face (Fig. 3).

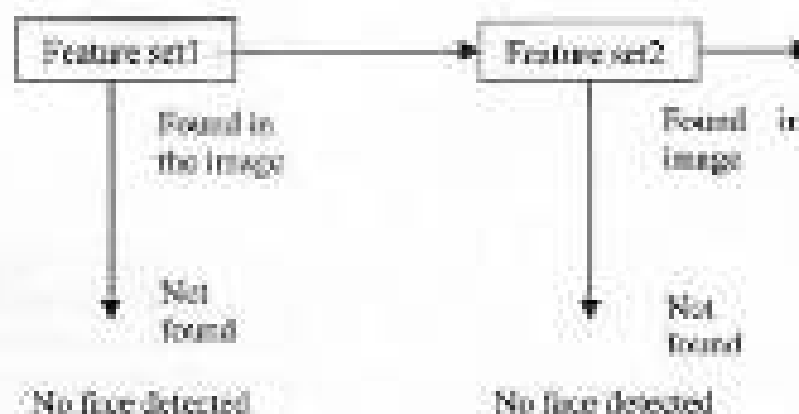


Fig. 3. Cascade classifier

3.2 Automation of Appliances

Raspberry pi is a microprocessor which is a small size CPU which runs on Raspbian OS. It supports cameras as peripherals and hence it is the best choice for this application. Up to three cameras can be connected to a raspberry pi board at a time (Fig. 4).



Fig. 4. Raspberry Pi model 3B

The automation of appliances happens by controlling them using a programmed Raspberry pi microcontroller to which a web camera is connected to record the real time video (Figs. 5, 6 and 7).

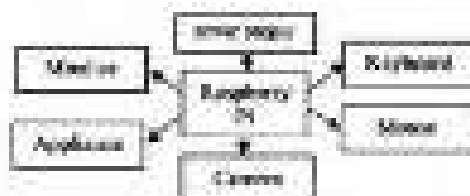


Fig. 5. Block scheme of hardware

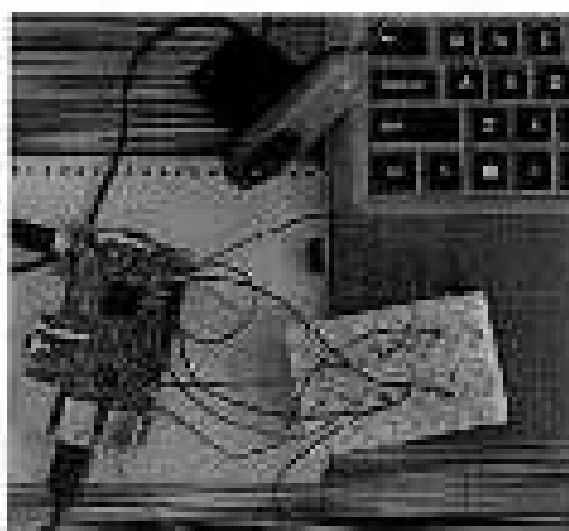


Fig. 6. Proposed system hardware

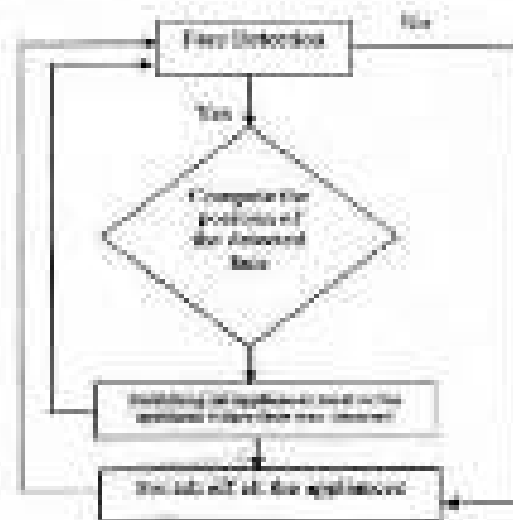


Fig. 7. Design flow of hardware

4 Results and Discussion

OpenCV is an open source computer vision library. The library is written in Python and runs under Raspbian Operating System. The proposed method with Viola-Jones detection was simulated in Python file using OpenCV Python library. The simulations were done on Raspberry Pi 3 Model B which has RAM of 1 GB. This system was tested on real time videos captured in laboratory. Video is captured at a resolution of 640×480 using a web camera. The video is split virtually into four regions. Each region has specific appliance which will be automated only if faces are detected in that region (Fig. 8).



Fig. 8. Region split of a video

Based on the region where the face got detected we can be able to say whether is very close or far to the camera. When camera placed at a certain height from the ground, if face gets detected at the top region we can say that the person is far from the camera since the scale factor of the image will be less. If the image gets detected in the bottom region we can say that the person will be closer to the camera since the scale factor will be high when face gets detected in this region (Figs. 9, 10, 11, 12 and 13).

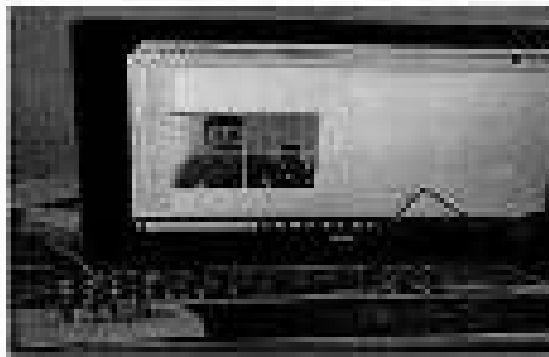


Fig. 9. Shows the face detected in top left and corresponding led is turned on and rest of the led is turned off.

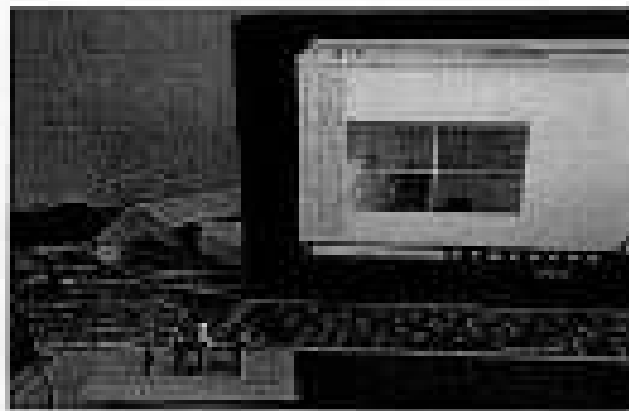


Fig. 10. Shows the face detected in bottom left and corresponding led is turned on and rest of the led is turned off.



Fig. 11. Shows the face detected in bottom right and corresponding led is turned on and rest of the led is turned off.



Fig. 12. Shows the face detected in top right and correspond ng led is turned on and rest of the led is turned off.



Fig. 13. Shows all the led turned off when there is no face detected.

The proposed method will help in efficient use of power. The system on the whole covers the drawbacks like human fatigue in conventional manual control of appliances, small coverage area of sensors. The system will have a disadvantage when there is no sufficient light to detect face. The above discussed idea was prototyped using led's. The work will be extended to controlling the device based on face recognition and also integrating the appliances to the system.

Compliance with Ethical Standards

- ✓ All authors declare that there is no conflict of interest
- ✓ No human/animals involved in this research work.
- ✓ We have used our own data.

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ANTENNA BEAMFORMING AND BEAM CONTROLLING FOR IMPROVING THE WI-FI SIGNAL

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Abstract:

Antenna design plays a prominent role when we consider the far-field patterns when compared with near field patterns. By using a PIN diode (which has a wide intrinsic region when compared with normal diode) as a switch and a capacitor as a filter we are proposing an antenna design that able to radiate far field regions. The reason behind choosing capacitor as a filter means that we are designing antenna to transfer high frequency signals. For antenna design simulations we have different software's (like FEKO, ZELAND IE 3D etc.) but the preferred software for this antenna design is Computer Simulation Technology Microwave Studio (CST MWS) as we are operating in a frequency band of 2 to 2.54GHz and the diameter of the antenna 20X20mm. Based on the diameter chosen the antenna is able to switch direction of signalling by using a switch with four different angles. The material used for antenna design FR-4 which is a Composite material composed of fibre glass cloth with an epoxy resin binder i.e. flame resistance and SMB connectors with 75Ω impedance.

Keywords: — Antenna Directivity, Strip antenna array, WI-FI.

Introduction:

If we want to increase the gain of the antenna as well as to reduce the unwanted source we must be able to control the beam of radiation pattern and this can be achieved by using the Beam Control antenna which is designed by using

PIN diode as a switch and capacitor as filter. Beam antenna must work on the principle of directional antenna which has the ability to concentrate to a single direction with high gain and to reduce the energy that gets wasted in different areas (example Yagi antenna). The material used for antenna design FR-4 which is a Composite material composed of fibre glass cloth with an epoxy resin binder i.e. flame resistance and SMB connectors with 75Ω impedance.

Antenna Design

In this method we are using four PIN diodes as switches with four different angles to allow the specific radiation pattern. Each diode need two capacitors to act as filter in order to prevent the dc current not to enter the antenna[1] [2]. If any one of the diode is supplied with energy of 5V DC then each angle is able to radiate 90 degree. This antenna design is done by using FR-4 material whose dielectric substrate is assumed to be 0.17mm and the thickness of copper wire is assumed to be 0.035mm. Here the pin diode which is acting as a switch restricts the DC current and allows the AC current to pass through the antenna. The Capacitor filter acts as a high pass circuit which allows only high frequency signals and avoids the low frequency signals [3].

Fig 1: Antenna Front View

The front view of the antenna is represented in fig 1 and the middle patch is simulate as 2.54 GHz frequency with the diameter 75mm X 75mm while the outside four patches is approximate the far-field



and diode switching and filtering is used for antenna impedance.

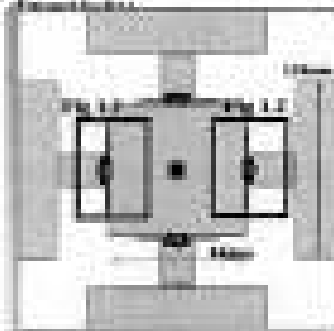


Fig. 2 Antenna Back view

The red square at the middle of the part in fig 1 indicates SMD connection with 75Ω impedance. This region shows the antenna connection from main section to ground. The mid region of the antenna is of diameter 86mm X 86mm. In the design point of view the PIN diode is connected between the ground and the small patch of the antenna as shown below

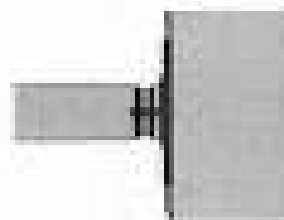


Fig. 3 Antenna Design with Switch as Short Circuit

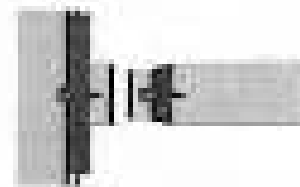
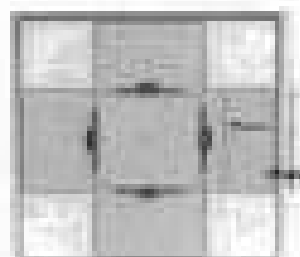


Fig. 4 Antenna Design with Switch as Open circuit

In fig 3 we can observe that the connection between the ground of the antenna and the small patch did not connect which shows that PIN diode is not connected i.e. switch is working as open circuit logic. Fig 2 and Fig 3 are the zoom versions of Fig 1.

In fig 1 we observe three capacitors are represented as three triangles with blue colour dots and the middle one is the PIN diode acts as switch whereas remaining three as filters. When we apply the DC current of 5V then PIN diode gets activated and it will drain the AC signal from antenna to other side and both the filters restrict the dc current.

From Fig 3 we can observe that if PIN diode is not connected then the signal from antenna unable to pass through the other side of antenna then it cannot generate the far field on the other side.

Result Analysis

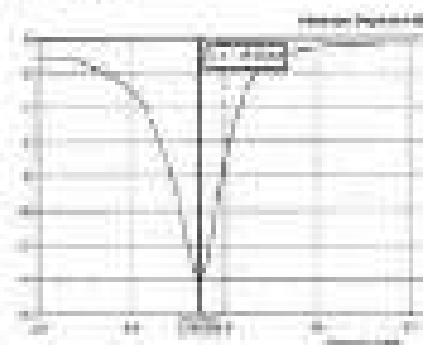


Fig. 4 S11 Parameter

From Fig 4 we observe a frequency drop at 2.4GHz and S11 magnitude at -14.00 and after simulation with and without diode the frequency

and S11 magnitude remained same. The desire frequency range is dropped to below -10 while the frequency 2.47 GHz is under ISM band and in the range of Wi-Fi band range i.e. 2.4 GHz to 2.5GHz [4].

Conclusion:

The antenna is able to radiate far field signal by using diode and capacitor and by using CST MWS software we are able to simulate the antenna to get desired frequency, far field and gain. As diode rotates it represents far-field with different angles.

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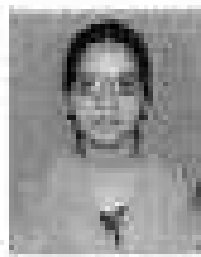
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Fig 1: Antenna Front View

The front view of the antenna is represented in Fig 1and the middle patch is simulate as 2.54 GHz frequency with the diameter 70mm X 70mm while the outside four patches is to generate the far-field

and diode switching and filtering is used for antenna feedline.

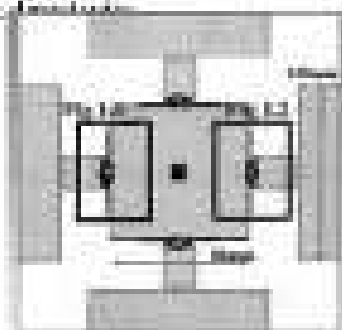


Fig. 2 Antenna Back view

The red square at the middle of the part in fig 1 indicates SMD connection with 75Ω impedance. This region shows the antenna connection from main section to ground. The total region of the antenna is of diameter 35mm X 35mm. In the design point of view the PIN diode is connected between the ground and the small patch of the antenna as shown below

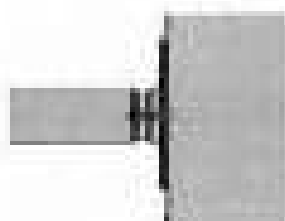


Fig. 3 Antenna Design with Switch as Short Circuit

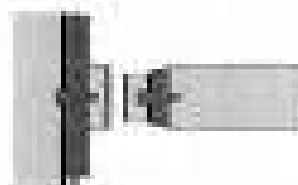
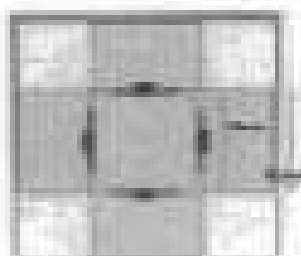


Fig. 4 Antenna Design with Switch as Open circuit

In fig 3 we can observe that the connection between the ground of the antenna and the small patch did not connect which shows that PIN diode is not connected i.e., switch is working as open circuit logic. Fig 2 and Fig 3 are the same versions of Fig. 1,

In fig 1 we observe three capacitors are represented as three triangles with blue colour dots and the middle one is the PIN diode acts as switch whereas remaining three as filters. When we apply the DC current of 5V then PIN diode gets activated and it will drain the AC signal from antenna to other side and both the filters restrict the dc current.

From Fig 3 we can observe that if PIN diode is not connected then the signal from antenna unable to pass through the other side of antenna then it cannot generate the far field on the other side.

Result Analysis:

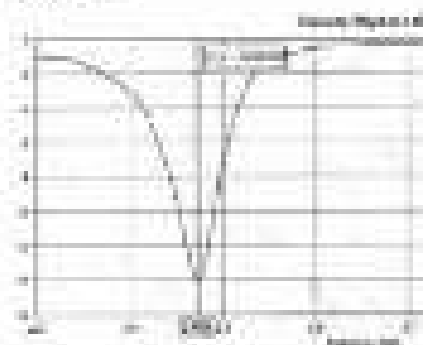


Fig. 5 S11 Parameter

From Fig 4 we observe a frequency drop at 24.5GHz and S11 magnitude is -14.50 and after simulation with and without diode the frequency

and S11 magnitude remained same. The desire frequency range is dropped to below -10 while the frequency 2.47 GHz is under ISM band and in the range of Wi-Fi band range i.e. 2.4 GHz to 2.5GHz [4].

Conclusion:

The antenna is able to radiate far field signal by using diode and capacitor and by using CST MWS software we are able to simulate the antenna to get desired frequency, far field and gain. As diode rotates it represents far-field with different angles.

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STUDIES ON MECHANICAL PROPERTIES OF SISAL AND JUTE FIBER HYBRID SANDWICH COMPOSITE

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Abstract

The present study deals with the fabrication of hybrid composites and mechanical properties are tensile, flexural, compression strength and impact of sisal fiber and jute fiber combination. Epoxy resin (EY556 grade) and hardener (H21931) is taken as the matrix and binder for the sisal and jute fiber combination. The five different kinds of samples of jute/epoxy and sisal fiber/epoxy composite were made with the fiber-matrix weight ratio of 35 %. After fabrication, the test specimens were prepared as per to the ASTM standards. The result shows the co-sisal tensile strength of jute/epoxy and sisal fiber/epoxy composite the result was increased 32% than sisal epoxy. The tensile, flexural, compression strength and impact of jute fiber/epoxy composites are higher than those of sisal fiber/epoxy composites at the same fiber loading. Additionally, the SEM images are studied in the different combinations.

Keywords: Hybrid composite, jute/epoxy, hardener

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1. INTRODUCTION

The composite material can be defined as the material which is composed of two or more distinct material on a macro scale with different properties to form a new material with a property that is entirely different from the individual constituents. The cost of natural fiber Composites is low and light in weight but strength was lower than synthetic fibers [1]. The primary phase of a composite materials called a matrix having a continuous character. In other words, the matrix is a material which acts as a binder and holds the fibers in the desired position there by transferring the external load to reinforcement. These matrices are considered to be less hard and more ductile. The composite material consists of a matrix along with a fiber with some filler material. The reinforced material can be either synthetic or natural fibers. In the demand of increasing environmental security, several natural fibers reinforced polymer composites (NFRPs) are brought into the competitive market. NFRPs provide a wide range of advantages over synthetic fiber based composites. Due to the superior properties of Natural fiber composites it become so popular and attractive. [2]. These advantages include high strength to weight ratio, high strength at elevated temperatures, high creep resistance and high toughness [3]. These advantage can also be in the form of high durability and design flexibility. M.Boşgelmez et al investigated the mechanical properties of raw jute and sisal fiber reinforced epoxy composites treated with sodium hydroxide and the results found better for treated composites [4].

Hybrid composites are more advanced composites as compared to conventional FRP composites. Hybrids can have more than one reinforcing phase and a single matrix phase or single reinforcing phase with multiple matrix phases or multiple reinforcing and multiple matrix phases. They have better flexibility as compared to other fiber reinforced composites. It usually contains a high modulus fiber with low modulus fiber. The high modulus fiber provides the stiffness and load bearing qualities, whereas the low modulus fiber makes the composite more damage tolerant and keeps the material cost low. M.K.Gupta et al studied about the dynamic mechanical properties and water absorption characteristics of hybrid sisal and jute fiber reinforced epoxy composite. from their work, they concluded that natural fibers are a better replacement of synthetic fibers due to its acceptable mechanical properties [5]. Subramaniam et al carried out an experimental work to study the wear and frictional properties of the hybrid composite with different operating parameters; their study reveals that the alkali treatment of fibers is used to increase the percentage reduction of wear rate and friction coefficient [6]. Herika Muruges et al [7] studied about the jute fiber reinforced with both polyester and epoxy resin; the results show that the jute reinforced epoxy composite exhibits better mechanical properties than jute polyester composites. Ahmed et al [8] investigated the properties of sisal reinforced epoxy composite using varying the fiber lengths (5,10,15,20cm) with constant 30% of sisal fiber. The results of their work reveal that 15cm and 20cm sisal fiber reinforced composite shows significant improvement in the flexural and impact property. Lakshmi et al [9] studied through a case study design about the possibilities for the replacement of glass fibers by natural jute fibers to produce a structural beam of an off-road vehicle (buggy motorized). Choudhary et al [10] discussed the optimum mixing of fiber and resin of sisal fibers. It made hybrid composites in dry and wet conditions. The mechanical properties of a hybrid composite can be varied by changing volume ratio and mixing sequence of different phase.

1.2 Problem Identification

Based on the literature presented in this chapter, it is found that some researchers are carried out on natural fiber reinforced composites; they have found the Co-sisal tensile properties and other mechanical properties of these composites. Their main objective is identification of Tensile axial properties of fiber sisal fiber reinforced composites. So it is decided to carry out research the topic of Co sisal and Tran axial Tensile Properties of composite material and analysis the mechanical properties of flexural behavior, impact behavior, compressive behavior.

2. Materials and Methods

Since the present study deals with the manufacturing of hybrid composites, the material for reinforcement is chosen as sisal fiber and jute fiber due to its suitable mechanical properties and epoxy resin (E-YS86 grade) and hardener (BPO/514) taken as the matrix and binder for the composite. This process carried out with the help of compressive moulding. Work piece prepared in the dimension 300x300x5 mm was used for carrying the composite sheet. The sample geometry is shown in Figure 1. There are five different kinds of samples were made with the fibre matrix weight ratio of 15 %. The sample is tabulated below the Table 1.

Table 1 Composite Sample Details

Sample Number	Wt % of pure fiber	Wt % of Glass fiber	Wt % of Epoxy and Hardener
Sample 1	3	55	65
Sample 2	10	55	65
Sample 3	20	15	65
Sample 4	25	10	65
Sample 5	55	4	65

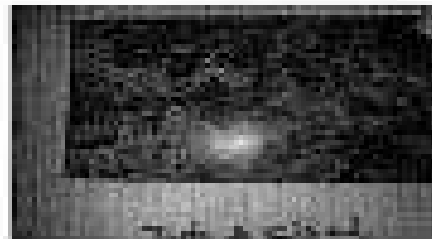


Figure 1 Compression molding process

3. Mechanical Testing

The test specimens were prepared and undergone different mechanical testing namely Tensile test, flexural test and impact test as per the ASTM standards. The specimen size and shape for corresponding test are as follows.

3.1 Tensile Test

A composite specimen is subjected to Tensile testing to measure the force required to break the test specimen and the extent to which the specimen stretches or elongates to that breaking point. The physical properties of materials at a temperature that simulate intended end user environment. The most common specimen for ASTM D3199 has a constant rectangular cross section, 25mm wide and 250mm long is used to study.

3.2 Flexural Test

Flexural strength is carried out to find the ability of the material to resist the deformation under the load. This is a three point bend test is used to precisely define by inter laminar shear. This test is conducted as per ASTM D790 standard by UTM. The specimen dimension is 13x125x3 mm. The flexural strength is reported as modulus of rupture (MR) in psi(MPa). The flexural strength is calculated using the Equation 1.

$$\sigma_f = (3P_{max}L) / 2bh^2 \text{-----(1)}$$

where,

- P_{max} = maximum load at failure
- L = specimen width, b = specimen thickness
- L = specimen length between the two support points

3.3 Impact Test

The impact is a single point test that measures the resistance of a material to impact from a swinging pendulum. The impact is defined as the kinetic energy needed to initiate fracture and continue the fracture until the specimen is broken. The ASTM D256 (1Jx50J) and size is taken for the test.

3.4 Compression Test

The main aim of a compression test is to determine the ability of the material to withstand the compressive force under the compressive load during a period of time to measure the parameters such as strain, stress, and deformation. The specimens prepared by ASTM D 3410.

3.5 Scanning Electron Microscopy(SEM)

SEM is a type of electron microscope that produces images of a sample by scanning it with a focused beam of electrons. The morphological characteristics of the composite surface is observed as scanning electron microscope of Model JEOLCAN VUGA 5. The composite samples are adequately cleaned, air dried and air coated with 100Å² thick platinum in JEOL sputter ion coater and observed SEM at 10 KV. The fracture surface morphology of the composite specimens is observed with SEM.

4 Results and Discussion

The mechanical properties of the Jute / Sisal fiber hybrid epoxy composite prepared for this present investigation is presented in this discussion. The results of various characterization tests are reported here. These include evaluation of tensile strength, flexural strength, impact strength and compression values are tabulated in Table 3 and the properties are discussed.

4.1 Tensile Test

Random directional tensile specimens were cut in the longitudinal and transverse direction for tensile test according to ASTM D 3039 type IV sample the tensile test specimens were carried out, using a tensile testing machine with crosshead speed of 2mm/min and a gauge length of 150 mm. Three specimens were tested for each set of samples and mean values were tabulated in Table 3.

4.1.1 Coaxial Tensile Test.

The coaxial tensile strength of hybrid composite at 35,50wt% Jute and Sisal fiber were 22.53 MPa². As can be seen Figure 2, increasing weight fraction of jute fiber resulted in the positive effect on tensile properties of hybrid composites. The maximum tensile stress value of the composite was 22.53 MPa². This is probably due to the loss of the integrity of the matrix and insufficient wetting between fiber and matrix as a result of high fiber loading.

Table 3 Mechanical Properties values

S.No	Coaxial Tensile strength	Trans-axial Tensile strength	Flexural Strength	Impact Strength
	MPa ²	MPa ²	MPa ²	J
30wt%	15.38	17.89	32.17	0.83
40wt%	14.21	14.84	40.50	0.4
50wt%	13.61	8.72	38.65	0.53
60wt%	15.78	9.37	34.48	0.55
Sisal	22.53	12.41	36	0.4

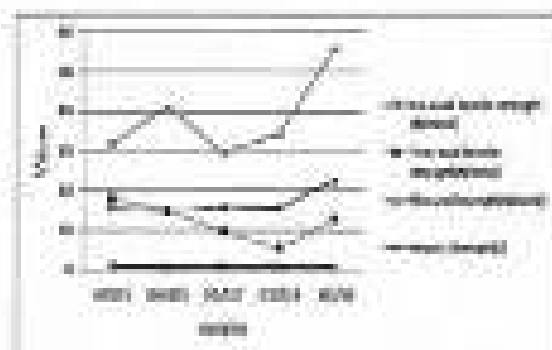


Figure 2 Mechanical properties of Jute / Sisal Fiber hybrid epoxy composite

4.1.2 Translational Tensile Strength

The effect of Jute / Steel fiber hybrid epoxy composite is presented in the tensile strength of hybrid composite at 0.15 wt% Jute and Steel fiber were 17.99 N/mm^2 . As can be seen figure 2, increasing weight fraction of jute fiber resulted in negative effect on tensile properties of hybrid composite. The maximum tensile stress value of the composite was 17.99 N/mm^2 . This is probably due to the loss of the integrity of the matrix and insufficient wetting between fiber and matrix as a result of high fiber loading.

The tensile strength of a composite material is mainly dependent on the strength and modulus of fibers, the strength and chemical stability of the matrix and fibers in transferring stress across the interface. When fiber reinforced composites are subjected to load, the fibers act as carrier of load and stress is transferred from the matrix along the fibers leading to effective and uniform stress distribution, which results in a composite having excellent mechanical properties.

4.2 Flexural Test

The flexural test was performed by the three-point bending method according to ASTM D 790, and a crosshead speed of 2 mm/min. 3 specimens were tested, and the averages were calculated. The maximum load was applied in the middle of the specimen, when the specimen was freely supported by a beam. The flexural modulus are evaluated from the slope of the initial portion of the load-deflection curve. The figure 2 shows that incorporation of Jute and Steel fiber at a 35% relative weight fraction leads to hybrid composite having a superior flexural strength. The maximum flexural strength of the composite was 36.3 N/mm^2 . Flexural strength is nothing but a strength combining the tensile and compressive strengths and modulus with the interfacial shear strength between the fiber and matrix. It was observed from figure 3 that the flexural strength and modulus of Jute / Steel fiber epoxy composite increased with increasing jute fiber. If adding steel fiber will reduce flexural strength.

4.3 Impact Test

The impact strength of Jute / Steel fiber hybrid epoxy composite for impact energy increasing with increasing weight fraction of steel fiber upto 15wt%. The relationship between fiber loading and impact strength is shown in figure 2. The load which applied on fibers is transferred by shear to the fibers, may exceed the fiber/matrix interfacial bond, and delamination may occur. The frictional force along the interface may transfer the stress to the deformed fiber. If the fiber stress level exceeds the fiber strength, fibers may break. The obtained results reveals that the fiber is capable of absorbing energy due to its strong interfacial bonding between the fiber and matrix up to 5% wt% steel fiber loading. However, at higher loadings, the lower fiber interaction decreases the effective stress transfer between the fiber and matrix. This contributes to a decrease in impact properties at higher fiber loadings. The high impact strength steel fibers found 81.1

4.4 Compression Test

Compression testing of a specimen prepared according to ASTM D 3410. The effect of Jute/Steel fiber hybrid epoxy composite is presented in figure 2. 9wt% Jute and steel fiber was 31.09 N/mm^2 . The fibers act as carrier of load and stress is transferred from the matrix along the effective and uniform stress distribution. When fiber reinforced composites are subjected to load. Due to this reason, it leads a good composite having excellent mechanical properties. At a low level of fiber loading, the matrix is not restrained by enough fibers causing the bond between matrix and fiber to break, leaving the matrix distorted by non-reinforcing deformed fiber.

4.5 SEM Images

Figure 3 shows the different samples and images. The variation of the Mechanical properties of prepared composite specimens through phase information shown by means of the topography properties. A figure clearly shows the poor bonding capacity of the composite so that mechanical properties are reduced. Sample 1 shows the jute/steel fiber bundles, which indicate that jute fibers were not adequately wet by epoxy polymer resin and this led to low stress SEM images indicate that presence of jute fibers which indicate that medium interfacial bonding between epoxy and jute and gives excellent mechanical properties compared to composite.

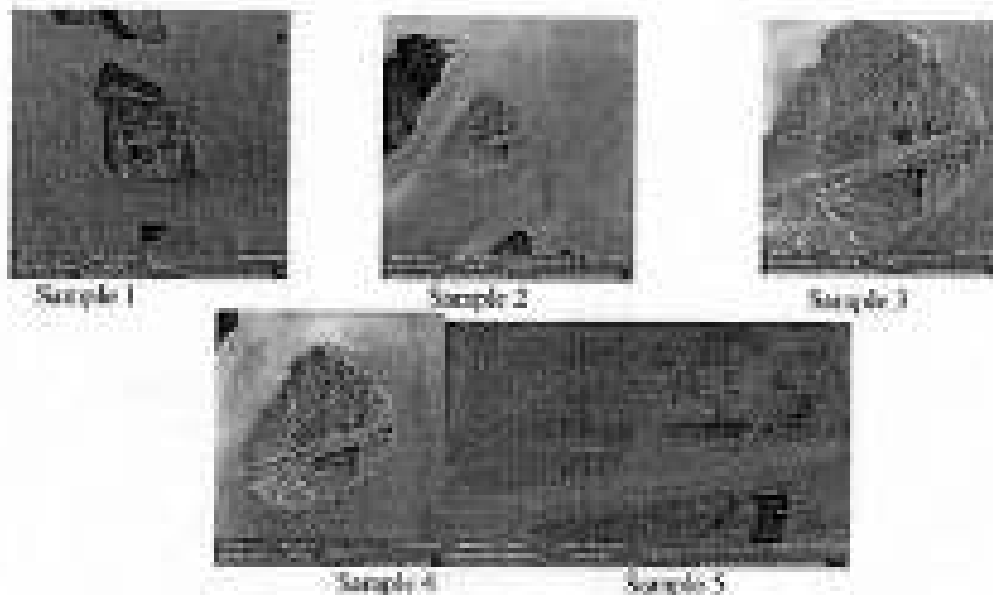


Figure 3 Different samples images.

5. Conclusions

The Mechanical properties of Jute and Steel Fiber reinforced hybrid composites were studied, and the following conclusions are drawn.

1. The jute, steel fiber was successfully used to fabricate natural hybrid composites with 10% fiber and 90 % resin, these fibers are biodegradable and highly crystalline with well-aligned structure. The tensile strength of jute/epoxy and steel fiber/epoxy composite having compared to steel epoxy the result was increased 32%. Hybrid composite the tensile strength is decreased.
2. Compare to Trans axial loading and osinal loading is a better result.
3. The flexural strength of jute/epoxy composite improved by increasing fiber loading up to 15wt.%.
4. Addition of steel fiber decreases the tensile, flexural, compression and increases the Impact properties of composite.
5. The tensile, flexural, and compression strength of jute fiber/epoxy composite are higher than those of steel fiber/epoxy composite at the same fiber loading.
6. Impact properties of steel/epoxy composite are better than the jute fiber epoxy composite.
7. The SEM analysis of the different hybrid composites with jute composite which gives excellent mechanical properties compared to jute and steel hybrid epoxy composite.

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Nano Composite Properties and its Applications: A Review

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Abstract

The field of nanocomposite is one of the most popular areas for current research and development in basically all technical disciplines. Nanocomposites, a high performance material exhibit unusual property combinations and unique design possibilities. With an estimated annual growth rate of about 25-30% and fastest demand to be in engineering plastics and elastomers, their potential is so striking that they are useful in several areas ranging from packaging to biomedical applications. In this unified overview the three types of matrix nanocomposites are presented underlining the need for these materials, their processing methods and some recent results on structure, properties and potential applications, perspectives including need for such materials in future space mission and other interesting applications together with market and safety aspects. Possible uses of natural materials such as clay based minerals, chrysotile and E-glass/fiberglass fibers are highlighted. Being environmentally friendly, applications of nanocomposites offer new technology and business opportunities for several sectors of the aerospace, automotive, electronics and biotechnology industries.

Keywords: nanocomposites, layered compounds, polymers, metals, ceramics

1. INTRODUCTION

Nanocomposites are composites in which at least one of the phases shows dimensions in the nanometric range ($1 \text{ nm} = 10^{-9} \text{ m}$) [1]. Nanocomposite materials

have emerged as suitable alternatives to overcome limitations of microcomposites and metallics. They are reported to be the materials of 21st century in the view of possessing design uniqueness and property combinations that are not found in conventional composites. The general understanding of these properties is yet to be reached [2], even though the first inference on them was reported as early as 1992 [3]. It has been reported that changes in particle properties can be observed when the particle size is less than a particular level, called the critical size (Table 1) [3]. Additionally, as dimensions reach the nanoscale level, interactions at phase interfaces become largely improved, and this is important to enhance materials properties. In this context, the surface area/volume ratio of reinforcement materials employed in the preparation of nanocomposites is crucial to the understanding of their structure-property relationships. Further, discovery of carbon nanotubes (CNTs) in 1991 [4] and their subsequent use to fabricate composites exhibiting some of the unique CNT related mechanical, thermal and electrical properties [5-9] added a new and interesting dimension to this area. The possibility of spinning CNTs into composite products and textiles [10] made further grounds for the processing and applications of CNT-containing nanocomposites. Nowadays, nanocomposites offer new technology and business opportunities for all sectors of industry, in addition to being environmentally friendly [11].

As in the case of microcomposites, nanocomposite materials can be classified, according to their matrix material, in three different categories as shown in Table 2.



Ceramic Matrix Nanocomposites (CMNC),

Metal Matrix Nanocomposites (MMNC) and

Polymer Matrix Nanocomposites (PMNC).

Finally, to the best of our knowledge, and in view of the very limited work on metal-based nanocomposites including the ones with CNT reinforcements, no review is available to-date on this system. Considering these facts and also the absence of a more general review comprising the three different kinds of nanocomposites (metal-, ceramic- and polymer-based), this paper gives an overview of them, including those with incorporation of CNTs. However, while doing so only a few relevant publications [2,4,7] are considered here. The main features, current status and recent developments in the area are provided, focussing on the preparation methods, properties and applications of these systems to avoid repetition. Also, the potential uses of nanocomposites and the opportunities they provide, along with perspectives for the future and market and safety aspects are also presented. Nanocomposite coating is not covered, in order to keep the focus of the review.

Table 1. Feature sizes for significant changes in properties reported in nano-composite systems [reference 5 – the Japan Society of Powder and Powder Metallurgy].

Properties	Feature size (nm) at which change might be expected
Catalytic activity	<5
Making hard magnetic materials and	<20
Producing refractive index changes	<50
Producing super-paramagnetic and others	<100
electromagnetic phenomena	
Producing strength hardening toughening	<100
Modifying kinetics and plasticity	<100

Table 2. Different types of nanocomposites.

Class	Examples
Metal	Fe-O/Ni ₂ O ₃ , Ni/Al ₂ O ₃ , Cu/Ti, Ti/MgO, Al/CNT, Mg/CNT
	Al ₂ O ₃ /SiC, SiC/Si, Al ₂ O ₃ /TiO ₂ , Al ₂ O ₃ /SiC, Al ₂ O ₃ /CNT
Ceramic	Thermoplastics/thermoset polycondensed sil
	silica, polymer/TiO ₂ , polymer/CNT, polymer/ layered double hydroxides.

On the other hand, even after a decade of research, CNTs have not fully realized their potentials as microscopic reinforcements in polymer matrices. Thus great challenges and opportunities are still expected for the system. These are listed on the following:

- a) CNTs with small number of defects per unit length possess 500 times more surface area per gram (in the basis of equivalent volume fraction) of a typical carbon fiber, high aspect ratio (~1000), very high tensile properties and electrical and thermal conductivities.
- b) Research on CNT-related areas has been more active, with publications doubling within six months. Even the patenting activity in this area has been impressive, with about 3,300 applications filed from 2001 to June 2006 as per the literature survey.
- c) Because of their hollow nature, CNTs can be opened and filled with a variety of materials including biological generating technological opportunities. Added to this, the challenges in obtaining homogeneous dispersions and strong interfacial interactions, which can be better done by surface grafting functionalizations, make the use of CNTs in composites more intriguing.
- d) Various applications of CNTs in composites have been reported extensively.
- e) The possibility of spinning polymers to obtain textiles [10] certainly constitutes a great promise for their extended use in a variety of applications, particularly in the electronics and thermal management sectors.

f. Nanocomposites with biodegradable polymers have a high potential for the design of environmentally friendly 'green materials' for future applications.

On the whole, opportunities and records appear to be great with nanocomposites and hence there is a tremendous worldwide interest in these materials.

II. PROCESSING OF NANOCOMPOSITES

A. Raw Materials

As with microcomposites, CMNC matrix materials include AlO_3 , SiC, SiN, etc., while metal matrices employed in MMNC are mainly Al, Mg, Pb, Sn, W and Fe, and a whole range of polymers, e.g. vinyl polymers, condensation polymers, polyolefins, specialty polymers (including a variety of biodegradable molecules) are used in PMNC. In general, it is the reinforcement that is in the nanorange size in these materials. Both synthetic and natural crystalline reinforcements have been used, such as Fe and other metal powders, clays, silica, TiO₂ and other metal oxides, although clays and layered silicates are the most common. This is so due to their availability with very low particle sizes and well-known intercalation chemistry, in addition to generating improved properties even when they are used at very low concentrations. Most of these reinforcements are prepared by known techniques: chemical, mechanical (e.g. ball milling), vapour deposition, etc.; details of these may be found in many of the references given in the following sections.

Similarly, CNTs are prepared mostly by chemical vapour deposition methods and details are available elsewhere. CNTs consist of graphene cylinders and are available in two varieties, as single walled (SWCNT) and multi-walled (MWCNT), with about 70% yield in the case of SWCNT. While SWCNTs are single graphene cylinders, MWCNTs consist of two or more concentric cylindrical sheets of graphene around a centre, as low cost. Both types exhibit physical characteristics of solids, either metallic or semiconducting nature, with nanocrystallinity and very high aspect ratios.

In the case of CNTs, use of surfactants, oxidation or chemical functionalization of surfaces are some of

the techniques employed. Chemical methods may be more effective, particularly for polymer and ceramic matrices. Physical blending and *in situ* polymerization are used for improving dispersion in the case of CNT-reinforced polymer composites, while alignment of CNTs could be achieved by techniques such as *ex-situ* techniques (filtration, template and plasma-enhanced chemical vapour deposition, laser field-inducement, etc.) [5].

B. Processing Methods

Ceramic Matrix Nanocomposites (CMNC)

Many methods have been described for the preparation of ceramic matrix nanocomposites. The most common methodologies, as used for microcomposites, are Conventional powder method, Polymer precursor route, Spray pyrolysis, Vapour techniques (CVD and PVD) and Chemical methods, which include the sol-gel process, colloidal and precipitation approaches and the template synthesis.

A large variety of parameters affecting the sol-gel process, such as type of solvent, timing, pH, precursor, water/metal ratio, etc., allow a versatile control of structural and chemical properties of the final oxide materials. Regarding the processing of carbon nanotubes (CNT)-reinforced ceramic nanocomposites, many approaches have been described.

Metal Matrix Nanocomposites (MMNC)

The most common techniques for the processing of metal matrix nanocomposites are Spray pyrolysis, Liquid metal infiltration, Rapid solidification, Vapour techniques (PVD, CVD), Electrodeposition and Chemical methods, which include colloidal and sol-gel processes. Only two reports are found, for example, on Fe-based nanocomposites prepared by solidification techniques. The first one, by Benzigar, is called 'de-ified nanocomposite steel'. This was obtained by quenching the metallic glass obtained from a Fe-based alloy, followed by de-ifying the glass precursor through heat treatment above its crystallisation temperature. This resulted in a material showing a crystalline multi-phase microstructure. The formation of nanophases was explained by the high nucleation

frequency within the limited time for growth of grains before impingement. In order to explain the very high hardness of these Fe-based nanocomposites, Benaygan and Tang studied novel nanostructures obtained in bulk Fe alloys by designing alloy compositions with different amounts of B and C to get maximum stability. Difficulties have been encountered in preparing composites with very fine particles due to their induced agglomeration and non-homogeneous distribution. Use of ultrasound helped to improve the wettability between the matrix and the particles.

Polymer Matrix Nanocomposites (PMNC)

Many methods have been described for the preparation of polymer nanocomposites, including layered materials and those containing CNTs. The most important ones are i) lateralization of the polymer or pre-polymer from solution; ii) in-situ intercalative polymerization; iii) Melt intercalation (Figure 1); iv) Direct mixture of polymer and particles; v) Template synthesis; vi) In-situ polymerization; and vii) Sol-gel process. Publications dealing with various methods for the incorporation of nanodispersed into conducting polymers are also available; the most pertinent one is probably the incorporation of inorganic building blocks in organic polymers.

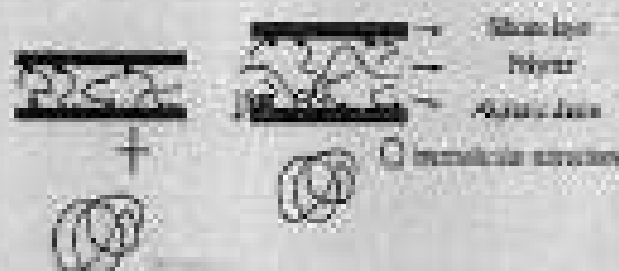


Figure 1. Melt intercalation synthesis of polymer/clay nanocomposites (The American Chemical Society, USA).

Similarly, different processing techniques, mostly chemical and electrochemical methods, have been employed for the preparation of conducting polymer nanocomposites.

When the polymer is unable to intercalate between the silicate sheets, a phase separated composite (Figure

2) is obtained, whose properties stay in the same range as that of traditional microcomposites. On the other hand, in intercalated nanocomposites, the insertion of a polymer matrix into the layered silicate structure occurs in a crystallographically regular fashion, regardless of the clay to polymer ratio. A well ordered multilayer morphology built up with alternating polymer and inorganic layers is generated. Normally, only a few molecular layers of polymer can be intercalated in these materials.

3. PROPERTIES

It can be seen from literature that there was significant improvement in the strength of the nanocomposites compared with its macro counterpart. The fracture strength, as an example, is noticeably higher because of the higher interfacial interaction between the particles in nanocomposites. Besides, AlO₃/5 to 15% SiC systems exhibited superficial grooves of plastic deformation compared to the intergranular fracture observed in monolithic materials. There was an interdependent wear transition for these composites over a load of 20-100 N, but pre-transition wear rates of $1-2 \times 10^{-4}$ mm³/m were observed for both the monolithic and composite materials. The following Tables 3,4 and 5 shows that the properties and examples of nanocomposites.

Table 3. Examples of ceramic matrix nanocomposites and their properties.

Matrix/Reinforcement	Properties
SiN ₂ /SiC	Improvement of strength and toughness
MgAl ₂ O ₄	
B ₂ O ₃ /TiB ₂	-
Al ₂ O ₃ /SiC	
MgO/SiC	-
MgAl ₂ O ₄ /SiC	
Al ₂ O ₃ /SiC	-
Al ₂ O ₃ /SiC, Al ₂ O ₃ /SiC, Al ₂ O ₃ /SiC	
	Improved thermal expansion

4. APPLICATIONS OF NANOCOMPOSITES

From the foregoing, it becomes evident that nanocomposites may provide many benefits such as enhanced properties, reduction of solid wastes [16]

thinner thickness films and lower reinforcement usage) and improved manufacturing capability, particularly for packaging applications. Tables 21 to 23 present potential applications of ceramic-, metal- and polymer-based nanocomposites, respectively. As it can be observed, the promising applications of nanocomposite systems are numerous, comprising both the generation of new materials and the performance enhancement of known devices such as fuel cells, sensors and coatings. Although the use of nanocomposites in industry is not yet large, their massive switching from research to industry has already started and is expected to be extensive in the next few years. Table 5 shows that the applications of nanocomposites.

Table 4. Properties of Al₂O₃/SiC nano- and microcomposites^{28, 29}.

Properties/Material	Al ₂ O ₃ /SiCp	Al ₂ O ₃ /SiCp microcomposit
	0.97980	0
Vickers Hardness [KPa]	-	22
Young's Modulus [GPa]	-	38
Fracture Strength [MPa]	166.20	509.60
Fracture Toughness [MPa ^{1/2}]	2.463	4.653

Table 5. Examples of metal nanocomposites and their properties.

Matrix/Reinforcement	Properties
Ag/Au	Improved catalytic activity
Ni/PZ and Ni/SSZ	Improved fracture and strength
Cu/Ni	Improved microhardness
Al/AlN	Higher compression resistance and low thermal expansion
Al/SiC	Improved fracture and electric moduli
CNT/Al and CNT/SiC	Improvements in E _f thermal expansion properties
Al ₂ O ₃ /Ti ₃ C ₂ TC	Device improvement in hardness
Cu ₂ Ni ₃ O ₄ Cu ₂ Ti ₃ O ₄	Improved microhardness Improved electrical conductivity

Table 6. Potential applications of metal nanocomposite systems.

System/Matrix	Applications
Fe/Al ₂ O ₃	Catalysis, magnetic devices
Ni/Al ₂ O ₃	Wear resistant coatings and chemically inert coatings
Fe/TiO ₂	Photo-decomposition applications
Al/SiC	Aerospace, metal and microelectronic structures
Cu/Al ₂ O ₃ Al/Al ₂ O ₃	Electronic packaging Microelectronic industry
Ni/TiN, Ni/CrN	High speed machinery, coating, optical and
Cu/SSZ	Magnetic storage systems
Fe/Al ₂ O ₃	Structural materials Catalysis Microelectronic industry
Al/Al ₂ O ₃	Microelectronics, optical devices, light emitting devices

CONCLUSIONS

In conclusion, new technologies require materials showing novel properties and/or improved performance compared to conventionally processed components. In this context, nanocomposites are suitable materials to meet the emerging demands arising from scientific and technologic advances. Processing methods for different types of nano-composites (CMNC, MMNC and PMNC) are available, but some of these pose challenges that giving opportunities for researchers to overcome the problem being encountered with nanosize materials. They offer improved performance over metallic and microcomposite counterparts and are consequently suitable candidates to overcome the limitations of many currently existing materials and devices. A number of applications already exists, while many potentials are possible for these materials, which open new vistas for the future. In view of their unique properties such as very high mechanical properties even at low loading of reinforcements, gas barrier and flame-retardant properties, many potential applications and hence the market for these materials have been projected in various sectors. Thus all the three types of nanocomposites provide opportunities and rewards creating new world wide interest in fabricating new materials.

Table 3. Potential applications of polymer nanocomposite systems.

Nanocomposites	Applications
Polycaprolactone/SiO ₂	Bone-biomimetic for skeletal tissue repair.
Polyurethane/SiO ₂	Microelectronics.
PMMA/SiO ₂	Dental applications, optical devices.
Polyethylene glycol/SiO ₂	Analysis support, stationary phase for chromatography.
Poly(p-phenylene styrene)/SiO ₂	Non-linear optical material for optical waveguides.
Polyamide (amide) / TiO ₂	Composite membranes for gas separation applications.
poly(1,4-ethylene- hexylenediene)/V ₂ O ₅	Electrode materials for rechargeable lithium batteries.
Polycarbonate/SiO ₂	Aluminum nanolaminate coating.
Shape memory polymer/SiC	Medical devices for pinning or releasing therapeutics within blood vessels.
Nylon-6/LS	Automotive timing belt - TOYOTA.
PEI/LS	Airplane interiors, fuel tanks, components in electrical and electronic parts, boxes and trays.
PLA/LS	Lubricant nanolaminate development.
PEI/clay	Food packaging applications. Specific examples include packaging for processed meats, cheese, on-factors, cereals and fruit as well as bag foods, fluid juice and dairy products, beer and carbonated drinks, etc.
Thermoplastic olefin/clay	Beverage container applications.
Polyimide/clay	Automotive suspension - GM Saturn and Acura Vaux.
Epoxy/MMT	Materials for electronics.
SPBBA/Laponex	Direct methanol fuel cells.

Table 3. Advantages and limitations of processing methods for metal-based nanocomposites.

Process	Advantages	Limitations
Spray Pyrolysis	Effective preparation of ultra fine spherical and homogeneous particles in multicomponent systems. reproducible size and quality.	High cost associated with processing large quantities of uniform, spherical particles.
Liquid Infiltration	Starts with wet stage between matrix and nanofiller. matrix, resulting into a different surface area steps of different thickness and volume of void reduction. used in industries both for wide and industrial scale production.	Use of high temperature, degradation of nanofiller. risks, formation of unwanted product during processing.
Rapid Solidification	Simple, effective Process (RSP)	Only metal-based nanocomposites, inherent agglomeration and non-homogeneous distribution of filler particles.
RSP with ultrasonic	Good distribution without agglomeration of nanofiller particles.	
High Energy Ball Milling	Homogeneous mixing and uniform distribution.	
CO ₂ PCVD	Capability to produce highly dense and uniform films, adherent at high deposition rates, good reproducibility.	Optimization of many parameters to arrive uniform CO ₂ nanocomposites.
Chemical Processes	Simple, low viscosity, low pressure, versatile, high chemical homogeneity, simple and efficient method, high purity products.	Void-bridging, low mass fraction, high process limits. 100-150 and difficult control of printing.

Table 3. Advantages and limitations of polymer-based nanocomposite processing methods.

Process	Advantages	Limitation	Ref.
Infiltration	Synthesis of low costed nanocomposites based on polymers with less or even no quality	Extensive use of large amounts of solvents	3-10
Freeze/thaw from Solution	Preparation of homogeneous dispersions of the filler in the polymer process.	Efficient removal of unnecessary polymerization	111-117
In situ Interfacial Polymerization	Layer procedure, based on the dispersion of the filler in the polymer process.	Limited applications	158-161
Melt Infiltration	Environmentally benign; use of polymer not suited for other processes; compatible with industrial polymer processes.	Limited applications (carbon fibers, who represent the majority of used polymers).	161-169
Composite Synthesis	Large scale production using procedure.	Limited applications; based mainly in water soluble polymers, contaminated by side products.	170-175
Sol-Gel Process	See Table 2a.	See Table 2b.	

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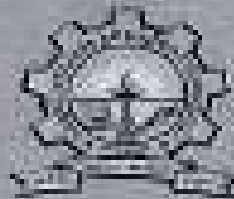
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INFLUENCE AND EFFECT ON MECHANICAL PROPERTIES OF STITCHED AND UN-STITCHED WOVEN GLASS FIBER/ EPOXY COMPOSITE LAMINATES

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ABSTRACT

The work presented in this project investigates the effect of stitched and un-stitched woven glass fiber / Epoxy composite laminates and to investigate the ability of through thickness reinforcement to improve the de-lamination resistance of laminates. Un-stitched and Stitched specimens were prepared. The test specimens are fabricated by simple hand lay-up technique and prepared according to ASTM standards and tensile test, compression and impact tests to conduct. Under low velocity impact loading, unidirectional laminates suffer from extensive delaminations damage that reduces their residual mechanical properties. To increase the resistance to delaminations, through the thickness stitching was adopted. From the experiment work to find the strength, deformation and breaking forces for both stitched and unstitched specimens. In order to distinguish the effect of fibers, a number of experiments were performed. Details of the experiments conducted along with the results were discussed in the paper and it concludes.

Keywords: Epoxy, Composite Materials, Hand lay-up, Tensile strength, flexural strength, Impact strength.

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Cryptography and Steganography are the areas of research in the field of Information Security [1]. Steganography is the technique of hiding information into the cover file i.e. text, audio, video or image without causing statistically significant modification on the cover file. The purpose of the steganography is to hide the existence of the information from the attacker [2, 3]. While embedding information in an image, a better balance is required in between the increasing embedding capacity and in reducing the stego-image distortion quality. In this proposed work, no real single bit of data was embedded in the pixels but the cover file was transmitted on the channel. Only the stego image generated with the help of differential vectors is transmitted on the channel. The cover file is retrieved with the help of repository of both sender and receiver side. Hence, there is no change to verify the presence of data on the cover image. At the destination and by using differential vector image and its cover image from the repository, the whole stego image was reconstructed. As there is no single bit of data was embedded in the cover image even a powerful steganographic technique cannot be able to detect the steganographic process.

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Date of Publication: 2004 March 22 | IEEE Access Number: 1702882



Contents

1 Introduction

Embedding is the process of embedding the data file in the alternate position of the pixel in the carrier file. The file on which data file will be embedded is called as carrier file or cover file. If the carrier file used is a color image, the data file is called as color image-embeddable image. After embedding the data file in the cover image, the cover image is called as image-image. Most of the most cryptographic algorithms will be using the Least Significant Bit (LSB) method, because it will not much affect the quality of an image.

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An Efficient and Robust Image Steganographic Technique Without Stuffing Data Bits

Authors: [K. S. Anandhavan](#) [S. J. Chinnaiyan](#)

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Cooling and Heating of a Substance Using Vapour Compression Refrigeration Cycle

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Abstract— the present invention relates to refrigeration and heating system based on vapour compression refrigeration cycle. Generally in vapour compression refrigeration cycle, refrigeration effect takes place in the evaporator as the hot part copper tubes in which cold refrigerant is flowing through the substance (Water) at evaporator side to produce cooling effect. During vapour compression refrigeration cycle heat rejection takes place in the condenser, at condenser side the refrigerant passing through copper tubes is hot due to compression, so we can utilize this heat to heat the substance (Water) by passing hot copper tubes through the substance before it passes through the condenser, here I am using two steel numbers filled with submersible water one is fixed at evaporator side to cool the substance(water) and other one is at condenser side to heat the substance(water).

Keywords—Compressor, Condenser, Evaporator, Expansion Device, Heater

1. INTRODUCTION

Refrigeration is the action of cooling, and to produce this requires removal of heat and discharging it at a higher temperature. Refrigeration is therefore the process of moving heat from low temperature to high temperature. In addition to chilling and freezing applications, refrigeration technology is applied to air conditioning and heat pumps, which therefore fall within the scope of this book. The fundamental principles are those of physics and thermodynamics, and these principles, which are relevant to all applications.

Water heating is a thermodynamic process that uses an energy source to heat water above its initial temperature. Typical domestic uses of hot water include cooking, cleaning, bathing, and space heating. In industry, hot water and water heated to steam have many uses. Fuel-fueled (natural gas, liquefied petroleum gas, oil), or solid-fuel are commonly used for heating water. There may be consumed directly or may produce electricity that, in turn, heats water. Electricity to heat water may also come from an other

electrical source, such as nuclear power or renewable energy. Alternative energy such as solar energy, heat pumps, hot water heat recycling, and geothermal heating can also heat water, often in combination with heating systems powered by fossil fuels or electricity.

II. Vapour Compression Refrigeration cycle

A. Working of vapour compression refrigeration cycle

Vapour Compression Refrigeration cycle is widely used in refrigeration systems, including refrigerators, refrigeration air conditioning, freezers, and auto air conditioners. R22 and R12 refrigerant are the most commonly used, but are being phased out due to effects on the ozone layer. The working principle of an ideal vapour compression refrigeration cycle is shown in figure.1 main parts of a vapour compression refrigerative cycle are compressor, condenser, expansion device and an evaporator. Here compressor and condenser are at high pressure side and expansion device and evaporator are at low pressure side.



Figure 1 Schematic diagram of vapour-compression refrigeration cycle





A. High pressure side

During compression the low pressure and low temperature vapour refrigerant enters to a compressor after compression the temperature and pressure of the refrigerant increases. This high pressure and high temperature vapour refrigerant passes through a condenser heat heat rejection takes place during this process the temperature of refrigerant decreases and pressure remains constant, and the phase of refrigerant changes from vapour state to liquid state.

C. Low pressure side

After condensation the high pressure and low temperature liquid refrigerant goes to expansion device which is also known as throttling device low pressure of the liquid refrigerant decreases at this point the pressure and temperature of liquid refrigerant are lower than its surrounding temperature and pressure then this low temperature and low pressure liquid refrigerant made passed through the evaporator here the refrigerant absorbs the space heat it changes its phase from liquid state to vapour state then again this low pressure and low temperature vapour refrigerant goes to a compressor to repeat the cycle.

D. Thermodynamic Analysis

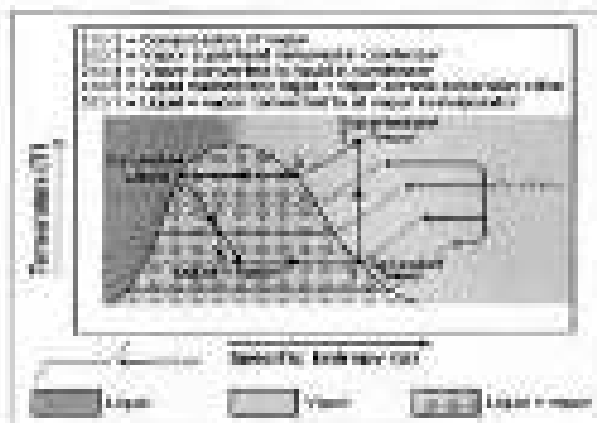


Figure 2: Temperature and Enthalpy diagram for vapour compression refrigeration cycle

The thermodynamics of the vapour compression cycle can be analyzed on a temperature-enthalpy diagram as depicted in Figure 2. At point 1 in the diagram, the refrigerant is at vapour state enters the compressor, here it

compresses the refrigerant reversibly and adiabatically (isentropically) and exits the compressor as a superheated vapour refrigerant at point 2 during this process the pressure and temperature of refrigerant increases.

From point 2 to point 3, the superheated vapour refrigerant travels through part of the condenser which removes the heat by cooling the vapour refrigerant. Between point 3 and point 4, the vapour refrigerant travels through the remaining part of the condenser reversibly and isothermally during this process temperature of the refrigerant decreases and pressure remains constant. Vapour refrigerant converts to a liquid refrigerant.

Between points 4 and 1, the saturated liquid refrigerant passes through the expansion valve and undergoes an abrupt decrease of pressure. This process is known as throttling process. It is an irreversible process during this process enthalpy remains constant therefore it is also known as isenthalpic process. During isenthalpic process the pressure and temperature decreases.

Between points 1 and 2, the cold and partially vaporized refrigerant travels through the coil or tubes in the evaporator where it is totally vaporized by the warm air (from the space being refrigerated) reversibly at constant pressure. During this process the liquid refrigerant converts to a vapour refrigerant then this saturated vapour refrigerant returns to a compressor to complete the cycle.

It should be noted that the above discussion is based on the ideal vapour-compression refrigeration cycle which does not take into account real world issues like frictional pressure drop in the system, slight internal irreversibility during the compression of the refrigerant vapour etc.

COP of Vapour Compression Refrigeration cycle

Coefficient of performance of a vapour compressor refrigeration is the ratio of refrigeration effect (heat removed in the evaporator) to the amount of work done.

$$COP = \text{Refrigeration effect} / \text{Work done}$$

$$\text{Refrigeration effect} = h_1 - h_4, \text{ work done} = h_2 - h_1$$

$$COP = \frac{h_1 - h_4}{h_2 - h_1}$$



II. OUR EXPERIMENT ON VAPOR COMPRESSION REFRIGERATION CYCLE

From figure 2 we can observe that the temperature of the superheated vapour refrigerant after compression is high at point 2 and at point 3 saturated vapour refrigerant enters in to condenser after condensation at point 4 the temperature of the liquid refrigerant decreases. So we utilize the heat which is rejecting during process 2 to 3, to heat the substance (water) after heating process 4 to 2 the temperature of the liquid refrigerant is low at point 5, during the process 5 to 1, we use use this low temperature liquid refrigerant to cool the substance (water).

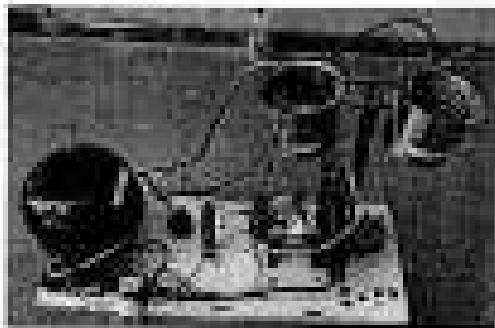


Figure 1: Diagram of a cooling and heating of a substance using vapor compression refrigeration system

Figure 3 shows the practical vapor compression refrigeration system it consists of a compressor, condenser, expansion device, two pressure gauges & two steel tanks, one (tank 1) for heating the substance (water) other one (tank 2) for cooling the substance (water) after running this vapor compression refrigeration system for some time observed that the water in the tank 1 through which superheated vapour refrigerant is passing is gets heating and the water in tank 2 through which low temperature liquid refrigerant is passing is cooling.

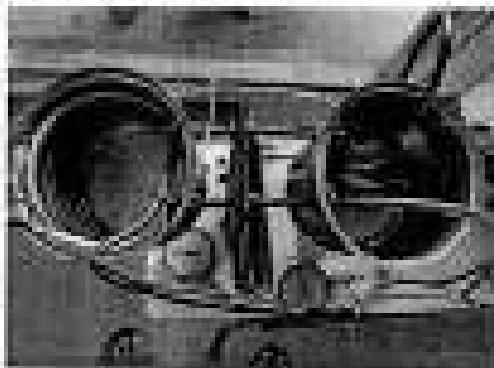


Figure 4: Diagram of water cooling and water heating in steel tank

From the figure 4 we can clearly observe that the water in the right tank gets heated and the water in the left tank gets cooled.

A. Refrigerant Used (R134a)

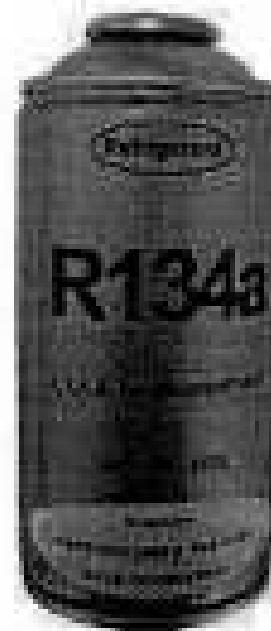


Figure 4: Diagram of Refrigerant (R134a)

R134a is also known as Tetrafluoroethane (CF₂ClCF₂) from the family of HFC refrigerant. With the discovery of the damaging effect of CFC's and HCFC's refrigerant to the ozone layer, the HFC family of refrigerant has been widely used as their replacement.

It is now being used as a replacement for R-12 CFC refrigerant in the area of centrifugal, rotary screw, scroll and reciprocating compressors. It is safe for normal handling as it is non-toxic, non-flammable and non-corrosive. The R134a is most environmental friendly refrigerant so we preferred the refrigerant R134a over R22, R12 these refrigerant cause an damage to ozone layer.

TABLE I



TABLE 1: Properties Of Refrigerant R134a

SNo	Property	Value
1	Boiling Point	-101.06 or -28.17°C
2	Air-liquid Temperature	101.06 or -28.17°C
3	Normal Evaporator Level	0
4	Solubility to Water	0.11% by weight at 33.3 or 0°C
5	Critical Temperature	110.06 or 122°C
6	Colour Code	Light Blue
7	Global Warming Potential (GWP)	138

10. Results & Discussion

Water cooling and water heating using vapour compression refrigeration cycle is successfully operated. The desired results of heating and cooling of water simultaneously is achieved and the rate of cooling and heating are as follows

TABLE 2

TEMPERATURE RESULT

SNO	Time in MINUTES	TEMPERATURE OF HOT WATER IN °C	TEMPERATURE OF COLD WATER IN °C
1	0	21	19
2	10	8	41
3	18	6	46
4	21	4	50
5	25	3	52

Table shows the temperature results of hot water and cold water in °C scale at 2°C interval in the cool tank, and then we started the vapour compression refrigeration system after 15 minutes we have checked the temperature of hot and cold water temperatures with the help of thermometer the temperature of hot water reaches to 41°C and the temperature of cold water reaches to 5°C after 18 minutes the temperature are 46°C & 6, after 21 minutes the temperatures of hot and cold water are 50°C & 9°C and our final observation at 25 minutes at that point of date the temperatures of hot and cold water are 52°C & 7°C.

A COP of Vapour Compression Refrigeration System with Water Loop

The condenser pressure we noticed in the high pressure gauge is 0.81Mpa and evaporator pressure is 0.377Mpa from

refrigeration table's enthalpies at various points are noted down

$$h_1 = 377.565 \text{ kJ/kg}$$

$$h_2 = 412.77 \text{ kJ/kg}$$

$$h_3 = 249.500 \text{ kJ/kg}$$

$$COP = \frac{h_2 - h_3}{h_1 - h_2}$$

$$COP = \frac{412.77 - 249.500}{377.565 - 412.77}$$

$$COP = 1.258$$

V. CONCLUSION

Finally we proved that for heating and cooling of a substance (water) simultaneously is possible during a vapour compression refrigeration cycle. Now a day's every house has at least one refrigerator in some food items and water heaters to heat water both water heaters and refrigerators require power to operate but by using this attachment of heater to the refrigerator we can refrigerate food in a water heater to save the power.

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Thermal Analysis Comparisons On Natural Ventilated Open Poly Greenhouse To Closed Poly Greenhouse

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Abstract: Poly Greenhouses create optimal climate conditions for crop growth and extend crop from outside parts. To achieve and sustain crop production climate, traditional Insected Poly Greenhouse design utilize a standard fan ventilation system to decrease temperature. The issue remains that such fans are energy intensive and are prone to failure when faced with tropical storms. The proposed solution is natural ventilation augmented cooling Poly Greenhouse, or NVAAC Poly Greenhouse. A NVAAC Poly Greenhouse compares insect, open-roof design improved by coupling natural ventilation.

In the present study we are analyzing the air ventilation and temperature changes in natural ventilation in closed conditions. The structural layout is analyzed to get the maximum optimal results to cope up with practical studies in present area conditions.

In the present study we are analyzing the air ventilation and temperature changes in natural ventilation in closed conditions. The structural layout is analyzed to get the maximum optimal results to cope up with practical studies in present area conditions.

Fluctuation in temperatures will alter the between summers to winter. The main application is to maintain the ambient temperatures through out the year. Such results were obtained by comparing outside temperatures to inside temperatures with the existing system functional. Two design configurations of the same Poly Greenhouse were considered open roof and open roof. The degree of sophistication also varies from a simple poly house with polyethylene film covering to highly sophisticated fully automated drip and fertigation systems, PAK lightings, full scale computerized (fully automated) systems. In conventional Agricultural practices, the crops are being grown / cultivated in the open field under natural conditions where the crops are most susceptible to various changes in

climate in the various boundary, light intensity, photo period and other conditions due to which the quality yield of a particular crop may get affected and may be decreased.

Key words: poly houses, natural ventilation, thermal conditions, temperature analysis.

1. INTRODUCTION

A Poly Greenhouse is a man-made environment for growing that is not open from the rest of the world. It is a space where certain conditions like heat, light and humidity can be controlled to a greater degree than is possible in an outdoor setting. Surely the degree of control has been a crucial factor in the increasing popularity of Poly Greenhouses.

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2. LITERATURE SURVEY

INTRODUCTION

The first study made by means of Computational Fluid Dynamics (CFD) for the study of the ventilation in Poly Greenhouses was carried out by Chaudhary et al. (1998) who compared this numerical method with the experimental results





conducted a field-based study in 1989. Although five trials showed little correlation with the experiment data, probably due to the limited period of the available computer facilities at that time, they provided important new information on the patterns of flow inside the Poly Greenhouses. This technique was applied the same time with the simulations of CFD were compared to a free span Poly Greenhouse with the data obtained by means of smoke photography (De et al., 1995; Debnath et al., 1996; Misra et al., 1997) studied bidirectional open Poly Greenhouses using CFD is a two-dimensional grid and they concluded that CFD is a powerful tool to develop improvements in the design of Poly Greenhouses for efficient ventilation.

1.1. Types of Poly Greenhouses

Keeping in view the better climatic conditions, two types of Poly Greenhouse have been there and considerable under the Govt. of India schemes i. National Horticulture Mission (NHM), Horticulture Mission for North East and Himalayan States (HIVES) for subsidy, and they are Naturally Ventilated Poly Greenhouse and Poly Greenhouse with Fan & Fuel Cooling System.

1. Open type Poly Greenhouse System
2. Closed type Poly Greenhouse System

1.2. Open type of Poly Greenhouse System

One of the main advantages of a naturally cool system is that the cost of the structure is not limited by the mechanical capacity of mechanical ventilation systems. Poly allows the wind passing over its lower edges, resulting in a more uniform growth response from the entire crop.

1.2.1 Naturally ventilated Poly Greenhouse



Fig. 1.2.1 naturally ventilated Poly Greenhouse

In order to enhance the design of the naturally ventilated Poly Greenhouse is popular for its level of protection and control it provides. Poly also particularly since when the natural conditions outside make ventilation impossible or inadequate. At some level of control improvements in their growing climate still need.

While growing in Poly Greenhouse and also the dependence on climate conditions outside, plant best is a full dependent on an open to complete set of means of climate conditions inside. The key to successful growing, growing and production, lies in recognition

of these conditions and their impact, and maximizing control of them.

Choosing inside Poly Greenhouse allows plants to grow in temperatures typically between 13-20 °C and relative humidity of around 60-80%. The reason these conditions are ideal is because the plant's rate of photosynthesis is optimized at this point.

Features

- + Maximum natural ventilation
- + Transmittance controlled Poly film edges and internal coating
- + Available with either double poly film or polyethylene covering
- + Increased open gate, increased
- + Protection of humidity and air complete fanless operation
- + Wide range of options to fit your needs

1.2.2 Siting for Natural Ventilation System

For some Poly Greenhouses, like a hoop house, simply rolling up a portion of the wall is sufficient, whereas in this is needed for natural ventilation. However, Poly Greenhouses that are built from more rigid materials need to follow a few guidelines to ensure proper ventilation. The most important aspect of natural ventilation for these Poly Greenhouses is the design of the ridge (roof) eave and sidewall vents. It is recommended that the total combined ridge vent area should be equivalent to the total combined area of the sidewall vents and each should equal approximately 20% of the floor area.

1.2.3 Temperature Control Poly Greenhouses

The Natural Ventilated Poly Greenhouse regulates outdoor breezes during the temperature in the greenhouse. One Poly Greenhouse (F100) that has been designed from the perspective of the use of natural ventilation for plant growth.

1.2.4 Air Circulation

Air circulation is an important within a Poly Greenhouse system many purposes. Proper air circulation ensure uniformity in temperature, humidity, CO₂, and oxygen within the environment. Plants respond better to environmental consistency and proper air circulation ensures each plant within the Poly Greenhouse receives the same atmospheric conditions.

1.3 Closed type Poly Greenhouse

Modern Poly Greenhouse production requires a high level of environmental control. In this respect, it has been suggested that fully closing a Poly Greenhouse, i.e. without air exchange with the outside environment, would have a number of advantages. A fully closed Poly Greenhouse could be a significant improvement in plant breeding. The main challenge of closing the Poly Greenhouse is to be able to continuously control temperature (T) and relative humidity (RH) within



lighting, heating and cooling conditions, the environmental control and management objectives included as follows:-

- Climate control system.
- Temperature distribution (horizontal and vertical).
- Energy use.

2.3.1 Poly Greenhouse with fan and pad cooling system (Mechanical Poly Greenhouse)

This type is most suited for hot dry climate conditions such as hotland plains where temperature goes above 30°C. The system is working on the principle that when water evaporates heat loss is absorbed from surroundings. This is possible by drawing cool air into Poly Greenhouse through wet pad located in one end of the hall, where hot air is replaced by cool air that moves on the opposite end of wall. The excessive temperature causes various damages to the morphology and physiological processes of plants such as flower shedding, leaf scorch, poor fruit quality, excessive transpiration, shortened life span of the plant and low crop photosynthesis due to excessive transpiration etc. Hence, it is important to maintain air temperature inside the Poly Greenhouse ranging between 21°C to 28°C for best crop growth.



Fig. 2.3.1 Poly Greenhouse with fan and pad cooling system

2.3.2 Orientation criteria for mounting fan & pad

The orientation of the multi span Poly Greenhouse with fan & pad cooling system should be made in such a way to take advantage of prevailing wind direction. Wet pad should be mounted on the wind of Poly Greenhouse facing inside and fan on the leeward side. The temperature gradient inside the Poly Greenhouse inside fan and fan & pad system. The wet side should be on the leeward side to prevent structural damage to the Poly Greenhouse.

2.3.3 Factors to reduce average temperature are as follows:

- Seal all cracks and holes in the structure
- Limit the maximum distance between fan & pad to 21 m.
- Decrease the distance of fan fan distance as flow
- Provide additional cooling facility if it necessary
- In case of failure to reduce water, use fan type of pad to reduce heat downwards through the pads has a tall

Working effect

- The water used for wetting the pad should be pumped from outside ground tank and provision to be made for automatic water.
- Uniform water pressure of the system must be maintained to keep the pad completely wet.
- The cooling efficiency of Poly Greenhouse can be increased by using electric fan up to 50%.

2.3.4 MATERIALS AND METHODS

To control the inside climate, air will be supplied and conditioned by means of air conditioning units. These units are equipped to cool, dehumidify, and heat the air. After treatment, the air is distributed through air distribution ducts that are made by extrusion and injection. The ends of the ducts are designed to make control vertical and horizontal temperature distribution. Thus, a grid will be used to control distribution of air, reduce humidity, and CO₂ concentration.

3. Design and Materials are structure of Poly Greenhouse

Poly Greenhouse are houses of white (transparent) color, with a transparent material in which crops are grown under natural environmental conditions. Poly Greenhouse utilization as well as other types of controlled environment cultivation has been studied to assess throughly possibilities, which shows the crop production could be possible all through the year or part of the year as required. Poly Greenhouses and other technologies for controlled environment plant production are associated with the efficient production of transpiration and both of high yield to crop density when relative humidity production is not possible. The primary environmental parameter traditionally controlled is temperature, usually providing heat to increase moisture and humidity. However, environmental control can also include cooling or drying conditions. Transpiration, light control either shading or cooling supplements, both carbon dioxide levels, relative humidity, plant growth and yield production.

3.1 Types of Glazing Materials

The most common Poly Greenhouse glazing materials are glass, rigid plastic and plastic film.

Glass

Glass has the highest light transmission, and the longest life span (30-40 years) and is the most expensive. Tempered or laminated glass is recommended because of its strength allowing the front support bars, and it is unbreakable safety for people working.

Plastic Film

Plastic film (e.g., polyethylene) are the cheapest Poly Greenhouse shading material. Laminated Poly Greenhouse material does have approximately 40 percent of the heat loss associated to single film material with a single layer of glass or plastic. It is important to always use exactly one to reduce the heat input of film laminated film will significantly reduce potential condensation between the layers.



3.2 Design

The designer has to carry the following loads and is to be designed accordingly:

a) Dead load: weight of all permanent construction, flooring, masonry and ceiling, equipment, water pipes and all other service equipment to be fixed.

b) Wind load: The structure should be able to withstand wind of 110 kph/min or less and or less 50 kg per square meter of wind pressure.

c) Snow load: There are no burden or put the area appraised full of the location.

3.3 Wind effects

If the Poly Greenhouse is naturally ventilated, the advantage of natural wind direction has to be taken to the maximum possible.

The maximum dimension (height) of Poly Greenhouse should be perpendicular to the wind direction especially in common. For hot and hot-humid climatic conditions the natural wind direction should be taken as the design wind.

4. MODELS AND METHODS

CFD application in Poly Greenhouse Development. Computational Fluid Dynamics is a sophisticated design and analysis tool that uses computers to simulate fluid flow, heat and mass transfer, pressure change, structural reaction, mechanical movement, and solid and fluid interaction. The technique creates a 3D operational model of a physical system to be studied under many different design conditions. CFD had the ability to efficiently develop spatial and temporal field solutions of fluid pressure, temperature and velocity, and the predict its effectiveness in system design and optimization within many industries. The application of CFD in the agricultural industry is becoming more prominent due to the above mentioned factors. The versatility, accuracy and applicability offered by CFD has led to its increased adoption by the agricultural engineering community to study and optimize the indoor climate of the Poly Greenhouse. This is reported by Hansen et al. (2017) for the Institute for past research papers of CFD application in agricultural buildings.



Figure 1: The number of published peer-reviewed publications of CFD application in the construction of agricultural buildings. (Nathan et al., 2021).

The values of the thermophysical properties of the gases in the air used as boundary conditions were calculated using software tools (Palau et al., 2004). The crop was also simulated using the porous media approach to the influence of a commercial greenhouse. However, the drag effect of the crop, in the simulated Fluid flow operation.

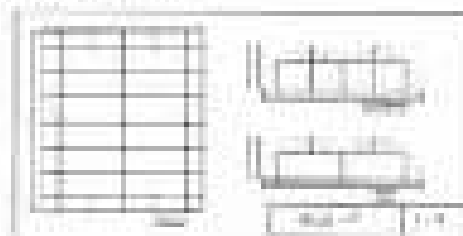


Fig 2.1 Poly house structure layout

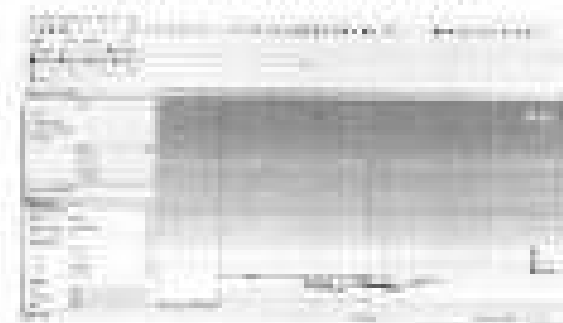


Fig 3.1 3D layout of green house

4.2 Heat Loss Calculations

Heat loss by conduction only is estimated with the following equation:

$$Q = A(U) \cdot T_{ext} - T_{int}$$

Where: Q = Heat loss, BTU/hr

A = Area of Poly Greenhouse surface, sq ft

U = U-value across heat flow

T_{int} = Polyethylene or other GEL, single layer 3.0

T_{ext} = Air temperature, different between inside and outside.

Less 4.0 for materials commonly used in Poly Greenhouse construction and their associated R-values.

Also take overall U-value for various construction assemblies. Note that high U-values indicate less heat flow. Use dry materials that absorb moisture will conduct heat faster than an 1-in. Use vapor barrier to prevent moisture that are permeable to water vapor.

There is also heat to the ground underneath and beneath a Poly Greenhouse. The perimeter heat loss may be added as other losses using the following equation:

$$Q = PL(T_i - T_g)$$

P = Perimeter heat loss coefficient, BTU/hr °F ft

L = Distance around perimeter

T_g = Substrate heat loss, in the construction heat losses. The equation for estimation heat transfer



Relative

$$Q = 0.02 V C_p (T_1 - T_2)$$

$V =$ Velocity of air flow, m/s

$C_p =$ Number of air exchanges per hour

Less volume of air exchanged through types of Poly Greenhouses. The number of air exchanges per hour will vary depending on the type and condition of the Poly Greenhouse and parameters of wind.

Maintain a temperature of 10 degrees inside a double layer plastic Poly Greenhouse with dimensions as well as insulation materials.

Amount of Required Energy

4.1 Surface Area

$$\text{Wall} = 7 \times 100 \times 2 = 1400 \text{ m}^2$$

$$\text{Roof} = (100 \times 10) \times 2 = 2000 \text{ m}^2$$

$$\text{End} = (20 \times 7) \times 2 = 280 \text{ m}^2$$

$$\text{Total} = 3680 \text{ m}^2$$

Conduction Heat Loss, Q_c

$$\begin{aligned} Q_c &= \text{Area} \times U \times \Delta T \\ &= 3680 \times 0.14 \times 40 \\ &= 20607 \text{ BTU/hr} \end{aligned}$$

$$\begin{aligned} \text{Infiltration} &= (7 \times 10 \times 10) + (10 \times 10 \times 100) \\ &= 21,000 + 1,000 \\ &= 22,000 \text{ m}^3 \end{aligned}$$

Air Infiltration Loss, Q_i

$$\begin{aligned} Q_i &= 0.02 \times \text{Volume} \times T_1 \times \Delta T \\ &= 0.02 \times 22,000 \times 10 \times 40 \\ &= 17,600 \text{ BTU/hr} \end{aligned}$$

Radiation Heat Loss, Q_r

$$\begin{aligned} Q_r &= T_1 \times E \times (A_1) \\ &= 0.05 \times 200 \times 40 \\ &= 4,000 \text{ BTU/hr} \end{aligned}$$

Total Heat Loss, Q_T

$$\begin{aligned} Q_T &= Q_c + Q_i + Q_r \\ &= 20607 + 17,600 + 4,000 \end{aligned}$$

Heat Required = 42,207 BTU/hr

Various factors such as temperature during winter days and supply of water should also be considered in the plant production. Another advantage of polyhouse is to reduce water evaporation and reduce it with clear air. High humidity is undesirable since it causes various condensation on seed packets and tends to increase the incidence of diseases.

5. RESULTS AND DISCUSSIONS

5.1 OPEN TYPE GREEN HOUSE



Fig 5.1.1 Velocity contour for open ventilation greenhouse structure



Fig 5.1.2 Velocity contour for open ventilation greenhouse structure



Fig 5.1.3 Temperature contour for open ventilation greenhouse structure



Fig 5.1.4 Pressure contour for open ventilation greenhouse structure

5.2 Closed Type Analysis results



Fig 5.2.1 Velocity contour for closed type ventilation greenhouse structure



5.1.1 Velocity vectors for closed type vertical greenhouse structure



Fig. 5.1.2 Temperature contour plot for closed type vertical greenhouse structure



5.1.4 Pressure contour for closed type vertical greenhouse structure

5. CONCLUSIONS

1. For each case, the distribution of temperature inside the Poly Greenhouse was quite different, and the resultant temperature profile was greatly affected by airflow. When the wind came with a velocity equal to 1 m/s and a temperature equal to 1°C, the temperature inside the Poly Greenhouse decrease significantly from 17°C to 14°C. This decrease depends largely not only on the wind velocity and its temperature but also on its direction, as stated.

This paper describes and evaluates the computational facilities using the finite element method to study the effect of natural ventilation on Poly-house indoor air properties especially during the day.

2. The influence of an insect proof screen on airflow and temperature patterns was numerically investigated for a closed Poly Greenhouse with continuous hot springs by means of a CFD package. The simulation results indicated that the screen had a considerable effect on Poly Greenhouse climate. A screen through reduced air velocity inside the Poly Greenhouse (especially near the crop area) and further air (20%)

injection resulting in a significant temperature increase. With respect to outdoor temperature (27 °C), maximum temperature augmentation was 1 °C higher for the Poly Greenhouse without screen and up to 7 °C higher for the Poly Greenhouse with screen.

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DESIGN AND ANALYSIS OF TORSEN DIFFERENTIAL

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Abstract— The Torsen differential is a purely mechanical device. It has no electronics, solenoids or viscous fluids. The Torsen (from Torque Sensing) works as an open differential when the amount of torque going to each wheel is equal. As soon as one wheel starts to lose traction, the difference in torque causes the gears in the Torsen differential to bind together. This paper presents an automotive differential in that if one wheel is held stationary, the counterpart wheel takes all torque. Its normal speed as can be seen by examining the complete scheme of automobile differential. The wheel without traction will spin without providing traction and the opposite wheel will stay still so that the car does not move. Torsen Differential is designed in CREO 2 and the fabrication of the gears in the differential is done, with its modelling and analysis is done in ANSYS 16 comparison with other differential, by this the traction which is produced will be reduced and slipper's of vehicle in road and wet roads will be completely avoided. Instead of going with All Wheel drive which is expensive in the market. How design of worm and worm wheel gear shaft, key and ballnut gear is done in CREO 2 and analysis is done in a gear tooth of open gear to find the efficiency in comparison with the standard worm wheel gear. By the use of Torsen Differential in an Automotive Vehicle the torque which is produced will be equally distributed even when the vehicle is moving in hilly areas or in mud or in wet areas instead of going with 4WD which is expensive. This also maintains the cost and efficiency of worm and worm wheel increases with the conventional one about 64 percent more.

Keywords—Torsen Differential, Worm and Worm Wheel gear, Ansys 16, CREO 2

INTRODUCTION

All the Automotive Vehicles moves with the differential, without the differential a vehicle cannot make a turn and control adjust its speed accordingly as axle differentials are used in all vehicles. In present day life some vehicles run with Front wheel drive and few vehicles run with rear wheel drive and few with all wheel drive, generally all wheel drive is used in off ROV's which can run everywhere in hilly areas, mud, wet areas with all wheel driving but it is quite expensive which is not affordable to all. This is the reason for a driver looks at a limited slip differential or traction control. The solution to this above problem is to have a Torsen Differential. This Torsen Differential works as an open differential when the amount of torque going to each wheel is equal. As soon as one wheel starts to lose traction, the difference torque causes the gears in the Torsen differential to bind together. The design of gears in differential determines the torque loss side. When the left wheel is spinning faster than the right wheel which is used transfer the torque which is available in the left wheel to right wheel by this torque will be equally distributed to both the wheels. (Even in India specially for most common vehicles used in Hilly Roads and present days Cross-over's are used but these vehicles when moving in shoddy places and hilly areas the kind of open differential which is used right now cannot equally distribute the torque to both, in order to avoid this problem a new kind of differential has to be used to avoid such affordable cost as Cross-over's. This will reduce the skidding in wet areas as well as over cost also be affordable.

LITERATURE SURVEY





Iman Rajjal [5], has performed the two Primary function of a differential system is to allow the driven wheels to rotate at different angular velocities relative to each other and also to transfer power to them from transmission. The first function is important to allow smooth turning as can be seen in the following Fig.1 which illustrates a left-hand turn.

Mich, K. and Buchanan, [2], has performed a Open differential make use of a planetary gear set mechanism which distributes torque equally between the drive axles while allowing the wheels to rotate at different rates. The input shaft transmits torque from the driveline to a large ring gear within the differential carrier. When the vehicle is travelling in a straight line the mechanism remains straight and the differential casing rotates at same rate as the drive axles. As the vehicle turns the two the gear set engages and the casing of the pinion gear allow the drive axle to rotate at different speeds [5].

Harlock J.H [6], has done an experiment in a model of the forces on the tires of the driving wheel has been presented by [6]. The forces transmitted by a tire to the ground are separated into longitudinal and lateral components. The longitudinal force is responsible for forward motion. When torque applied to the wheel exceeds the maximum longitudinal force the tire can transmit, slippage results. The maximum force transmittable by the tire is a function of the normal force acting against the bottom surface of the tire and thus also a function of the amount of load above the tire. During high speed cornering, the weight of a vehicle shifts away from the turning direction due to inertia creating a higher load on the outer wheel and a smaller load on the inner.

H. B. Panglos, [7], has performed the analysis on the outer wheel ought to receive a progressively larger amount of torque over time as the load shifts. Several alternate solutions have been developed to mitigate the problems experienced by open differentials during the negotiation of turns at high speed. The maximum transmittable longitudinal force is therefore reduced for the inner tire and increased for the outer tire, but the outer tire becomes capable of handling more torque and the inner tire, too. As torque is divided equally between both drive axles in an open differential, the inner wheel soon experiences more torque than its tire can transmit causing a slippage and loss of control. In high performance conditions, this instability can pose a significant risk to the occupants safety.

DESIGN

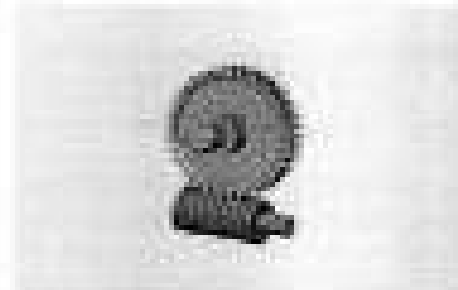


Fig 1 Worm and worm wheel is driven in two parameters 2D software

Worm gear drives are used to transmit power between two non-intersecting shafts, which are, in general, at right angles to each other. The worm is a threaded screw, while the worm wheel is a toothed gear. The teeth on the worm wheel mesh with the threads on the worm and give rise contact between mating points.

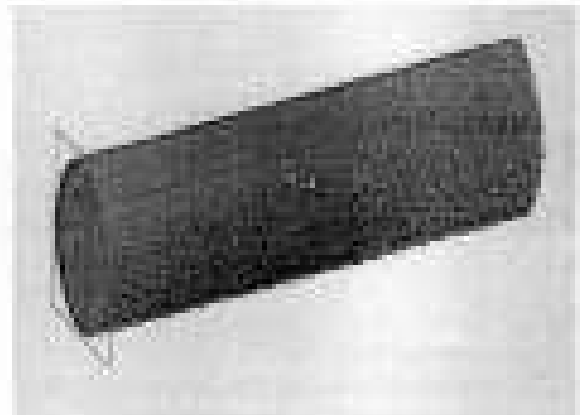


Fig 2 Shaft is driven in two parameters 3D software

In order to transfer the power from one shaft to another, the various machine tasks as pulleys,gears etc., are mounted on it. These members, along with the forces exerted upon them cause the shaft to bend (to other words, we may say that a shaft is used for the transmission of torque and bending moment)

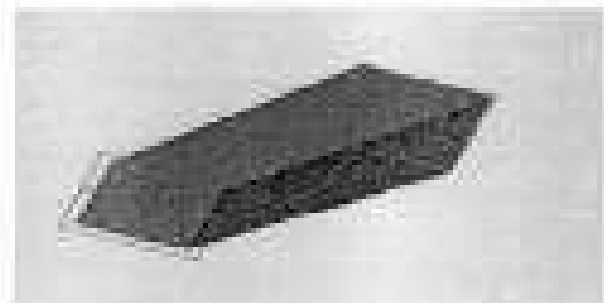


Fig 3 Key is driven in two parameters 3D software

Key is a piece of metal used to connect the shaft and hub or base of the pulley to connect their together in order to



prevent relative motion between them. It is always inserted parallel to the axis of the shaft. Keys are used as temporary fixings and are subjected to considerable shearing and shearing stresses. A keyway is a slot or groove in a shaft and hub of the pulley, to accommodate a key.



Fig 4 Propeller spline shaft gear

The propeller shaft identifies points from the gear sets, to the rear side. The spline on the shaft of the propeller shaft fit perfectly into the spline in the sleeve. This allows a tight rotation between the driving and the driven side to vary slightly without damaging the sleeve and input bearings.



Fig 5 Differential gear assembly with motor gear

With a single shaft gear, the gear 30P runs at the same, the worm gear reduction only one rank of the gear. Therefore, regardless of the worm also, the gear ratio is the size of the worm gear - to - 1. Given a single gear worm, a 12 rank worm gear will reduce the speed by the ratio of 12:1. With gear gear, a gear of 12 rank (the smallest size permissible, it depends on gear engineering practical) would have to be mated with a 240 rank gear to achieve the same ratio of 20:1.

Fig 6 Shaft gear for the wheel

ANALYSIS

Static structure



Fig 7 shows the worm and worm wheel single mesh it considered for the analysis here in which load is applied axially

with this all the forces gets in it is numerically and failure of that mesh in different stages is calculated.

Static Structure Failure



Fig 8 shows the failure analysis on bearing in different stages structure and it what load it could still maintain set up to what load it can with steel structure is shown here in this analysis. And the efficiency on bearing by this loads is related to power and torque.

Thermal analysis

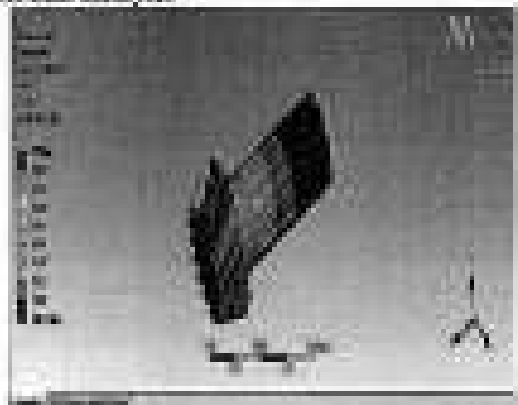


Figure 9 shows the temperature distribution on the gear teeth, minimum of 150°c was found. The initial temperature of the gear teeth was 100°c which was due to preheating. The teeth had the maximum temperature of 400°c which was applied temperature.

Direction Heat flux along X axis

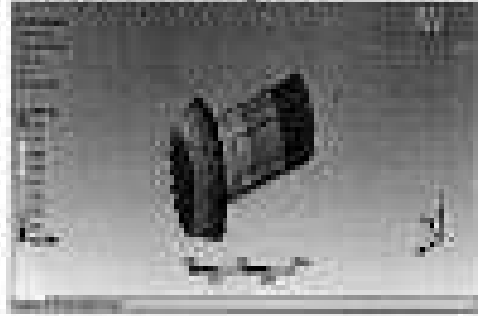


Figure 10 shows the thermal stress of the teeth, maximum of 220 was observed and the maximum thermal stress observed in the analysis was found to be below the permissible error and this shows that complete analysis has been successfully carried under with the teeth.

Heat flux along y Axis

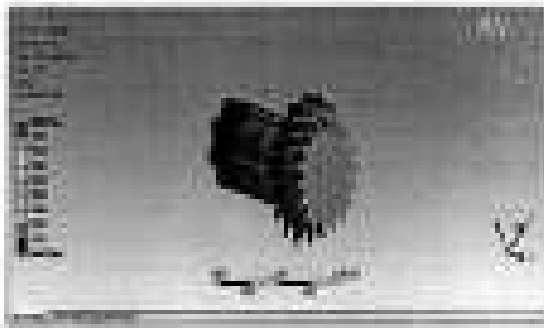


Figure 11 shows the Heat flux variation across the side of tooth. The maximum heat flux was found to be 4 (W/cm²) along the side face of the tooth. The total heat flux was maximum at the top face due to thermal load specified. Heat flux is quantity of heat flowing per unit area.

Heat flux along X Axis



The Figure 12 shows the variation of heat flux in X direction along the tooth.

The heat flux along X direction is very consistent at the opening face of the tooth. This shows the non uniformity of temperature distribution on the gear tooth for the thermal load of 149°C.

Temperature distribution

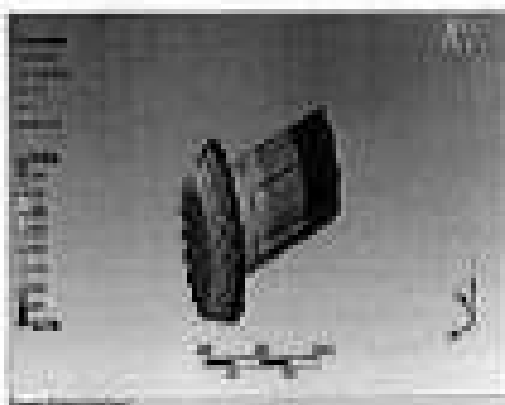


Figure 13 shows the thermal error of the tooth, maximum of 200 was observed and the maximum thermal error obtained by the analysis was found to be below the permissible error and this shows that transient analysis has been successfully carried out on the tooth.

Temperature distribution along XYZ axis

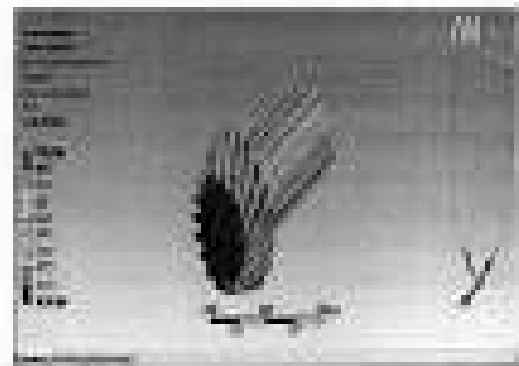


Figure 14 shows the temperature distribution on the tooth, and minimum of 81°C was found. The tooth has the maximum temperature of 325°C, which was applied temperature.

Temperature distribution along XY Plane

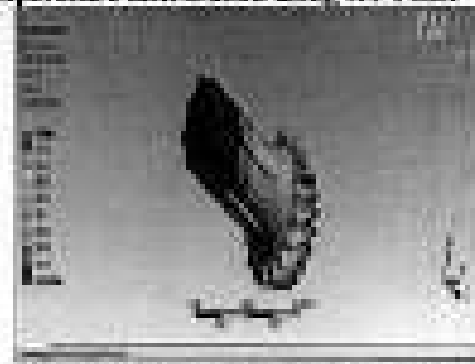


Figure 15 shows the temperature distribution on the tooth, and minimum of 149°C was found. The tooth has the maximum temperature of 325°C, which was applied temperature.

Temperature distribution along YZ Plane

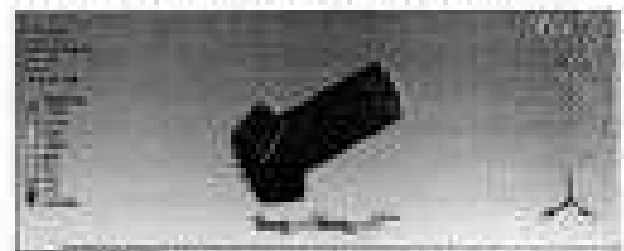


Figure 16 shows the temperature distribution on the tooth, and minimum of 216°C was found. The tooth has the maximum temperature of 325°C, which was applied temperature.

Temperature gradient





Figure 17 shows the temperature distribution on the teeth in 200 place. The initial temperature of the test specified as 1-0°C, which was due to preheating. The tooth has the maximum temperature of 120°C which was applied temperature.

RESULTS & DISCUSSIONS

TABLE 1 Shows the observation of experimental system efficiency in comparison of load with other. Increasing the load simultaneously on helical gear, the effort and efficiency of these gear are calculated with the conventional gear which gives an average value of about 33 percent approximately.

Observation Table for conventional system

Sr No.	Load (%)	Effort (N)	Efficiency	Avg. Efficiency
1	14.71	1.738	51.888	31.78%
2	19.67	1.951	50.484	
3	24.52	1.775	51.779	
4	29.43	1.968	50.484	
5	35.24	2.060	51.286	
6	44.14	2.305	51.558	
7	48.87	2.432	51.880	

Table 2 shows the comparing the above results from the conventional system to the actual system considered efficiency of the running worm gear is increased. Increasing the load simultaneously on helical gear, the speed, torque, power and efficiency of the worm and worm wheel gear are calculated with the conventional gear which gives an average value of about 34 percent approximately which is more than the conventional.

Result table for running worm gear system

Sr No.	W.G. (rpm)	Speed (rpm)	Torque (Nm)	Power (Watt)	Efficiency (%)	Avg. Efficiency (%)
1	100	3.48	1.893	1.13	61.8	64.28%
2	200	3.26	1.871	1.26	63.1	
3	300	3.18	1.76	1.407	61.8	
4	400	3.26	1.76	1.64	62.71	
5	500	3.48	1.77	1.86	63.7	
6	600	3.79	1.76	1.87	63.7	
7	700	3.50	1.76	1.86	64.5	
8	800	3.50	1.80	1.87	64.5	
9	900	3.50	1.76	1.76	63.7	
10	1000	3.50	1.76	1.76	63.7	

Worm Gear and Helical Gearing From This System

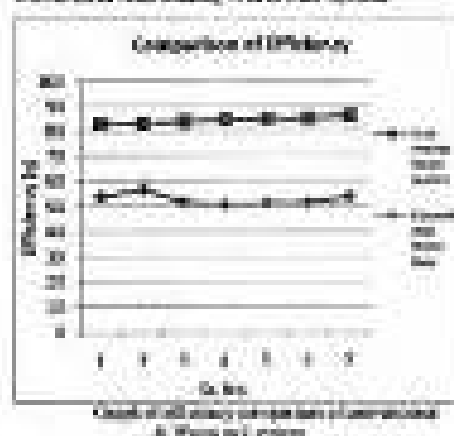


Fig 4 shows the efficiency of worm gear system in comparison with the conventional to standard experiment values, by using worm gear gear set we are achieving 51% efficiency. By increasing the load on the worm and worm wheel gear gradually the effort is also known from conventional gear in comparison with the standard gear values which is higher than the previous one.

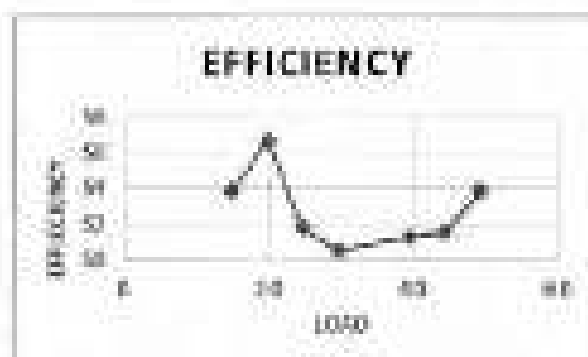


Fig 5 shows the efficiency of worm gear system in comparison with the load to standard experiment values, by using worm gear gear set. As the load is increasing the efficiency first increases and then decreases slowly and then increases finally by loading the maximum load.

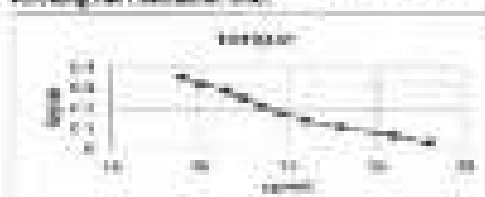


Fig 6 shows the torque variation of worm gear system in comparison with the speed to standard experiment values, by using worm gear gear set. As the speed of the vehicle is increased initially the torque produced could be high initially and then when it reaches to the maximum speed it has a low minimum torque in this figure shows the same where initially it is high and then finally reaches to the optimum value.

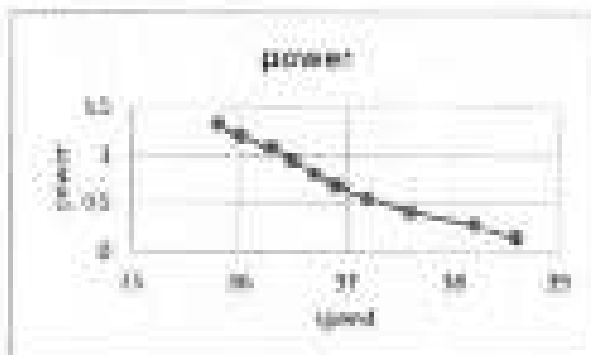


Fig. 11 shows the power variation of worm gear system in comparison with the speed to standard experiment values, by using worm gear pair. Here by increasing the speed of the vehicle the power has to be increased as when the vehicle is turning or moving in the zigzag area the power produced has to shared equally by the gears so it should be already low and it should increase by increasing the speed this is achieved here.

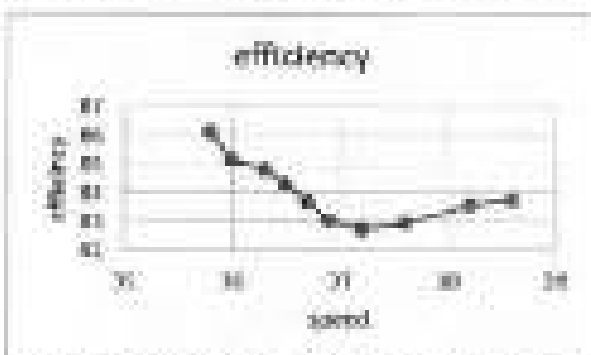


Fig. 12 shows the power variation of worm gear system in comparison with the speed to standard experiment values, by using worm gear pair. By slowly increasing the speed of the vehicle initially which has the minimum efficiency for the worm wheel has to be increased and then slowly it decreases and then reaches to the maximum value which is achieved 85 more than the conventional car.

CONCLUSIONS

This work would help in realizing a complete system with FWD or RWD. A more logical and realistic method is proposed considering simulation as well as further real-time implementations of Torque differential in different steps of analysis. As simulation proved a promising outcome a real-time for all vehicles is a future work.

The presented work was aimed at reducing the sliding and pits in rainy season in all vehicles without oil wheel drive. The drive shaft of a constant value of a car was chosen for determining the dimensions which were then used for creating a model in CAD. Using a complex availability for a number of parts it has to be analyzed in ANSYS 14 work bench.

Despite the improvements made in design, there is of course still room for improvement.

The finite element analysis is performed to analyze the efficiency of mesh and gears when gear. It is seen that the efficiency is approx. 84% which is more than conventional gears gear. However from it concludes that design is not. In concluding the work done for torque differential is very much useful in reducing a considerable amount of loss due to the transmission through a differential in all FWD, RWD vehicles from pits in rainy season and in the zigzag areas this could be a reason for the above said solution.

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CFD ANALYSIS ON TURBULENT CONVECTION HEAT TRANSFER WITH Fe₃O₄ NANOPARTICLES IN WATER

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Abstract— A theoretical study of single phase through a pipe will be carried out. The CFD simulation of heat transfer characteristics of a nanofluid in a circular tube under constant heat flux will be considered using CFX solver (version 15.0) in the turbulent flow. Fe₃O₄nanoparticles in water with concentrations of 0.02%, 0.1%, 0.2%, and 0.5% will use in this simulation. All of the thermo-physical properties of nanofluids are assumed to be temperature independent. The average particle sizes of 20 nm will be used in this research. The goal is to find out the change in heat transfer coefficient with changing the Reynolds number and the concentration of nano-particles. The maximum convective heat transfer coefficient will be observed with the change in concentration of nano-particles in water. The computational fluid dynamics (CFD) model equations will be solved to predict the hydrodynamic and thermal behaviour of the pipe. The geometry of the problem and meshing of it will be done in ANSYS Workbench.

Keywords— Nano Technology, Heat Transfer, CFD Analysis

1. INTRODUCTION

Thermal conductivity of Fe₃O₄ nanofluids is explained by many researchers. Most of the experimental work is undertaken for the estimation of heat transfer coefficient of Al₂O₃ and Cu nanofluids in plain tube and some researchers have concentrated for the estimation of heat transfer of coefficient of Al₂O₃ nanofluid in plain tube with twisted and wire coiled inserts.

Thermal conductivity of magnetic Fe₃O₄ nanofluids literature is available, experimental

turbulent convective heat transfer and friction factor of Fe₃O₄ magnetic nanofluid for tube flow and with twisted tape inserts data is not available. The advantage with this fluid is separation of magnetic nanoparticles (Fe₃O₄) from the base fluid is possible, which is not possible with non magnetic (Al₂O₃/Cu and TiO₂) type nano particles. The present investigation is carried out to estimate turbulent forced convective heat transfer and friction factor at different volume concentrations of Fe₃O₄nanofluid in a plain tube under turbulent flow conditions. Based on the experimental data generalized regression equations are developed for Nusselt number and friction factor.

PREPARATION OF NANOFUID

There are two primary methods to prepare nanofluids. A two-step process in which nanoparticles or nanotubes are first produced as a dry powder. The resulting nanoparticles are then dispersed into a fluid in a second step. Single-step nanofluid processing methods have also been developed. Several studies, including the earliest investigations of nanofluids, used a two-step process in which nanoparticles are first produced as a dry powder. This method is more extensively used to produce nanofluids because nanopowders are commercially available nowadays.

The nanoparticles may agglomerate during the drying, storage, and transportation process, leading to difficulties in the following dispersion stage of two-



very method. Consequently, the stability and thermal conductivity of nanofluid are not ideal. In addition, the production cost is high.

OBJECTIVES

A theoretical study of single phase through a pipe will be carried out. The CFD simulation of heat transfer characteristics of a nanofluid in a circular tube under constant heat flux will be considered using CFX solver (version 13.0) in the turbulent flow. Fe₃O₄nanoparticles in water with concentrations of 0.02%, 0.1%, 0.2%, and 0.6% will used in this simulation. All of the thermo-physical properties of nanofluids are assumed to be temperature independent. The average particle sizes of 36 nm will be used in this research. The goal is to find out the change in heat transfer coefficient with changing the Reynolds number and the concentration of nanoparticles. The maximum convective heat transfer coefficient will be observed with the change in concentration of nano-particles in water. The computational fluid dynamics (CFD) model equations will be solved to predict the hydrodynamic and thermal behavior of the pipe. The geometry of the problem and meshing of it will be done in ANSYS Workbench. The models will get solved by ANSYS CFX 13.0 solver.

Enhancement Mechanisms

There are many studies in the literature regarding the effect of Brownian motion on the thermal conductivity of nanofluids. Bhattacharya et al. [64] used Brownian dynamics simulation to determine the effective thermal conductivity of nanofluids, by

considering the Brownian motion of the nanoparticles. Effective thermal conductivity of the nanofluid was defined as

$$k_{eff} = \phi k_p + (1 - \phi)k_f,$$

where k_p is not simply the bulk thermal conductivity of the nanoparticles, but it also includes the effect of the Brownian motion of the nanoparticles on the thermal conductivity. A method called the Brownian dynamics simulation was developed, the expressions were provided to substitute k_p , then the effective thermal conductivity of Carboxylene glycol and Al₂O₃/ethylene glycol nanofluids were calculated for different particle volume fractions. The results were compared with previous experimental data [43, 60] and they were found to be in agreement. The prediction of the Hamilton and Crosser [44] model (Eq. 2, 1) for these two nanofluids was also included in the comparison. It was found that conduction-based Hamilton and Crosser model underpredicted the effective thermal conductivity of the nanofluid, since it does not take into account the formation motion of the particle within the base fluid.

Importance of numerical simulations:

Numerical simulation of fluids, often known as CFD (computational fluid dynamics), has become one of the three basic approaches that can be employed to solve problems in fluid dynamics and heat transfer, the other two approaches being experimental and analytical methods. CFD not only employs disciplines of fluid mechanics with mathematics, but it also uses the knowledge of computer science. With the rapid advancement of digital computers, the numerical aspect has emerged as another viable approach in



studying fluid and heat transfer problems. Density of nanofluids can be determined by using the following expression

$$\rho_{nf} = \phi\rho_p + (1 - \phi)\rho_f \quad (1)$$

Here, ϕ is particle volume fraction and subscript of ρ , p , and f correspond to nanofluid, particle, and base fluid, respectively. Pak and Cho [12] experimentally showed that Eq. (1) is an accurate expression for determining the density of nanofluids.

Specific Heat

There are two expressions for determining the specific heat of nanofluids

$$c_{p,nf} = \phi c_{p,p} + (1 - \phi)c_{p,f}$$

$$(\rho c_p)_{nf} = \phi(\rho c_p)_p + (1 - \phi)(\rho c_p)_f$$

It is thought that above Eq. is theoretically more consistent since specific heat is a mass specific quantity whose effect depends on the density of the components of a mixture.

SIMULATION OF SINGLE PHASE FLUID FLOW:

SPECIFICATION OF PROBLEM:

Consider a steady state fluid flowing through a circular pipe of constant cross section as shown in Fig. 3. The diameter and length of circular channel are 0.014 m and 1.7 m, respectively. The inlet velocity is 0.1 m/s, which is constant over the inlet cross-section. The fluid exits into the ambient atmosphere which is at a pressure of 1 atm.

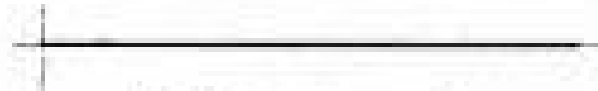
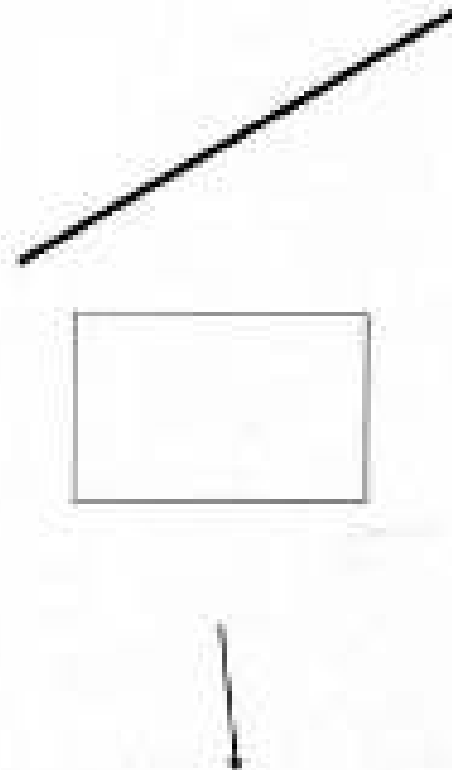


Fig.3 circular pipe geometry

As fluid flows through in a pipe at both hydraulic and thermally fully developed condition, the Nusselt number is constant for laminar flow and it follows the Dittus-Boelter equation for turbulent flow.

MESHING OF GEOMETRY:

Structured meshing method done in ANSYS Workbench was used for meshing the geometry. Nodes were created. The 2D geometry of circular channel with structured mesh is shown in Fig. 4.



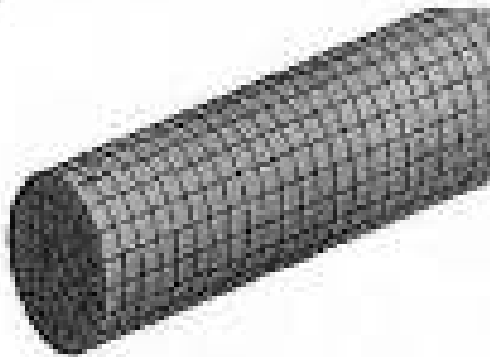


Fig. 4. Mesh model of pipe with coiled wire

Wind2) is applied on the channel wall. A uniform mass flow inlet and a constant inlet temperature were assigned at the channel inlet. At the exit, pressure was



MATERIAL PROPERTIES

Pure water is used as base working fluid and magnetic Fe₃O₄nanofluid is taken as nanoparticles. The density, heat capacity and thermal conductivity of Fe₃O₄are 5180kg/m³, 6700J/kgK, and 81.4W/mK, respectively. The properties of nanofluids (nf) are given in Table 4.2 at 30°C temperature and 100 kPa pressure.

Table 4.2 Water base fluid properties with different concentration of Fe₃O₄ nanoparticles

Volume fraction (%)	Density (kg/m ³)	Specific heat (kJ/kgK)	Thermal conductivity (W/mK)
0.02	999.93	4172.16	0.606933
0.1	999.88	4244.50	0.610182
0.3	1000.25	4176.45	0.611771
0.4	1000.850	4171.60	0.611978

BOUNDARY CONDITIONS: A no slip boundary condition was assigned for the exit porous wall

inlets, where bulk velocity components were set to zero at that boundary. A constant heat flux (11399

Boundary condition

METHOD OF SOLUTIONS

There are two ways to solve the problems stated above: (i) analytical method and (ii) CFD method. Lee and Mahvar, 2007 have used analytical method. The method consists of calculation of heat transfer coefficient (h) from either of Eq. 4.5 or Eq. 4.4 depending on whether flow is laminar or turbulent, calculation of bulk mean temperature of fluid using Eq. 4.12 followed by the calculation of wall temperature using eq. 4.13. The results given by Lee and Mahvar, 2007 are having errors in wall temperature and mean temperature calculation at the

study. The analytical values of heat transfer coefficient are calculated using Eq. 4.5 + Eq. 4.4. The heat transfer coefficient can also be obtained using Eq. 4.12. The CFD method follows the use of commercial software ANSYS-FLUENT 15.0 to solve the problem. The

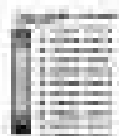
specified solver uses a pressure correction based iterative SIMPLE algorithm with 1st order upwind

scheme for discretizing the convective transport terms. The convergence criteria for all the dependent

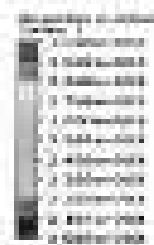


variables are specified as 0.05. The default values of under-relaxation factor as shown in Table 4.3 are used in the simulation work.

Results and discussion

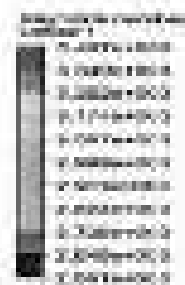


Pure water

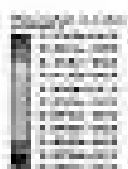


0.0% nanoparticle

Fig : Friction factor plot



0.02% nanoparticle



0.1% nanoparticle



0.2% nanoparticle

Pure water



0.0% nanoparticle

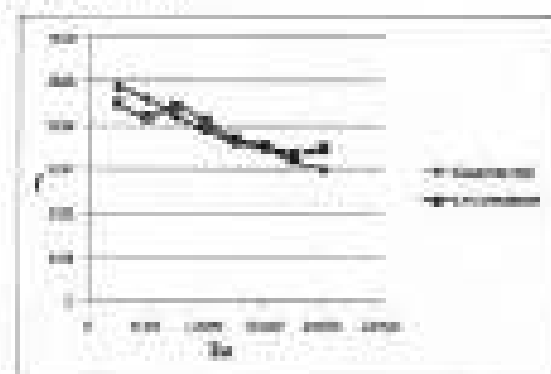


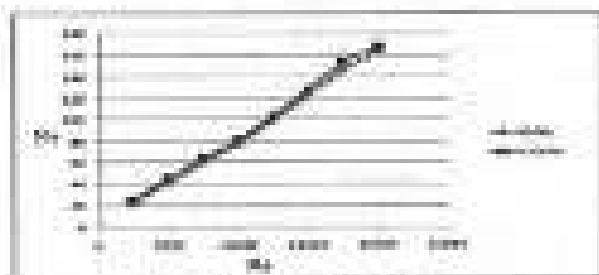
Fig : Reynolds num Vs Nusselt Number Graph

In the figure, it is seen that the local Nusselt number is larger for nanoparticles throughout the tube. This is



mainly due to the thermal dispersion in the flow. Thermal dispersion results in a higher effective thermal conductivity at the center of the tube which Nusselt number become higher when compared to the flow of pure water. Figure 1 above also shows that increasing particle volume fraction increases Nusselt number. This is due to the fact that the effect of thermal dispersion becomes more pronounced with increasing particle volume fraction.

It should be noted that the fully developed nanofluid Nusselt number values are also higher than pure water case. Associated values for different particle volume fractions of the Fe₃O₄/water nanofluid are presented in above figures. It is seen that increasing particle volume fraction increases the fully developed Nusselt number.



0.1% nanofluid

0.6% nanofluid

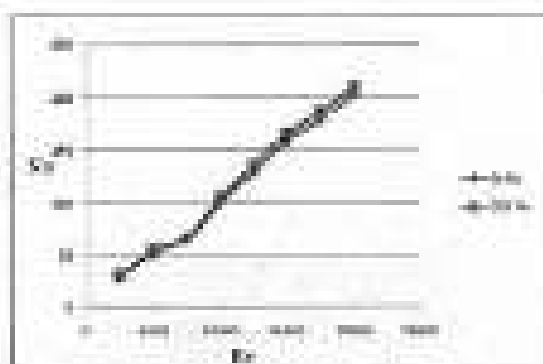


Fig 13: Reynolds Number Vs friction Factor

graph.

CONCLUSION

The heat transfer coefficient in the developed region of pipe flow containing Fe₃O₄ water nanofluid during the constant heat flux was simulated using CFD. The focal point of investigation was to evaluate the effect of particle volume concentration on convective heat transfer characteristics in the developed region of the tube flow containing water-Fe₃O₄ nanofluid. It was observed that 0.6% of nanofluid showed highest heat transfer characteristics than that of the base fluid (water).

- The computational results successfully validated the analytical data for circular pipe channel.
- Heat transfer coefficient is constant throughout the circular channel due to its fully developed condition.
- As the concentration of nano particle increases heat transfer coefficient also increases, with the increase in Nusselt number.
- Wall temperature increase within the flow direction of circular channel as very low Re simulation of Single Phase Fluid Flow in a Circular channel.

FUTURE SCOPE

- In the present work Fe₃O₄ is used with the base fluid, it can be assisted by using other some fluids such as Al₂O₃ or CuO.
- The analysis can be extended for flow through reentrant tubes and heat exchanger tubes instead considering single tube.



- Sedimentation aspects of near field should be studied.
- Analysis can be done for different volume fractions of near field which are not considered in the present work.

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ADVANCED FUEL OPERATED INTERNAL COMBUSTION ENGINES

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Abstract—Fossil fuels (i.e., petroleum, natural gas and coal), which meet most of the world's energy demand today, are being depleted rapidly. Also, their combustion products are causing global problems, such as the greenhouse effect, ozone layer depletion, acid rain and pollution, which are posing great danger for our environment, and eventually, for the total life on our planet. Many engineers and scientists agree that the solution to all of these global problems would be to replace the existing fossil fuel system with the clean hydrogen energy system.

Hydrogen is a very efficient and clean fuel. Its combustion will produce no greenhouse gases, no ozone layer depleting chemicals, and little or no acid rain ingredients and pollution. Hydrogen, produced from renewable energy (solar, wind, etc.) sources, would result in a permanent energy system which would never have to be changed.

However, at the present time, hydrogen is not competitive with other energy carriers. Global utilization of fossil fuels for energy needs is rapidly resulting in critical environmental problems throughout the world. There is an urgent need of implementing the hydrogen technology. A worldwide conversion from fossil fuels to hydrogen would eliminate many of the problems and their consequences. The production of hydrogen from non-polluting sources is the ideal way.

Solar hydrogen is a clean energy carrier. Hydrogen obtained from solar energy is ecologically

responsible along its entire energy conversion chain. Energy stored in hydrogen would be available at any time and at any place on Earth, regardless of when or where the solar irradiance, the hydrogenase, or other renewable sources such as biomass, ocean energy or wind energy was converted. Solar hydrogen is a clean energy carrier.

Solar hydrogen combines the advantages of hydrocarbons (storability and transportability) with the advantages of solar energy (ecological acceptability, renewability and low risk). Solar hydrogen has no need for the carbon atom, which makes for hydrocarbons almost infinitely storable at room temperatures, but is also the reason for their negative ecological impact.

Keywords—electrolysis of water, Ferrobibitysin, thermal decomposition of water and finally produced the H₂O gas for IC Engine.

INTRODUCTION

Fossil fuels possess very useful properties not shared by non-current total energy sources that have made them popular during the last century. Unfortunately, fossil fuels are not renewable. In addition, the pollutants emitted by fossil energy systems (e.g. CO, CO₂, CH₄, SO_x, NO_x, radioactivity, heavy metals, ashes, etc.) are greater and more damaging than those that might be produced by a renewable based hydrogen energy system (Winter et al. 1987).



Solar hydrogen is a clean energy carrier. Hydrogen obtained from solar energy is ecologically responsible along its entire energy conversion chain. Energy stored in hydrogen would be available at any time and at any place on Earth, regardless of when or where the solar irradiance, the hydropower, or other renewable sources such as biomass, ocean energy or wind energy was converted. Solar hydrogen is a clean energy carrier. It makes solar energy storable and transportable as oil and natural gas are by nature, but without the burden of their negative environmental impact. Solar hydrogen combines the advantages of hydrocarbons (stability and transportability) with the advantages of solar energy (ecological acceptability, renewability and low risk). Solar hydrogen has no need for the carbon stove, which makes the hydrocarbons almost infinitely storable at room temperatures, but is also the reason for their negative ecological impact.

Hydrogen has long been recognized as a fuel having some unique and highly desirable properties, for application as a fuel in engines. It is the only fuel that can be produced directly from the plentiful renewable resource water, though through the expenditure of relatively much energy. Its combustion in engines produces uniquely only water but in air it also produces some oxides of nitrogen. These features make hydrogen as an excellent fuel is potentially rare, the ever increasingly strict environmental controls of exhaust emissions from combustion devices, including the reduction of greenhouse gas emissions.

The use of hydrogen as an engine fuel has been attempted on very limited basis with varying degrees of success by numerous investigators over many decades, and much information about their findings is available in the open literature. However, these reported performance data do not display consistent agreement between various investigators. There is also a tendency to focus on results obtained in specific engines and over narrowly changed operating conditions. Moreover, the increasingly greater emphasis being placed on the nature of emissions and efficiency considerations often makes much of the very early work fragmentary and mainly of historical value. Obviously, there is a need to be aware of what has been achieved in this field while focusing both on the attractive features, as well as the potential limitations and associated drawbacks that need to be overcome for hydrogen to become a widely

accepted and used fuel for engine applications. Also, there is a need to indicate practical steps for operating and design measures to be developed and incorporated for hydrogen to achieve its full potential as an attractive and superior engine fuel.

LITERATURE OF SURVEY

1.1 Use of H_2 as IC engine fuel

In the early years of the development of internal combustion engines hydrogen was not the "exotic" fuel that it is today. Water splitting by electrolysis was a well-known laboratory phenomenon. Otto, in the early 1870s, considered a variety of fuels for his internal combustion engine, including hydrogen. He rejected gasoline as being too dangerous. Later developments in combustion technology made gasoline viable.

Most early engine experiments were designed for burning a variety of gases, including natural gas and propane. When hydrogen was used in these engines it would backfire. Since hydrogen burns faster than other fuels, the fuel-air mixture would ignite in the intake manifold before the intake valve could close. Injected water controlled the backfiring. Hydrogen gave less power than gasoline with or without the water.

1.2 H_2O Gas

The H_2O gas is nothing but the electrolytic form of water. It is also called as oxyhydrogen or brown gas. It is produced by electrolysis process, where an electrical power source is connected to two electrodes and which are placed in a mixture of water and electrolyte. Oxyhydrogen appears to be a favorable alternative fuel on account of its high specific energy per unit weight, its all-time availability as a component of water, good combustion characteristics and eco-friendly, fast burning and higher flame propagation rates are the attractive features of H_2O gas. H_2O gas is a mixture of hydrogen and oxygen gases, typically in a 2:1 molar ratio, the same proportion as water. At normal temperature and pressure, oxyhydrogen can burn when it is between about 4% and 94%



hydrogen by volume, with a flame temperature around 2000°C. Oxyhydrogen will combust (burning into water vapour and releasing energy which sustains the reaction) when brought to its auto-ignition temperature. For a stoichiometric mixture at normal atmospheric pressure, this is about 570°C(1058°F). The minimum energy required to ignite such a mixture with a spark is about 0.02J. The quantity of heat evolved, according to Julius Thomson, is 34,110 calories for each gram of hydrogen burned. This heat-disturbance is quite independent of the mode in which the process is conducted, but the temperature of the flame is dependent on the circumstances under which the process takes place. It obviously attains its maximum in the case of the firing of pure "oxyhydrogen" gas (a mixture of hydrogen with exactly half its volume of oxygen, the quantity it combines with in becoming water, German Kaelde-gas). It becomes less when the "oxyhydrogen" is mixed with excess of one or the other of the two reacting gases, or an inert gas such as nitrogen, because in any such case the same amount of heat spreads over a larger quantity of matter.

1.3 Performance characteristics of petrol and HHO₂ gas when used as a fuel

The performance of petrol engine with HHO up to 30% of full load, brake thermal efficiency will remain same for both the case (petrol and HHO). Beyond this point, brake thermal efficiency increase linearly for fuel mixture of petrol and HHO gas. And also up to 30% of full load for both cases the brake thermal fuel consumption increase linearly and reaches maximum value. BMEP decreases more rapidly for fuel mixture of petrol and HHO gas than petrol as a fuel.

If comparing the results of above two cases, the mixture of petrol and HHO gas gives better result for higher loads. Since HHO is highly combustible fuel, it causes the complete combustion of the fuel mixture. Thus it results in the increased speed of the engine and in turn increases the power output.

1.4 PROPERTIES OF HHO GAS

There are many unique and unusual properties that HHO Gas possesses. Below is a list of some of the properties: Gas proves to be colorless, colorless and lighter than air. In the production of HHO Gas, there is no evaporation process at all, the electric energy used being insufficient for evaporation.

The variable character of the energy content of HHO Gas is evidence that the gas has a unique structure with a chemical composition including bonds beyond those of atomic type.

HHO Gas does not follow the fundamental PVT Law for gases.

- HHO Gas demonstrates an anomalous adherence to gases, liquids and solids. HHO Gas bonds to gaseous fuels (such as natural gas, kerosene fuel, and others) and also to liquid fuels (such as diesel, gasoline, liquid petroleum, and others).
- Sarrilh describes the creation of the gaseous and combustible HHO from distilled water at atmospheric temperature and pressure via a process structurally different than evaporation or separation, which suggests the existence of a new form of water.

HHO is described to have the structure H-O-H where represents the new trigonal bond and the conventional molecular bond. The transition from the conventional H-O-H configuration to the new H-O-E species is explained as being a change of the electric polarization of water caused by the electrolysis.

1.5 Design consideration for kit

The following are the design consideration for a HHO generating kit

- Compact in size
- Suitable for long operation
- Energy efficient
- Safe
- Easy to operate and maintenance free
- Leak proof
- Affordable Price

1.6 BASIC DETAILS OF KIT

The HHO generator (kit) is basically an electrolytic cell. Here the kit used is of simple in construction, whose case (container) is made of PVC



pipe and the electrodes are made from copper plate of length 500mm and thickness 1mm. The 500mm length copper plate is cut into pieces such that two equal length plate are of 250mm respectively. The lengths of both plates are of 250mm respectively and the plates are separated from each other by means of an insulator. Now these arrangements are placed inside a case (container) of diameter 150mm and it's both sides are sealed with end caps (PVC). The upper end cap houses the electrode terminals and also a hole for HHO gas outflow. The lower end cap has a hole for take off electrolyte solution. The electrode connections are made in such a way that the small part of the electrode is projected outwards so the melting of the connection between the electrode and the connection is eliminated.

In the electrolysis process the oxygen is generated at anode & hydrogen is generated at cathode. A small free space at the top of the cylinder will allow the gases to mix together. The HHO gas generated from the kit is supplied to the engine. A hole of 3mm is drilled at the engine manifold to supply the gas. Since HHO gas is highly inflammable, the gas is passed through the water. The container of water acts as a Back Fire Arrestor. This is for safety reasons. Thus the gas is passed through the capacitor to run the engine.



Figure 5.1 Supply of HHO gas to the engine

Design calculation

Some of the important design calculations are as mentioned below,

2.1 Length and diameter of the coils

The length and diameter of the coils depend on the production capacity of kit. We have decided to design this system for 12 Amps current. Following are the design calculations for determining the length and diameter of coils.

Design current = 12 Amps

Design voltage = 15.5volts

However in our design, the length and diameter of the copper wire are 2500mm and 1mm respectively. The length of coils (i.e. both electrodes) is 300mm. The dimensions are selected such that to get the optimum results based on both power consumption as well as HHO gas liberation.

2.2 Features of kit

Some of the features of kit are as given below,

2.2.1 Water temperature

The output voltage of battery or an alternator of a vehicle depends on the engine speed (i.e. between 12 volts and 13.8 volts). If the amount of power supply to the kit is car battery, the voltage and the current to the kit will always be fluctuating due to the varying engine speed. The amount of gas produced in the kit depends on the voltage applied to the system (i.e. at 12 volt the Hydroxyl production will be low however at 13.8 volts the Hydroxyl production will be at peak). Similarly when the water is cold the voltage required to break the water molecules is around 2 volts. The voltage required to break the water molecules decreases as the water temperature of water increases, as this reduces the resistance. In the kit (i.e. reduction in the voltage drop due to lesser of resistance). At this point of time there is an additional voltage available at out of electrodes, which further heats up the water.

2.2.2 Efficiency of kit

As per the Faraday's law thumb rule around 3.24 watts energy is required to produce 1 ltr of gas per hour. The kit is producing 1 ltr at 15.5 volts and 12 amps (i.e. 186 watts).

As per our experiment the most efficient water temperature to operate a kit is between 40-55 deg centigrade. At this stage the production of the Hydroxyl is at peak.

2.2.4 NaOH Concentration

In most of the coils, it is required to increase the concentration of the solution as time to time basis. However in this kit, a mesh is fixed just above the coils to hold the KOH fumes coming out with the Hydroxyl

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produced in the system. Cost of KOH is also high (nearly ten times of NaOH).

Due to these problems we started to use NaOH solution. This NaOH produces fewer fumes. So there is no problem of reduction in concentration.

2.1.5 Water level indicator

The water level indicator is made using transparent 10mm diameter tube. These components are joined at leak proof joint on the outer case. This indicator also provides the provision for water topping up.

2.1.6 Fire arrester

HHO gas is a highly combustible fuel. In order to protect the kit from the back fire produced from the engine the fire arrester are used. It is a simple method in which, instead of directly supplying HHO gas to the engine it is made to flow through a container containing water. Such that even though the gas catches fire it won't reach the kit.

PRODUCTION OF HYDROGEN

Hydrogen is not a fluid that occurs free in nature. The fossil fuel. Primary source of energy like solar, nuclear or hydro-electric is necessary to separate it from original combined state. The following methods are considered suitable for hydrogen production:

3.1 Electrolysis of water

In this method, electrical energy is used to break water into H_2 and O_2 . In principle, an electrolytic cell consists of two electrodes, commonly flat metal or carbon plates, immersed in an aqueous conducting solution called the electrolyte.

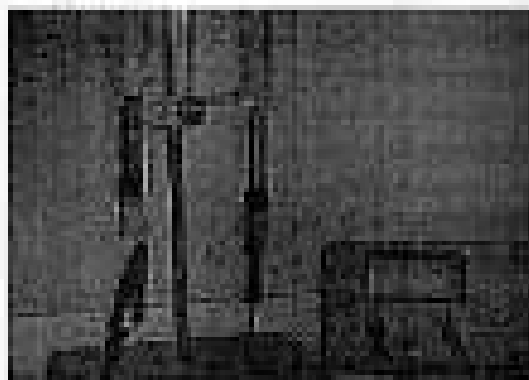


Figure 4.1. Electrolytic Process

A source of DC voltage connected to the electrodes so that an electric current flows through the electrolyte from anode to cathode. As a result, water in the electrolyte solution is decomposed into H_2 which is released at the cathode and oxygen at the anode. Since water itself is the poor conductor of electricity an electrolyte like KOH is used to increase the electric conduction.

3.2 Thermochemical method

This method is considered potentially most promising. It depends on complex series of interactions between the primary energy, water and some specific chemicals to produce hydrogen at temperatures substantially lower than thermal decomposition. The chemicals used are recyclable. A variety of compounds of iron, iodine, sulfur and carbon are used for the purpose.

3.3 Photohydrolysis

In this process, action of certain catalyst to produce H_2 from water by use of direct sunlight at ambient temperature. Though, it appears attractive, the present efficiency of production is only 1%.

3.4 Thermal decomposition of water

In this method, heat at high temperature ($2000^{\circ}C$) is used to thermally decompose water into H_2 and O_2 .

3.5 Advantages of HHO gas as a fuel in Diesel engine

- HHO gas mixture have nearly 12 times lesser compared to gasoline air mixture.
- HHO ignition limits are much wider than gasoline's. So it can burn easily and give considerably higher efficiency.
- High self-ignition temperature but very little energy is required to ignite it.
- Its clean exhaust is the most attractive feature of all.
- No greenhouse effect.

3.6 Disadvantages of HHO gas as a fuel in Diesel engine

- Produces toxic emission of NO_x .



- One of the major practical difficulties using BHK as car fuel is its very low density either in gas or liquid form.
- The handling of BHK gas is more difficult and storage requires high capital and running cost.
- Weight of hydrogen H_2 : 1kg
- Coefficient of discharge: 0.62
- Specific gravity of diesel: 0.8275
- Calorific value of diesel: 45357KJ/Kg K
- Specific heat of exhaust gases: 1.005KJ/Kg K
- Specific heat of cooling water: 4.187KJ/Kg K

3.7 Engine

The engine used in this project is 4-stroke single cylinder diesel engine (Kirloskar)

Engine specifications

- Type: 4-stroke water cooled.
- Cylinder: single
- BHP: 3HP
- RPM: 1500RPM
- Fuel: diesel
- Bore: 80mm
- Stroke length: 110mm
- Crank diameter: 32.2mm
- Compression ratio: 16:1
- Brake drum diameter: 0.12m

3.8 Dynamometer (mechanical type)

It is a simple device used for measuring brake power of an engine. It consists of a flat belt of 5mm thickness and 100mm width, wound around the rotating drum of 50mm of diameter, attached to the shaft of 30mm diameter, which is connected to the engine out put through chain drive. The both ends of the flat belt is

DESIGN CALCULATION

4.1 Brake dimensions

Brake Drum Diameter = 50mm

Radius R=25mm

BHP= (2 π NT)/4500

DESCRIPTION OF TEST RIG

connected to the spring gauge, it consists of 0 to 100 kg of graduations. The load is applied on the drum by using nut and bolt arrangement.

3.9 Manometer (U-tube)

It consists of a U tube filled partially with water. One end of the tube is connected to air drum to measure inside the drum and another led to measure atmospheric pressure.

3.10 Barometer

It is a glass tube with graduation. This is used to measure the fuel consumption by the engine. It is connected in between fuel tank and injector.

3.11 Digital RPM indicator

It consists of a digital display with a sensor, which measures the revolution of the output shaft.

3.12 Air drum

It is a cylindrical drum of 500mm length and 300mm diameter, with two passages. One end is connected to the engine through intake manifold other end is free to absorb atmospheric air. The intake air from atmosphere is purified using filter present in the intake passage.

3.13 C-frame

The base of the test rig is made of C-section steel frames. The width of the frame is 80mm and the height is 50mm. Thickness of frame is 5mm.

Where N= speed in rpm

$$T = W_{max} R$$

Where T=torque

W_{max} = maximum load carrying capacity of engine

R= brake drum radius

4.1.1 To find Maximum Load



$$W_{max} = 4500 \text{ BHP} (2\pi \cdot N \cdot R)$$

$$= (4500 \cdot 5) (2\pi \cdot 1500 \cdot 0.16)$$

$$= 14.93 \text{ Kg}$$

∴ Maximum Load for the engine, $W_{max} = 14.93 \text{ Kg}$

4.2 CALCULATIONS

Referring to table 6.1

For load, $W = 1.3 \text{ kg}$

$$\text{Brake Power, BP} = \frac{(2\pi \cdot W \cdot R \cdot 9.81)}{60000} \cdot N$$

Where,

N is speed in RPM

W is load in kg

R is radius of brake drum in meters

$$\text{BP} = \frac{(2\pi \cdot 1.3 \cdot 0.16 \cdot 9.81)}{60000}$$

$$= 3.2681 \text{ kW}$$

Total Fuel consumption, $\text{TFC} = \frac{10^6 \cdot \text{Specific gravity} \cdot W}{1000 \cdot T}$

Where,

Specific gravity of Diesel = 0.8275

T is time for 1000g of fuel consumption in seconds

$$\text{TFC} = \frac{10^6 \cdot 0.8275 \cdot 1000}{1000 \cdot 25}$$

$$= 1.1916 \text{ kg/hr}$$

Friction power, $FP = 4.7 \text{ kW}$

Indicated power, $IP = FP + BP$

$$= 4.7 + 3.2681 = 7.968$$

Brake specific fuel consumption, $\text{BSFC} = \frac{\text{TFC}}{\text{kg/Whr}}$

$$\text{BP} = \frac{1.1916}{1.28565} = 0.9271 \text{ kg/Whr}$$

Brake mean effective pressure, $\text{BMEP} = \frac{\text{BP} \cdot 60}{L \cdot A \cdot (N/2)}$

Where,

L is the stroke length = 90mm = 0.09

A

A is the piston area $A = \pi r^2$

$$= 0.0042 \text{ m}^2$$

$$\text{BMEP} = \frac{3.2681 \cdot 60}{0.08 \cdot 0.0042 \cdot (1500/2)} = 0.28985 \text{ bar}$$

Brake thermal efficiency, $\eta_{br} = \frac{\text{BP} \cdot 3600}{\text{TFC} \cdot \text{CV}} \cdot 100 \%$

$\text{TFC} \cdot \text{CV}$ Where,

CV is the caloric value of fuel = 45357 kJ/kg

$$\eta_{br} = \frac{3.2681 \cdot 3600}{1.1916 \cdot 45357}$$

$$= 21.3 \%$$

Mechanical efficiency, $\eta_{me} = \frac{\text{BP}}{\text{IP}} \cdot 100 \%$ Where, BP is the brake power

IP is the indicated power

$$= \frac{3.2681}{7.9681} \cdot 100 = 42.78 \%$$

Indicated thermal efficiency, $\eta_{id} = \frac{\text{IP} \cdot 3600}{\text{TFC} \cdot \text{CV}}$

$$= \frac{7.9681 \cdot 3600}{1.1916 \cdot 45357} = 49.98 \%$$

CONCLUSION

5.1 Optimum method of HHO generation

By comparing the reading from table 6.1 and table 6.2 the discharge of HHO gas is 14.4 LPH for KOH and NaOH, which is almost same for both the cases. KOH solution produces lot of foam which leads to loss of concentration of electrolyte and it also decrease the generation of HHO gas. Therefore comparing the production, rate of change of electrolyte concentration and cost of KOH and NaOH, NaOH electrolyte solution is best suited for the generation of HHO gas.

5.2 Performance characteristics of Diesel and HHO gas when used as a fuel

From graph 1 up to 50% of full load, brake thermal efficiency will remain same for both the case. Beyond this point brake thermal efficiency increase linearly for fuel mixture of diesel and HHO gas. From graph 2 up to 50% of full load for both cases the total



Fuel consumption increases linearly and reaches a maximum value. TFC decreases more rapidly for fuel mixture of Diesel and HHO gas than diesel as a fuel.

Result

By comparing the results of above three graphs we can conclude that The mixture of diesel and HHO gas gives better result for higher loads. Since HHO is highly combustible fuel, it ensures the complete combustion of the fuel mixture. Thus it results in the increased speed of the engine and in turn increases the power output.

Future scope

It was found that the HHO engine gives as much power and at the same time cost also less compared to normal diesel operated engine. However, such results should be interpreted with caution and the finding should be supported by a larger body of statistical data. In future efforts, it is helpful for researching of HHO gas operated IC Engines up to 100% than 50% efficiency results.

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Performance test on single cylinder four stroke diesel engines by using convergent divergent inlet manifold

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ABSTRACT

Creating thrust on reduction of Internal Combustion Engine fuel consumption with increase of its performance new designs and optimization of existing ones are introduced. Air motion in CI Engine influences the atomization and distribution of fuel injected in the air charge. Better atomization of injected fuel allows for a more complete burn and helps to reduce the engine knock. A four stroke compression Ignition engine with power 5 HP and rated speed 1400,1500,1600 rpm is selected for the present work to investigate the performance test. The inlet motion of the air is an important parameter to optimizing the performance of the engine in order to increase the air velocity in the inlet manifold a convergent-divergent nozzle is used. The rise is related with the use of nozzle geometry techniques at the exit of the manifold which facilitates for better combustion of injected fuel. The Performance test is calculated without noise and without noise in the inlet manifold and compared.

Keywords: Inlet manifold, Diesel engine, Performance Characteristics, Speed, Velocity

1. INTRODUCTION

Thermal energy (heat) is one of the oldest forms of energy known to mankind. Thermal energy is usually derived from sources such as chemical energy and electrical energy. The device for converting one form of energy to another is termed as engine. In an energy conversion process, the conversion efficiency plays a vital role and it determines the efficient use of the supplied energy. Heat engine is the device that can transform chemical energy of a fuel into thermal energy and utilize this thermal energy to perform useful work. Here, in this work we have implemented the helical threaded manifold by varying pitch, for generating the swirl while starting the cylinder. The objective was achieved in the inlet manifold by housing the inlet manifold of size 4mm width and 1mm depth with different pitches to direct the air flow. The tests are carried with different configurations by varying the pitch of the helical

inlets from 10mm to 25 mm in steps of 5mm inside the intake manifold. The measurements were done at constant speed of 1400,1500,1600 rpm. The results are compared among normal manifold.

1.1. Analysis of gas flow in De Laval nozzle.

The analysis of gas flow through de Laval nozzle involves a number of concepts and assumptions:

1. For simplicity, the gas is assumed to be an ideal gas. The gas flow is isentropic (i.e., at constant entropy). As a result the flow is reversible (frictionless and no dissipative losses), and adiabatic (i.e., there is no heat gained or lost).
2. The gas flow is constant (i.e., steady) during the period of the propulsion burn.
3. The gas flow is along a straight line from gas inlet to exhaust gas exit (i.e., along the nozzle's axis of symmetry).
4. The gas flow behavior is compressible since the flow is at very high velocities (Mach number > 0.3).

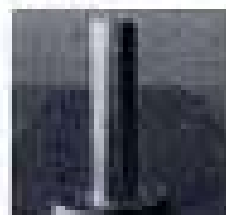


Fig.1 Convergent-Divergent manifold



Fig.2 Photograph of Convergent-Divergent manifold



Fig.3 Preparation of Convergent-Divergent nozzle/flange

1.1 BASIC ENGINE TERMINOLOGY:

A number of basic terms are used to describe and compare engines. A few commonly used terms are explained here.

1) Top dead centre (T.D.C):

This refers to the position of the crank shaft when the piston is in its top most position i.e., the position closest to the cylinder head (position 1 in Fig).

2) Bottom Dead Centre (B.D.C):

This refers to the position of the crankshaft when the piston is in its lowest position, i.e., the position furthest from the cylinder head (position 2 in Fig).

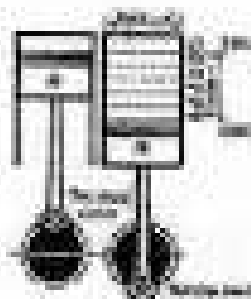


Fig.4 Piston and Crankshaft

3) Bore: Diameter of the engine cylinder is referred to as the Bore.

4) Stroke: Distance traveled by the piston in one cycle— the value of 'r' varies during the power stroke the from T.D.C to the B.D.C. In the power stroke is constantly varying.
Volume: The volume of cylinder (including clearance volume there is no torque delivered during the first part of its duration) above the piston when it is in the top stroke.

T.D.C position (volume between 1 and 2 in Fig) referred to as clearance volume:

5) Piston displacement: This is the volume swept by the piston in moving from T.D.C to B.D.C. This is also called 'swept volume'. If 'd' is the cylinder bore and 's' the stroke, the piston displacement, V_d , is given by

$$V_d = \frac{\pi}{4} \times d^2 \times s$$

6) Engine Capacity: This is a total piston displacement or the swept volume of all the cylinders. If 'n' is the number of

cylinder and V_d is the piston displacement, then 'Engine displacement' or engine capacity V_d is given by

7) Compression ratio: This indicates the extent to which the charge in the engine is compressed. This is calculated as the ratio of the volume above the piston at B.D.C to the volume above the piston at T.D.C. If 'r' is the compression ratio, then

$$r = \frac{V_{at} + V_c}{V_c}$$

For petrol engines, compression ratios are about 8 to 10, whereas for diesel engines, these vary from 15 to 24.

8) Mean effective pressure: This is the average effective pressure throughout the whole power stroke. In fact the cylinder pressure varies considerably during the power stroke. Thus it is more helpful to refer to the mean pressure instead. It is expressed in bars or kPa (Pascals) (1 bar=100kPa).

9) Power: It is the work done in a given period of time. During the same amount of work in a longer time would supply more power.

10) Indicated power (I.P.): The power developed within the engine cylinder is called indicated power. This is calculated from the area of the engine indicator diagram. It is usually expressed in kilowatts (K.W).

11) Brake Power (B.P.): This is the actual power delivered at the crank shaft. It

obtained by deducting various power losses in the engine from the indicated power. It is measured with a dynamometer and is expressed in kilowatts. It is always less than the indicated power, due to frictional and pumping losses in the cylinder and the reciprocating mechanism.

12) Engine Torque: It is the force of rotation acting about the crank shaft axis at any given instant of time.

It is given by

$$T = F \times R$$

- T - Engine torque (Nm)
- F - Force Applied to crank (N)
- R - Effective crank radius (m)

Therefore, the engine manufacturers always quote the average value of torque throughout the engine cycle. Engine torque goes through the vehicle transmission system to the rear wheel and is responsible for rotation of the latter and hence for pulling of the vehicle.

2. EXPERIMENTAL SETUP & PROCEDURE

2.1 Introduction: The details of the experimental set-up are presented in this chapter. The alterations made to the instrumentation are also described. The experimental setup is designed to carry the objective of the present work. The various

$$V_d = V_d \times n$$



components of the experimental set up including modification are presented in this chapter.

2.2 Experimental set up: The experimental set up consists of engine, an alternator, top load system, fuel tank along with transmission loader, exhaust gas measuring digital device and manometer.

Engine:

The engine which is supplied by M/s Kirloskar Company. The engine is single cylinder vertical type, four stroke, air-cooled, compression ignition engine. The engine is self governed type whose specifications are given in Appendix 1. It is used in the present work.

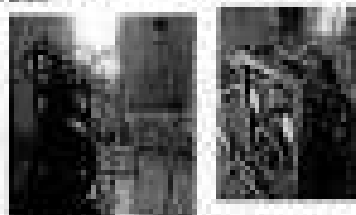


Fig. 2. Diesel engine. Plate look view and ICD manifold

Fuel injector (BOSCH)

A most standard view of a typical BOSCH fuel injector:

The injector assembly consists of

- i. A needle valve
- ii. A compression spring
- iii. A needle
- iv. An injector body

U-tube manometer

The one of end of the U-tube manometer is connected to the orifice of the air tank and the other end is exposed to the atmosphere, the manometer liquid used is water.

Digital thermometer

It consists of a temperature sensing element connected to the electronic digital display which is operated by battery.

2.3 VARIOUS PARTS OF EXPERIMENTAL SET UP

1. KIRLOSKAR ENGINE
2. ALTERNATOR
3. DIESEL TANK
4. AIR FILTER
5. THREE WAY VALVE
6. EXHAUST PIPE
7. PROBE
8. EXHAUST GAS ANALYSER
9. ALTERNATIVE FUEL TANK
10. BURBITE
11. THREE WAY VALVE
12. CONTROL PANEL

2.5 Instructions

1. Measurement of Brake power

The power developed by the engine is measured by using electrical dynamometer. The pump is run by using the power developed by the engine. The total power is obtained by adding pump power to the product of voltage and current.

1. Measurement of fuel:

The fuel flow is measured by volume through a burette tube which is fixed between fuel tank and fuel pump. A T-joint prepared and one side of it is connected to the fuel measuring tube. The remaining two sides of the joints are connected to the fuel tank and the fuel pump respectively. Fuel flow is measured by noting the time taken for 10cc of fuel consumption by stop watch.

2. Measurement of air flow:

Air flow is measured by using a Rotameter flow air meter. A paper element filter is an integral part of the meter. The meter consists of an orifice, the pressure drop across the orifice is measured by manometer, it is ensured that there are no leakages in the connecting tubing.

4. Load on engine:

Initially fuel tank and auxiliary fuel tanks are filled with right amount of required fuel. The instruments such as PMG meter and CO/HC analyser are connected to the exhaust pipe. The engine is started and allowed to run for 20 minutes to attain steady state condition.

5. Measurement of exhaust gas:

Engine emissions are measured by using exhaust gas analyser.

6. Measurement of exhaust gas temperature:

The temperature of exhaust gas is measured by using digital electronic device. It gives the exhaust gas temperature directly.

2.6 Experimental Procedure

Before starting the engine, the fuel injector is separated from the fuel system, it is cleaned on the fuel injector pressure tested and sprayed the tester pump. Observe the pressure reading from the dial. At which the injector starts spraying, in order to achieve the required pressure by adjusting the screw provided at the top of the injector.

This procedure is repeated for obtaining the various required pressures.

As first trial, diesel engine is allowed to run the engine for about 30 min, so that it gets warmed up and steady running conditions are attained. Before starting the engine, the lubricating oil level in the engine is checked and it is also ensured that all moving and rotating parts are lubricated.

The various steps involved in the setting of the experiment are explained below:

1. The Experiments were carried out after installation of the engine.
2. The injection pressure is set at 300 bar for the entire test.
3. Precautions were taken, before starting the



experiment:

4. Always the engine was started with no load condition.
5. The engine was started at no load condition and allowed to work for at least 10 minutes to stabilize.
6. The readings such as fuel consumption, spring balance reading, cooling water flow rate, maximum reading etc., were taken as per the observation table.
7. The load on the engine was increased by 10% of FIE.L. Load using the engine stands and the readings were taken as shown in the table.
8. Step 7 was repeated for different loads from no load to full load.
9. After completion of test, the load on the engine was completely relieved and then the engine was stopped.
10. The results were categorized as follows.

The above experiment is repeated for various loads on the engine. The experimental procedure is similar as formal. While starting the engine, the fuel tank is filled to required fuel proportion up to its capacity. The engine is allowed to run for 20 min, for steady state conditions, before load is performed.

We have not been observed any separation in the blend of 50% Fish oil & 50% diesel. We have heated the blend about 70°C in order to reduce the viscosity for easy flowing in the tubes, better injection and atomization in the cylinder the viscosity is nearly equal to the viscosity of diesel.

Finally, the engine is run by blend (50:50) at various

3. TABLES AND GRAPHS

Table 3 Performance test Results on normal fuel
Fig.8 Load against Mechanical efficiency



Fig. 9 Load against Brake Thermal Efficiency



5. RESULTS AND DISCUSSION

Injection pressure the corresponding observations are noted.

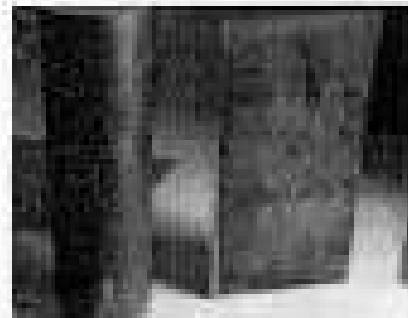
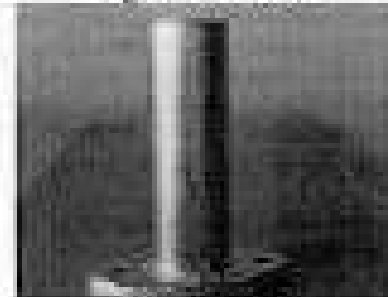


Fig.7. Raw material



Prepared material

material at 1400rpm:

5.1 Brake specific fuel consumption:

The graph for the variations in the brake specific fuel consumption (BSFC) is presented in the Fig.4.3. For all the fuels the BSFC falls with increasing load. The differences of BSFC are very small when using different speeds. The maximum BSFC values are 1.51 kg/kWh for diesel, 0.77 kg/kWh for M100, 0.975 kg/kWh for the M140, 1.025 kg/kWh for the M160, 1.113kg/kWh for and for C10:50, 1.095 kg/kWh for. The higher

5.2 Brake thermal efficiency:

The variation of brake thermal efficiency with load is shown in Fig. 4.3 Brake thermal efficiency gives an idea of the output generated by the engine with respect to heat supplied in the form of fuel. For all the fuels the brake thermal efficiency increases with load. The brake thermal efficiency values at full load are 12.5% for diesel, 16.42%.

5.3 Volumetric efficiency:

The Fig.4.14 shows the variation of volumetric efficiency with load for various speeds. Volumetric efficiency is a measure of success with which the air supply, and thus the charge, is inducted in to the engine.



It increases the breathing capacity of the engine. From the figure 8 it is evident that the volumetric efficiency values of N1500 and CD1500 are exceeding the volumetric efficiency values of diesel at all loads.

5.4 Brake mean Effective Pressure:

The variation of Brake Mean effective pressure with load. Mean effective Pressure is the average Pressure inside the cylinders of an internal combustion engine based on the measured output. From the figure it can be seen that, Brake mean effective values of neem oil and its diesel blends are slightly less than diesel.

5.5 Indicated Mean Effective Pressure:

Figure 6 shows the variation of Indicated Mean effective pressure with load. Mean effective Pressure is the average Pressure inside the cylinders of an internal combustion engine based on the measured output. From the figure it can be seen that, Brake mean effective values of neem oil and its diesel blends of N1500 and CD1500 are slightly low.

5.6 Exhaust gas Temperature:

Figure 6 shows the variation of exhaust gas temperature with load for various test fuels. It is observed that the exhaust gas temperature increases with load because more fuel is burnt at higher loads to meet the power requirement. It is also observed that the exhaust temperature increases for N1500 and CD1500 neat at all loads. This may be due to the oxygen content of the neem oil, which improves combustion and that may increase the exhaust gas temperature.

5.7 Mechanical Efficiency:

Mechanical efficiency indicates how good an engine is inverting the indicated power to useful power. Figure 7 shows the mechanical efficiency for N1500 is less than CD1500 for pure diesel. Because higher fuel injection pressure causes the decrease of atomization. The fitness of atomization reduces again later fig.

CONCLUSION

The Performance characteristics of an engine without nozzle and with nozzle in the intake manifold were compared in the present work. A Convergent Divergent Nozzle of different speeds N1400,N1500,N1600, CD1400,CD1500 &CD1600rpm.

- 1) Brake power is increased by 2.58%.
- 2) Test fuel consumption is reduced by 2.57%.
- 3) BS (2006) 1714-1718.

- 4) Specific fuel consumption is reduced by 5.50%
- 4) Indicated power is increased by 4.27%
- 5) Mechanical efficiency is reduced by 1.84%
- 6) Heat input is reduced by 2.59%
- 7) Brake thermal efficiency is increased by 3.44%
- 8) Indicated thermal efficiency is increased by 7.18%
- 9) Volumetric efficiency is reduced by 1.33%
- 10) Brake mean effective pressure is increased by 0.18%.

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DESIGN AND THERMAL ANALYSIS OF WARM (COLD) FORGING PROCESS AND DIES

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Abstract— Forging temperature is one of the basic considerations in forging process. In warm forging, the metals are forged at temperatures about the recrystallization temperature and below the traditional hot forging temperature. Warm forging has many advantages when compared when compared to cold forging. In this study, forging process of a part which is currently produced at the hot forging. Temperature range and which work metal temperature range is analyzed. Material usage and energy concept, is analyzed. The proposed process is simulated using Finite Element Method and Ansys Experimental study is also carried out in IITM Center Forging Research and Application Laboratory and it has been observed that experimental result are in good agreement.

Waste is reduced by proposing using of energy saving and better accuracy in dimensions is achieved by reducing the forging temperature from the hot forging to the warm forging temperature range.

Keywords - Warm Forging, Finite Element Analysis, Die Stress Analysis, Metal Forming, Cold Chamber Forging.

INTRODUCTION

The term "forging" is applied to processes in which a piece of metal is worked in a machine in the desired shape; plastic deformation of the working stock. The energy that produces deformation is applied by a hammer, press, or other of the metal, direct stress or its combination. A forging is

produced in three distinct phases: stock preparation in the form of blooms, billets, bar or ingots; plastic deformation of the metal component or shape; stock reduction of hot shape in one of the forging processes; and appropriate secondary operations.

In this study, forging process of a part which is currently produced at the hot forging temperature range and which needs some improvements in accuracy, surface rough and energy concept, is analyzed. The forging process and new design with a die is proposed as per the given value part is proposed to warm forging temperature range and the proposed process is simulated using Finite Element Method. In the simulation, coupled thermal-mechanical analysis are performed and the dies are modeled as deformable bodies to derive die stress analysis.

OBJECTIVES OF PROJECT

- A forging die designed and simulated as per the terms of contract and the design.
- Process will be simulated.
- Thermal properties of die material as well as FEM simulation are observed.
- Pre-thermal heat of new materials in the process and compare with the old die for CHD.
- First to verify while the forging, the component that component to die material will be simulated with the new die material.
- The finite element analysis and pre forging heat simulation analysis is carried out for the process is production of the component of die cast.





- The material behavior will be observed precisely
 for better prediction compared to theoretical analysis.

LITERATURE SURVEY

Chen et al. has being process was simulated coupled with thermal analysis in order to determine effects of the process on the behavior of the fiber polymer stages in some being process. The different cases the speed distribution of the fiber has been presented. The stress and strain distribution was also used for the design analysis of the case.

Kim, Yung, Yatsunami et al., a history of material use of FE simulation in fiber; was was briefly reviewed. The pros and cons and benefits of FE simulation in being was was discussed with examples. Finally, key points for successful and effective use of FE simulation were explained followed by current issues for further use of the FE simulation at present and in the being industry.

THE HAYNERS

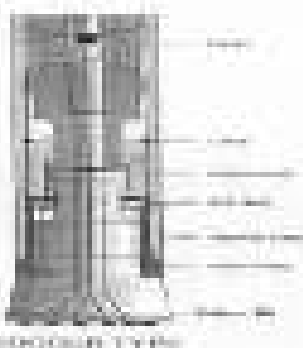
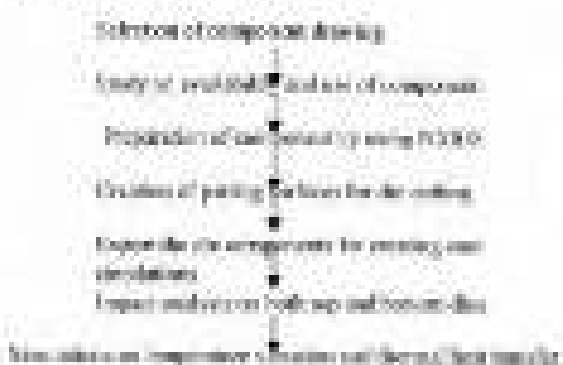


Fig1 The hayn were used in R & D ring and were available

1. METHODS FOR DESIGNING AND ANALYSIS TO DEVELOP THE WORK

Design and development procedure



1.1 DESIGNING PROCEDURE

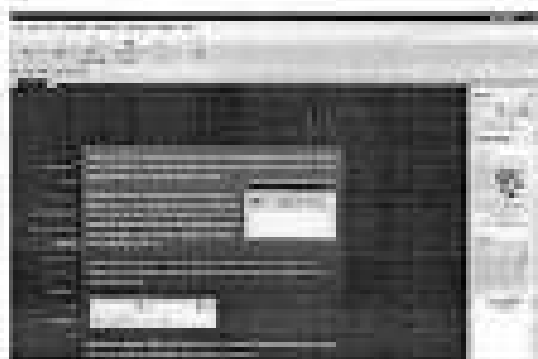


Fig 1.11 Showing 3D

Features and tools



Fig 1.12 Shows the Simulation tool for design

1.2 Computer 3D drawing with MSC A design



Fig 1.24 Shows the simulation tool in MSC A design of casting part for the design



Fig 1.24 Shows the feature of the design which is used as pattern for the experimental analysis. Material can be added in the pattern domain for the printing for casting trials.



Fig.12.3 Shows the design of root fillet forms die with mesh. It is the 3D design image of the used as process for finite element analysis.



Fig.12.4 Shows the machining of form die. It is generated by Delcam software, imported to CNC machine for manual machining.



Fig.12.5 Shows the physical preparation of form die from an So that the material which we are going to being the die is removed by using plastic materials. In good work of the die-casting.

2 MATERIAL USED- D6

(D6) (Average Iron and Steel Industry) D6 can used in a high velocity, high clearance and sand alloyed with tungsten that is characterized by high compressive strength, high wear resistance, high surface hardness and good hardening ability.

2.1 Thermal Properties of the material used

The thermal properties of D6 steel used are given in the following table.

Property	Value
Thermal expansion coefficient (20-1000°C) (10 ⁻⁶ /°C)	13.4 µm/m°C
Thermal conductivity (300°C) (W/mK)	48.1 W/mK

2.2 Boundary condition

Fixed support at the flange and base of the die as the cavity. The various other flash formation as represented

$$u_{xx} = 2\alpha_1 \sigma_{xx} (\epsilon - 1) + \sigma_{yy}$$

$$\sigma_{xx} = (K_1 / K_2) (1 + \epsilon^2) (K_1 + K_2) \epsilon + \sigma_{yy}$$

$$\sigma_{yy} = (2.2 \sigma_{xx} \epsilon^2) / (\epsilon + \sigma_{xx})$$

Flange

$$K_1 = -1.2 \sigma_{xx} \epsilon$$

$$K_2 = -\sigma_{xx} K_1 + 2.2 \sigma_{xx} \epsilon^2 + 1.6 \epsilon^2 / \epsilon$$

$$K_3 = \epsilon - 1.2 K_1$$

$$\sigma_{xx} \epsilon = \epsilon - (\sigma_{xx} \epsilon^2) - (\sigma_{xx} \epsilon^2) / 2.2 \sigma_{xx} \epsilon^2 - (\sigma_{xx} \epsilon^2)$$

2.3 Thermal Analysis of Boundary condition

Input parameters used in the simulation of analysis.

Initial temperature: 100°C, Temperature on the die cavity: 50°C, Convection coefficient: 100 W/m²°C and boundary condition of an-D6 steel.



Fig.12.6 Shows the boundary condition used for thermal analysis, convection and temperature was given as thermal.



3. RESULTS & DISCUSSION



Fig 3.1 Shows Oil Barrel with practical approach of 100%



Fig 3.1.1 Shows the barrel test for impact temperature



Fig 3.1.2 Shows the Ball counter weight



Fig 3.1.3 Shows the output comparison of present project

3.1 Analysis in Area 15.1 for present project

General Information	
Project No.	000000
Project Name	Oil Barrel
Project Location	Oil Barrel
Project Start Date	01/01/2020
Project End Date	31/12/2020
Project Status	Completed
Project Manager	Dr. J. K. S. S. S.
Project Sponsor	Dr. J. K. S. S. S.
Project Stakeholders	Dr. J. K. S. S. S.
Project Risks	Dr. J. K. S. S. S.
Project Issues	Dr. J. K. S. S. S.
Project Change Log	Dr. J. K. S. S. S.
Project Deliverables	Dr. J. K. S. S. S.
Project Budget	Dr. J. K. S. S. S.
Project Resources	Dr. J. K. S. S. S.
Project Performance	Dr. J. K. S. S. S.
Project Quality	Dr. J. K. S. S. S.
Project Security	Dr. J. K. S. S. S.
Project Compliance	Dr. J. K. S. S. S.
Project Sustainability	Dr. J. K. S. S. S.
Project Innovation	Dr. J. K. S. S. S.
Project Impact	Dr. J. K. S. S. S.
Project Legacy	Dr. J. K. S. S. S.

3.1.1 STATIC STRUCTURAL ANALYSIS OF THE OIL

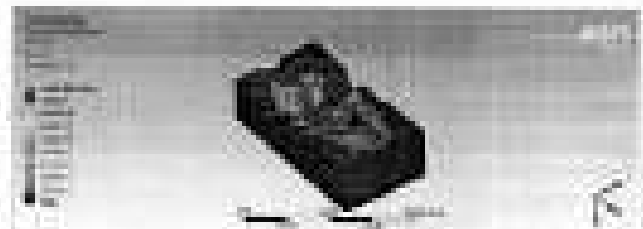


Fig 3.1.1 Shows the static deformation of the bottom due to the load applied. The maximum deformation was found to be 0.0013mm at the bottom of the oil

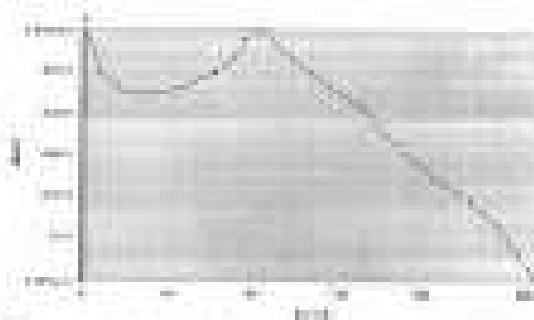


Fig 3. Total deformation in mm along y-axis and length of the barrel in mm along x-axis



Fig.3 Shows the equivalent stress variation across the circular neck of 160 MPa von-Mises

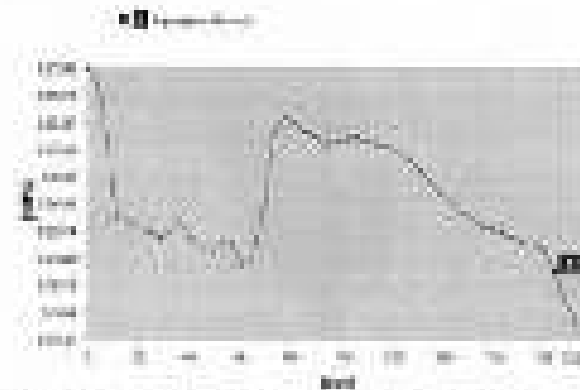


Fig.4 Stress variation at the various cross-sectional points A and change of the stress points.

5.0 Hot forging temperatures of different metals and alloys

Metal or alloy	Approximate range of forging temperature, °C (°F)
Aluminum alloys (cast & wrought)	300-350 (570-660)
Magnesium alloys	250-300 (480-540)
Copper alloys	500-800 (910-1470)
Carbon and low alloy steels	850-1150 (1550-2100)
Nonferrous alloy-free steels	1100-1200 (2000-2170)
Maraging steels	1100-1200 (2000-2170)
Aluminum stainless steels	1100-1250 (2000-2280)
Tool steels	1100-1250 (2000-2280)
Low temperature TiH titanium alloys	1100-1250 (2000-2280)
Crucible alloys	700-900 (1290-1640)
Low alloy superalloys	1050-1300 (1910-2360)
Cast base superalloys	1050-1200 (1910-2180)
Nickel alloys	900-1100 (1650-2000)
Titanium alloys	1000-1200 (1830-2160)

Manufacturing alloys: 1100-1200 (2000-2160)
 Nickel-base superalloys: 1050-1300 (1910-2360)
 Turbine alloy (cast & wrought): 1000-1300 (1830-2360)

Process dimensions for carbon and low-alloy steels

Dimension of the Bulk Region	Process Dimension
Plane	Slab/plate
Elementary width (B_0)	$B_0 \leq B_1 \leq C \leq B_2$ $B_0 \leq C \leq B_1 \leq B_2$
Final width (B_f)	$B_f \leq 1.2 B_0 \leq 1.14 \text{ mm}$ (0.045 in.)
Elementary length (L_0)	When B_0 is less than 10 mm (0.39 in.), then $L \leq 2$ mm (0.078 in.)
	When B_0 is between 10 mm and 25 mm (0.39 in. to 0.98 in.), then $L \leq 3$ mm (0.118 in.)
	When B_0 is between 25 mm and 50 mm (0.98 in. to 1.97 in.), then $L \leq 4$ mm (0.157 in.)
	When B_0 is greater than 50 mm (1.97 in.), then $L \leq 7$ mm (0.276 in.)

B_0 : Width of side of primary; B_1 : The width of primary; B_2 : width of secondary; C : thickness of the slab; L : length of slab

6.THERMAL ANALYSIS



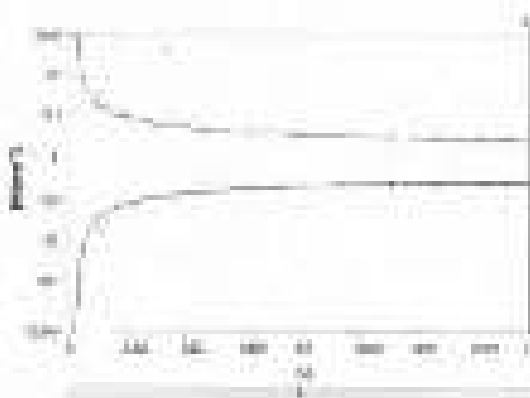
Fig.1 Shows the temperature distribution on the die, maximum at 300 °C was found. It shows that the temperature variation



Fig.2 It shows the normal view of the die, maximum at 100 °C was observed and the change of temperature gradient



Fig 3 Shows the Total Heat Flux variation across the direction X-axis in all XY-Zal sections.



> Directional Heat Flux

Fig 4 Increase and Decrease of directional heat flux along the direction

CONCLUSION

- 1) Finite element method based simulation is a powerful tool that can provide valuable data about the variables that can affect temperature distribution. The stress and temperature curves for particular points of a forged part.
- 2) The knowledge of these values is the key to optimizing the existing processes and developing new processes and materials. This is, however, impossible without method models required for a reliable analysis of the process.
- 3) The present work starts with the application of a FEM simulation on the casting and forging manufacturing process in order to do forging.
- 4) The first application of the numerical simulation method and constructing a new model is needed. It described the heat transfer change in detail in the forging during the heating to the manufacturing temperature. Indeed, the present study has

limited the heating and cooling rates of the forging in the furnace can be determined, the optimum aspect of the forging in the furnace can be found in all various types of problems solved.

Scope of the Future work

- It will focus on the thermo-mechanical analysis and predict precisely thermal properties of a work piece in conventional forging.
- However, such results should be interpreted with caution and this study should be supported by a large body of experimental data.
- In future works, the FEM simulation of forging processes will be refined, e.g., the plastic behavior of a forged part instead of the stress plots employed in this.
- In a future work, different die and workpiece material can be analyzed, perfect geometry and forging of the part at lower temperatures can be analyzed.
- Finally, the results of this study, from they can be used, good numerical analysis may be performed by a better Finite Element programs.

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DESIGN AND THERMAL ANALYSIS OF WARM (COLD) FORGING PROCESS AND DIES

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Abstract— Forging temperature is one of the basic considerations in forging process. In warm forging, the metals are forged at temperatures about the recrystallization temperature and below the traditional hot forging temperature. Warm forging has many advantages when compared when compared to cold forging. In this study, forging process of a part which is currently produced at the hot forging. Temperature range and which needs some improvements in accuracy, material usage and energy concepts, is analyzed. The proposed process is simulated using Finite Element Method and Ansys. Experimental study is also carried out in SRM Center Forging Research and Application Laboratory and it has been observed that experimental results are in good agreement.

Waste is reduced by proposing saving of energy using and better accuracy in dimension is achieved by reducing the forging temperature from the hot forging to the warm forging temperature range.

Keywords : Warm Forging, Finite Element Analysis, Die stress Analysis, Metal Forging, Cold die Forging.

INTRODUCTION

The term "forging" is applied to processes in which a piece of metal is worked in a machine to the desired shape by plastic deformation of the working metal. The energy that produces deformation is applied by a hammer, press, or other working rolls, other dies or its combination. A forging is

produced in three distinct phases: stock preparation in the form of blooms, billets, bar or ingots, plastic deformation of the metal component to rough, close tolerance or net shape in one of the forging processes and appropriate secondary operations.

In this study, forging process of a part which is currently produced at the hot forging temperature range and which needs some improvements in accuracy, material usage and energy concepts, is analyzed. The forging process required design with a new process design for the particular part is proposed in warm forging temperature range and the proposed process is simulated using Finite Element Method. In the simulation, coupled thermal mechanical analysis are performed and the dies are modeled as deformable bodies to consider die stress analysis.

OBJECTIVES OF PROJECT

1. A forging die designed and manufactured to get the correct of hammer and the design.
2. Process will be identified.
3. Thermal properties of die material as well as steel materials are observed.
4. The forged part of steel materials in the furnace and temperature will be studied by FEM.
5. How feasible while hammering the component from component to the material will be optimized with the aim of profit.
6. The work feasibility and cost saving has thermo-mechanical analysis is carried out for the maximum production of the component of steel rod.



- The material formation will be observed physically. (A better conclusion compare to theoretical analysis).

LITERATURE SURVEY

Chen et al., hot forging process was simulated coupled with thermal analysis, in order to determine effects of the process on the hammer die for the taper perforate stage in upset forging process. For different cases, the stress distribution in the hammer die had been presented. The obtained stress distributions were also used for the fatigue analysis of the die.

Kim, Yagi, Yamazaki et al., a history of practical use of FE simulation in forging area was briefly reviewed. Then, practical use and benefits of FE simulation in forging area were discussed with examples. Finally, key points for successful and effective use of FE simulation were explained followed by current trends for better use of the FE simulation as a main tool in the forging industry.

DIE HAMMERS

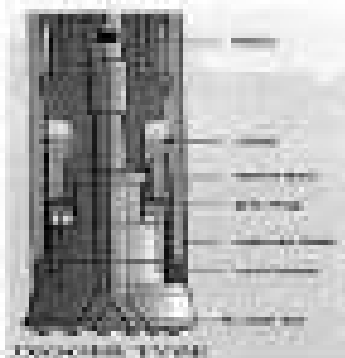
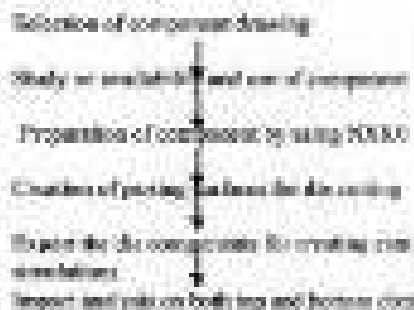


Fig1 The legs are used in R.R ring and sets as rollers

I. METHODS FOR DESIGNING AND ANALYSIS TO DEVELOP THE WORK

Design and Experimental procedure:



Simulation on temperature variation and thermal war distortions.

1.1 DESIGNING PROCEDURE



Fig 1.1 Shows Fig 1.1

Toolbars and tools

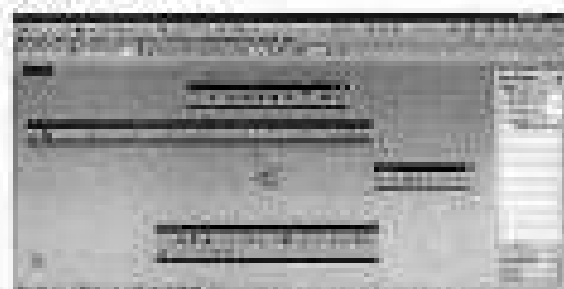


Fig 1.2 Shows the software tool bar codes for design

1.2 Computer 3D drawing with NX/UG design

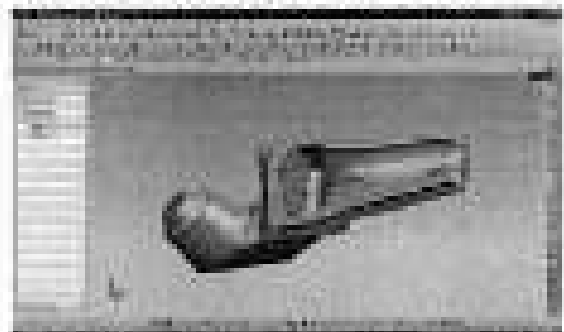


Fig. 1.2.1 Shows the component before to start the construction of existing part for die design.



Fig 1.2.2 Shows the hammer die casting which is used at present for this experimental analysis. Material can be added in the present for use die to facilitate manufacturing process.

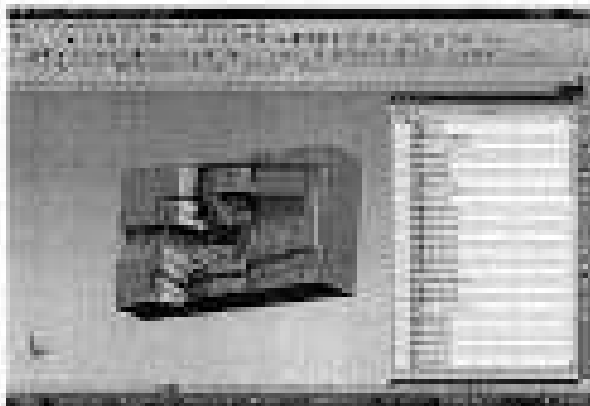


Fig 1.2.3 Shows the design of sectioned between die work metal die punch for manufacturing with model app. It is the 2D design image of Die used as present for this experimental analysis.



Fig.1.2.4 Shows the machining of section die. By generating the programming by Die and the work placed to CNC machine for practical machining.



Fig 1.2.5 Shows the paper preparation of section die because before the material which we are going to forging the die is machine by using plastic materials for good result of the component.

2. MATERIAL USED- D6

AISI (American Iron and Steel Institute) D6 tool steel is a high-carbon, high-chromium tool steel alloyed with niobium that is characterized by high compressive strength, high wear resistance, high surface hardness and good hardening ability.

2.1 Thermal Properties of the material used

The thermal properties of D6 tool steel are given in the following table:

Property	Value
Thermal expansion coefficient (20°C - 400°C/400°C)	11.5 ppm/°C
Thermal conductivity (20°C/400°C)	55.2 W/mK

2.2 Boundary conditions:

Fixed support at the 4 faces and back of 1000MM on the cavity. The stresses after each formation is represented

$$\sigma_{xx} = 2\mu(\sigma_y(\psi/\bar{r}) + \sigma_z)$$

$$\sigma_{yy} = (E_1/E_2)(\mu\bar{r}^2/E_1 + E_2\psi/\bar{r}) + \sigma_{zz}$$

$$\sigma_{zz} = (E_1/E_2)(\mu\bar{r}^2/\bar{r}) + \sigma_{zz}$$

Where,

$$E_1 = -\tan^2 \beta$$

$$E_2 = -\sigma_{zz}E_1 + 2\mu\sigma_{zz}E_1 + \tan^2 \beta$$

$$E_3 = \bar{r} - r_1E_2$$

$$\tan^2 \beta = [1 - 2(\bar{r}/r_1) - 1 + 2(\bar{r}/r_1)(\mu\bar{r}/r_1)]/4\mu^2$$

2.3. Thermal Analysis of Boundary conditions

Input parameters used in the numerical simulation.

Initial temperature 100°C, Temperature on the die cavity 400°C, Convection on the surface with convective coefficient of 1000W/m²



Fig2.1.1 Shows the boundary conditions used for thermal analysis, convection and temperature was given at all over.



3. RESULTS & DISCUSSION



Fig 3.1.1 Shows the furnace with practical approach of 100%



Fig 3.1.2 Shows the furnace with pre heating properties



Fig 3.1.3 Shows die with both heating and cooling



Fig 3.1.4 Shows die cut pre component of process project

4.0 Analysis in Ansys 15.5 for present project

ANSYS Workbench	
Job Information	
Job Name	Job_1
Job Path	C:\Program Files\ANSYS Inc\Workbench\
Job Date	10/09/2023 10:00:00 AM
Job Time	10:00:00 AM
Job Status	
Job Status	Completed
Job Progress	100%
Job Error	None
Job Warning	None
Job Message	None
Job Details	
Job ID	Job_1
Job Name	Job_1
Job Path	C:\Program Files\ANSYS Inc\Workbench\
Job Date	10/09/2023 10:00:00 AM
Job Time	10:00:00 AM
Job Status	Completed
Job Progress	100%
Job Error	None
Job Warning	None
Job Message	None

4.1 STATIC STRUCTURAL ANALYSIS OF THE DIE



Fig 4.1.1 Shows the total deformation of the bottom die for load applied, the maximum deformation was found to be 0.0015 mm at the corner of the die.

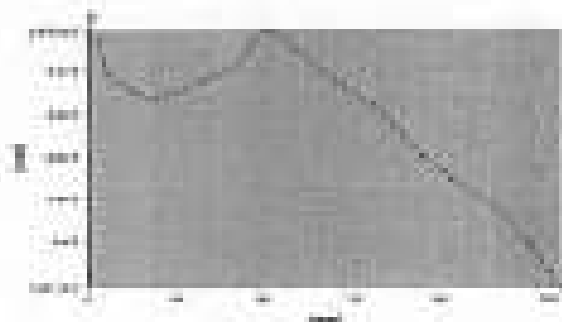


Fig 4.1.2 Total deformation in mm along y-axis and Length of the cavity in mm along X-axis

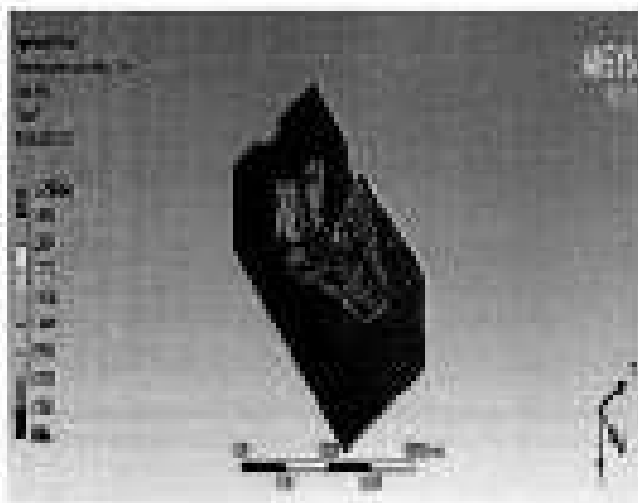


Fig.3 Shows the equivalent stress varied across the die and maximum of 1400MPa was found.

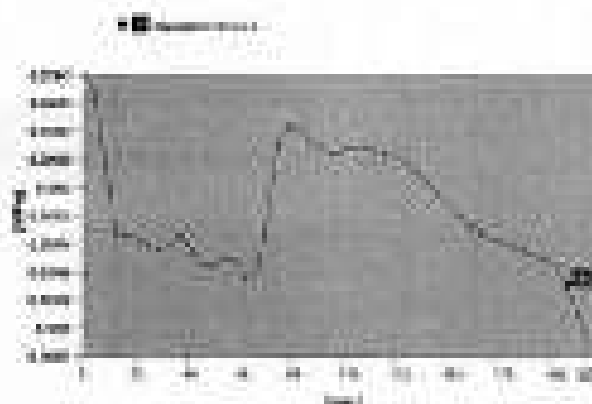


Fig.4 Shows equivalent of the stress across the die and point of stress and change of the stress point.

5.8 Hot forging temperatures of different metals and alloys

Metal or alloy	Approximate range of forging temperature, °C (°F)
Alumina alloys (most difficult)	400-500 (750-930)
Magnesium alloys	250-450 (480-840)
Copper alloys	600-900 (1110-1650)
Carbon and low-alloy steels	950-1200 (2000-2160)
High-alloy steels	1100-1250 (2000-2280)
Maraging steels	1100-1250 (2010-2280)
Austenitic stainless steels	1100-1250 (2010-2280)
Nickel alloys	1000-1150 (1830-2100)
Superalloys: Fe-Ni alloys	1100-1200 (2010-2160)
Titanium alloys	700-850 (1290-1540)
Inconel superalloys	1050-1150 (1910-2100)
Cold-chamber superalloys	1100-1200 (2010-2160)
Nickel alloys	900-1150 (1650-2100)
Titanium alloys	1100-1150 (2000-2100)

Nickel-based alloys	1100-1250 (2010-2280)
Nickel-base superalloys	1050-1200 (1910-2160)
Titanium alloys (most difficult)	1100-1150 (2010-2100)

6. Prefore dimensions for carbon or low-alloy steels

Dimension of the Die or Cavity	Prefore Dimension
Blank	No Blank
Blank or with (M_1)	$R_p = R_d + C$ $M_1 < R_p + M_2$
With with (M_2)	$R_p = 1.2 R_d + 1.15$ mm (0.025 in.)
Depth of cavity (M_3) , mm (in.) and depth of adjacent cavity (M_4) , mm (in.)	When M_3 is less than 10 mm (0.4 in.), then $C = 2$ mm (0.08 in.) When M_3 is between 10 mm and 25 mm (0.4 in. to 1 in.), then $C = 3$ mm (0.12 in.) When M_3 is between 25 mm and 50 mm (1 in. to 2 in.), then $C = 4$ mm (0.16 in.) When M_3 is greater than 50 mm, then $C = 5$ mm (0.2 in.)

R_p , finished radii of prefore; R_d , die radii of prefore; M_1 , cavity depth; C , adjacent cavity depth. Source: Ref 7

6.THERMAL ANALYSIS



Fig.5 Shows the temperature distribution over the die, maximum of 1070°C was found. It also shows the temperature variations.

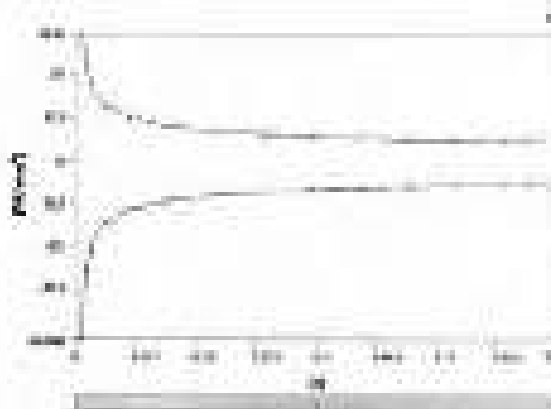


Fig.6 Shows the thermal error of the die, maximum of 130 was observed and the change of temperature gradient.



Heat Flux Plot

Fig.63 Shows the Heat flux variation across the die in the X, Y, Z directions.



Directional Heat Flux

Fig.64 is showing the Directional variation with heat flux along the Direction

CONCLUSION

1. Finite element method based simulation is a powerful tool that can provide information about the variables that are difficult to measure otherwise like the stress and temperature contours for particular points of a forged part.
2. The basic design stress values is the key to optimizing the forging process and developing new parameters and materials. This is, however, responsible without verified models required for accurate analysis of the process.
3. The present work deals with the application of a FEM simulation to optimizing the manufacturing process of die-cast forging.
4. The first application of the numerical simulation method simulating a compression model. It described the temperature changes in die-cast forgings during the forging to find out an ideal temperature before the quenching. Using this

model, the forging and working stress of the forging in hot furnace can be determined. An optimum layout of the forgings in the furnace can be found and various types of products tested.

Scope of the Future work

- It was found that thermo-mechanical treatment can produce practically identical properties of a work piece as conventional heat-treating.
- However, such results should be interpreted with caution and this finding should be supported by a larger body of verified data.
- In future efforts, the FEM simulation of forging processes will be refined, e.g., the plastic behavior of a forged part instead of the curve plots employed so far.
- As a future work, different die and workpiece materials can be obtained perhaps geometries and forging of the part at lower temperatures can be analyzed.
- By using the results of this study, finite element analysis can be redesigned. Statistical analysis may be performed by different finite element programs.

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